Boundary Zonal Flow in Rotating Turbulent Rayleigh-Bénard Convection

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For rapidly rotating turbulent Rayleigh–Bénard convection in a slender cylindrical cell, experiments and direct numerical simulations reveal a boundary zonal flow (BZF) that replaces the classical large-scale circulation. The BZF is located near the vertical side wall and enables enhanced heat transport there. Although the azimuthal velocity of the BZF is cyclonic (in the rotating frame), the temperature is an anticyclonic traveling wave of mode one, whose signature is a bimodal temperature distribution near the radial boundary. The BZF width is found to scale like $Ra^{1/4}Ek^{2/3}$ where the Ekman number *Ek* decreases with increasing rotation rate.

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Turbulent fluid motion driven by buoyancy and influenced by rotation is a common phenomenon in nature and is important in many industrial applications. In the widely studied laboratory realization of turbulent convection, Rayleigh-Bénard convection (RBC) [1,2], a fluid is confined in a convection cell with a heated bottom, cooled top, and adiabatic vertical walls. For these conditions, a large scale circulation (LSC) arises from cooperative plume motion and is an important feature of turbulent RBC [1]. The addition of rotation about a vertical axis produces a different type of convection as thermal plumes are transformed into thermal vortices, over some regions of parameter space, heat transport is enhanced by Ekman pumping [3–10], and statistical measures of vorticity and temperature fluctuations in the bulk are strongly influenced [11–17]. A crucial aspect of rotation is to suppress, for sufficiently rapid rotation rates, the LSC of nonrotating convection [12,13,18,19], although the diameter-to-height aspect ratio $\Gamma = D/H$ appears to play some role in the nature of the suppression [20].

In RBC geometries with $1/2 \le \Gamma \le 2$, the LSC usually spans the cell in a roll-like circulation of size H. For rotating convection, the intrinsic linear scale of separation of vortices is reduced with increasing rotation rate [21,22], suggesting that one might reduce the geometric aspect ratio, i.e., $\Gamma < 1$ while maintaining a large ratio of lateral cell size to linear scale [5]; such convection cells are being implemented in numerous new experiments [23]. Thus, an important question about rotating convection in slender cylindrical cells is whether there is a global circulation that substantially influences the internal state of the system and carries appreciable global heat transport. Direct numerical simulations (DNS) of rotating convection [24] in cylindrical geometry with $\Gamma = 1$, inverse Rossby number 1/Ro = 2.78, Rayleigh number $Ra = 10^9$, and Prandtl number Pr = 6.4(Ro, Ra, and Pr defined below) revealed a cyclonic azimuthal velocity boundary-layer flow surrounding a core region of anticyclonic circulation and localized near the cylinder sidewall. The results were interpreted in the context of sidewall Stewartson layers driven by active Ekman layers at the top and bottom of the cell [25,26].

Here, we show, through DNS and experimental measurements for a cylindrical convection cell with $\Gamma = 1/2$ at large *Ra* and for a range of rotation rates from slow to rapid,

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FIG. 1. Sidewall temperature PDFs, r/R = 1, for z/H = 1/4 (diamonds), z/H = 1/2 (circles), and z/H = 3/4 (squares), with 1/Ro = 0 (a), (c) and 10 (b), (d). Experimental measurements with $Ra = 8 \times 10^{12}$ (a), (b) and DNS with $Ra = 10^9$ (c), (d), both with Pr = 0.8. Bimodal Gaussian distributions (solid lines), the sum of two normal distributions (dashed lines), are observed for rapid rotation (b), (d). $\langle \cdot \rangle_{z_e}$ denotes average in time and over all sensor positions at distance z from the hot plate.

that a wider (several times the Stewartson layer width) annular flow, denoted as boundary zonal flow (BZF), has profound effects on the overall flow structure and on the spatial distribution of heat flux. In particular, this cyclonic zonal flow surrounds an anticyclonic core. The BZF has alternating temperature sheets that produce bimodal temperature distributions for radial positions r/R > 0.7 and that contribute greatly to the overall heat transport; 60% of heat transport are carried in the BZF. Although the location of the azimuthally averaged maximum cyclonic azimuthal velocity, the root-mean-square (rms) vertical velocity fluctuations, and the normalized vertical heat transport at the midplane are consistent with a linear description of a Stewartson-layer scaling [24], the dynamics of temperature, vertical velocity, and heat transport in the BZF are more complex and interesting. The robustness of the BZF state as evidenced by its existence over 7 orders of magnitude in Ra in DNS and experiment and over a range $1/2 \le \Gamma \le 2$ and $0.1 \le Pr \le 4.4$ (results to be presented elsewhere) suggests that this is a universal state of rotating convection that needs a physical understanding.

The dimensionless control parameters in rotating RBC are the Rayleigh number $Ra = \alpha g \Delta H^3 / (\kappa \nu)$, Prandtl number $Pr = \nu/\kappa$, cell aspect ratio Γ , and Rossby number $Ro = \sqrt{\alpha g \Delta H} / (2\Omega H)$ or, alternatively, Ekman number $Ek = \nu/(2\Omega H^2)$. Here, α is isobaric thermal expansion coefficient, ν kinematic viscosity, κ fluid thermal diffusivity, g acceleration of gravity, Ω angular rotation rate, and Δ temperature difference between the hotter bottom and colder top plates. The main integral response parameter we consider is the Nusselt number $Nu \equiv \langle \mathcal{F}_z \rangle_{t,V}$, where $\langle \cdot \rangle_{t,V}$ denotes the time and volume averaging and $\mathcal{F}_z \equiv [u_z(T - T_0) - \kappa \partial_z T] / (\kappa \Delta/H)$ is the normalized vertical heat flux with u_z being the vertical component of the velocity and T_0 the average of the top and bottom temperatures.

We present numerical and experimental results [27] for rotating RBC in a $\Gamma = 1/2$ cylindrical cell and 1/Ro = 0, 0.5, and 10. The DNS used the GOLDFISH code [28,29] with Pr = 0.8 and $Ra = 10^9$. The experiments used pressurized

sulfur hexafluoride (SF₆) and were performed over a large parameter space in the High Pressure Convection Facility (HPCF, 2.24 m high) at the Max Planck Institute for Dynamics and Self-Organization in Göttingen [30]. In the studied parameter range, the Oberbeck–Boussinesq approximation is valid [31–33], and the centrifugal force is negligible [8,34,35].

First, we consider the azimuthal variation of the temperature measured by thermal probes at or near the sidewall, a commonly used technique for parametrizing the LSC in RBC [18,20,36-38]. We measured, experimentally and in corresponding DNS, the temperature at eight equidistantly spaced azimuthal locations of the sensors for each of three distances from the bottom plate: z/H = 1/4, 1/2, and 3/4. The probability density functions (PDFs) of the experimental data without rotation $(1/Ro = 0, Ra = 8 \times 10^{12})$ in Fig. 1(a) show a distribution with a single peak and slight asymmetry to hotter (colder) fluctuations for heights smaller (larger) than z/H = 1/2, whereas the PDFs for rapid rotation [1/Ro = 10, Fig. 1(b)], show a bimodal distribution that is well fit by the sum of two Gaussian distributions. The corresponding PDFs of the DNS data (at $Ra = 10^9$) show the same qualitative transition from a single peak without rotation to a bimodal distribution in the rapidly rotating case with similar hot-cold asymmetry for different z [Figs. 1(c) and 1(d)]. To understand the nature of the emergence of a bimodal distribution near the radial boundary, we consider the DNS data in detail.

The LSC for nonrotating convection in cells with $1/2 \le \Gamma \le 2$ and at large Ra extends throughout the entire cell with a large roll-like circulation [39]. With slow rotation, Coriolis forces induce anticyclonic motion close to the plates owing to the diverging flow between the LSC and the corner rolls. At the midplane, the LSC is tilted with a small inward radial velocity component that rotation spins up into cyclonic motion. These tendencies are illustrated for 1/Ro = 0.5 in Figs. 2(a) and 2(c), respectively, where streamlines of time-averaged velocity are overlaid on the field of azimuthal velocity. Figure 2(a) shows fields evaluated at the thermal boundary layer (BL) height $z = \delta_{\theta} \equiv H/(2Nu)$,



FIG. 2. Horizontal cross sections of time-averaged flow fields (DNS), visualized with streamlines (arrows) and azimuthal velocity $\langle u_{\phi} \rangle_{t}$ (colors) (a), (b) at height of thermal BL, $z = \delta_{\theta} \equiv H/(2Nu)$ and (c), (d) at the midplane, z = H/2, with $Ra = 10^{9}$ and 1/Ro = 0.5 (a), (c) and 10 (b), (d). Blue (pink) color indicates anticyclonic (cyclonic) motion. In (d), locations $r = r_{0}$ of $\langle u_{\phi} \rangle_{t} = 0$ (solid line) and $r = r_{u_{\phi}^{max}}$ of the maximum of $\langle u_{\phi} \rangle_{t}$ (dashed line) are shown.

demonstrating the dominant anticyclonic flow near the boundary. The situation is reversed at the midplane [Fig. 2(c)] where cyclonic motion extends over almost the entire cross sectional area.

For rapid rotation, viscous Ekman BLs near the plates induce anticyclonic circulation with radial outflow in horizontal planes as in Fig. 2(b). The outflow is balanced by the vertical velocity in an increasingly thin (with increasing 1/Ro) annular region near the sidewall where cyclonic vorticity is concentrated at the midplane, see Fig. 2(d). The core region, on the other hand, is strongly anticyclonic owing to the Taylor-Proudman effect [40,41] that tends to homogenize vertical motion. The circulation for a rotating flow in a finite cylindrical container consists of thin anticyclonic Ekman layers on top and bottom plates and compensating Stewartson layers along the sidewalls with upflow from the bottom and downflow from the top [24,42]. This classical BL analysis was successfully applied to rotating convection [24] for a $\Gamma = 1$ cylindrical cell with Pr = 6.4 and $10^8 \le Ra \le 10^9$ in both experiment and DNS. No evidence for a coherent large-scale circulation for rapid rotation was found in those studies.

For our conditions, Pr = 0.8, $Ra = 10^9$, and 1/Ro = 10, we compute the time- and azimuthal-average azimuthal velocity $\langle u_{\phi} \rangle_{t,\phi}$ (normalized by the free-fall (ff) velocity $u_{\rm ff} = \sqrt{\alpha g R \Delta}$) as a function of height z for fixed r = 0.95R and of radius r at fixed z = H/2. The height dependence of $\langle u_{\phi} \rangle_{t,\phi}$, Fig. 3(a), shows an anticyclonic (negative) circulation close to the top and bottom plates and an increasingly cyclonic (positive) circulation with increasing (decreasing) z from the bottom (top) plate. The radial dependence, Fig. 3(b), demonstrates the sharp localization of cyclonic motion near the sidewall as parametrized by the zero-crossing r_0 (solid line) and the maximum $r_{u_{\phi}^{\text{max}}}$ (dashed line). Corresponding distances from the sidewall are $\delta_0 = R - r_0$ and $\delta_{u_{\phi}^{\max}} = R - r_{u_{\phi}^{\max}}$ where $\delta_{u_{d}^{\text{max}}} \approx \delta_{u_{z}^{\text{ms}}}$ (based on maximum of rms of u_{z}). $\delta_{u_{z}^{\text{ms}}}$ was used to define the sidewall Stewartson layer thickness in rotating convection [24], and our results for $\langle u_{\phi} \rangle_t$ are consistent with that description. What was absolutely not expected is the strong azimuthal variation of the instantaneous temperature T shown in Fig. 3(c), a feature that defines the global flow circulation, namely, the spatial distribution of the heat transport which is the origin of the bimodal temperature distributions seen in the experiments and DNS.

The strong variations in instantaneous temperature shown in Fig. 3(c) organize into anticyclonic traveling waves illustrated in the angle-time plot of *T*, Fig. 4(a). The BZF height is order *H*, Fig. 3(c), but is increasingly localized in the radial direction as the rotation rate increases (*Ro* and *Ek* decrease) so that $\delta_0/R \ll 1$. The azimuthal mode of *T* is highly correlated with a corresponding mode of the vertical velocity, Fig. 4(b), with a resulting coherent



FIG. 3. For $Ra = 10^9$, 1/Ro = 10: (a) $\langle u_{\phi} \rangle_{t,\phi}$ vs z at r = 0.95R. The inset shows the same data for $0 \le z \le H/2$ in a log plot. (b) $\langle u_{\phi} \rangle_{t,\phi}$ vs r at z = H/2; radial zero crossing $r = r_0$ (solid line) and radial maximum $r = r_{u_{\phi}^{\text{max}}}$ (dashed line). (c) Instantaneous thermal field at $r = r_{u_{\phi}^{\text{max}}}$ vs z and ϕ .



FIG. 4. For $Ra = 10^9$ and 1/Ro = 10: (a), (b) angle-time plots at $r = r_{u_{\phi}^{\text{max}}}$, z = H/2 of (a) T and (b) u_z ; (c) normalized timeaveraged vertical heat flux $\langle \mathcal{F}_z \rangle_t$ at z = H/2. In (c), location of rwhere $\langle \mathcal{F}_z |_{z=H/2} \rangle_t = Nu$ (dashed-dotted line) and locations r = r_0 of $\langle u_{\phi} \rangle_t = 0$ (solid line) and $r = r_{u_{\phi}^{\text{max}}}$ of the maximum of $\langle u_{\phi} \rangle_t$ (dashed line) are shown. Color scale from blue (min values) to pink (max values) ranges (a) between the top and bottom temperatures, (b) in $[-u_{\text{ff}}/2, u_{\text{ff}}/2]$, (c) from 0 to midplane magnitude of $\langle \mathcal{F}_z \rangle_t$, which is $\approx 3.4Nu$.

mode-1 (m = 1) anticyclonic circulation in ϕ with a warm upflow on one side of the cell balanced by a cool downflow on the other side of the cell (for $\Gamma = 1, 2$, the dimensionless wave number $m/\Gamma = 2$ is independent of Γ , to be presented elsewhere). The anticyclonic circulation is the speed of the anticyclonic horizontal BL, suggesting that the thermal wave is anchored at the horizontal BLs so that it travels against the cyclonic circulation near the sidewall. The coherence between T and u_z leads to localization of vertical heat flux near the sidewall shown in Fig. 4(c) where the heat flux within the annular area defined by δ_0 is $\approx 60\%$ of the total heat flux.

We arrive at a compact description of the BZF. The radial distances from the sidewall $\delta_{u_z^{\text{rms}}}$, $\delta_{\mathcal{F}_z^{\text{max}}}$, and $\delta_{u_z^{\text{max}}}$ of maxima of u_z -rms, heat flux \mathcal{F}_z , and u_ϕ , respectively, scale as $Ek^{1/3}$, Fig. 5(a), consistent with the expectations of Ekman-Stewartson BL theory [24,42]. On the other hand, the cyclonic zone width δ_0 decreases more rapidly with *Ek*, i.e., as $Ek^{2/3}$ with a $Ra^{1/4}$ dependence (presented elsewhere). Thus, the inner layer is consistent with Stewartson theory whereas the outer structure reflects the more complex character of interacting thermal and velocity fields. The bimodal temperature distribution is now explained by the alternating thermal field. We plot the radial dependence of the mean values of the bimodal distributions (the bimodal PDFs are well fit by the sum of two Gaussians) from the DNS for $Ra = 10^9$, 1/Ro = 10 in Fig. 5(b). The unimodal distribution for small r/R bifurcates sharply to a bimodal distribution for $r/R \approx 0.72$. The corresponding experimental measurements do not provide data at intermediate r/R, but are consistent (dashed curve) with a scaled BZF width based on the scaling $Ra^{1/4}Ek^{2/3}$. Finally, the transition value of $1/Ro \approx 2$ from unimodal to bimodal distributions is roughly independent of Ra as indicated in Fig. 5(c).

Our observations provide insight into experimental results for $\Gamma = 1/2$ in water with Pr = 4.38 [20], where the mode-1 LSC for nonrotating convection was reported to



FIG. 5. (a) Scaling of BZF widths δ_0 , $\delta_{u_{\phi}^{\text{max}}}$, $\delta_{u_z^{\text{max}}}$, $\delta_{u_z^{\text{max}}}$ with Ek (DNS for $Ra = 10^9$); (b) Fitted peak values of bimodal PDF distributions (normalized by σ , standard deviation of T) at z/H = 1/2 vs r/R: DNS ($Ra = 10^9$) and measurements ($Ra = 8 \times 10^{12}$), both for 1/Ro = 10; (c) diagram of the bimodal and unimodal temperature distributions at r = R, according to our DNS ($Ra = 10^9$) and experiments (larger Ra) of rotating RBC for $Pr \approx 0.8$ and $\Gamma = 1/2$. Critical inverse Rossby number equals $1/Ro_c = 2 \pm 1$ (shown with a dashed line).

transition into a then unknown state. Our BZF is that unknown global mode. We conclude that the BZF exists over a broad range of parameters $1/2 \le \Gamma \le 2$, $0.1 \le Pr \le 4.4$, and $10^8 \le Ra < 10^{15}$ (details to be published elsewhere). Here, we presented details for Pr = 0.8 and $\Gamma = 1/2$ and for Ra spanning 7 orders of magnitude [27]. A fully quantitative understanding remains a challenge for the future.

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- G. Ahlers, S. Grossmann, and D. Lohse, Heat transfer and large scale dynamics in turbulent Rayleigh-Bénard convection, Rev. Mod. Phys. 81, 503 (2009).
- [2] D. Lohse and K.-Q. Xia, Small-scale properties of turbulent Rayleigh-Bénard convection, Annu. Rev. Fluid Mech. 42, 335 (2010).
- [3] T. H. Rossby, A study of Bénard convection with and without rotation, J. Fluid Mech. 36, 309 (1969).
- [4] F. Zhong, R. Ecke, and V. Steinberg, Rotating Rayleigh-Bénard convection: Asymmetric modes and vortex states, J. Fluid Mech. 249, 135 (1993).
- [5] Y. Liu and R. E. Ecke, Transport Scaling in Turbulent Rayleigh–Bénard Convection: Effects of Rotation and Prandtl Number, Phys. Rev. Lett. 79, 2257 (1997).
- [6] R. P. J. Kunnen, H. J. H. Clercx, and B. J. Geurts, Heat flux intensification by vortical flow localization in rotating convection, Phys. Rev. E 74, 056306 (2006).
- [7] E. King, S. Stellmach, J. Noir, U. Hansen, and J. Aurnou, Boundary layer control of rotating convection systems, Nature (London) 457, 301 (2009).
- [8] J.-Q. Zhong, R. J. A. M. Stevens, H. J. H. Clercx, R. Verzicco, D. Lohse, and G. Ahlers, Prandtl-, Rayleigh-, and Rossby-Number Dependence of Heat Transport in Turbulent Rotating Rayleigh-Bénard Convection, Phys. Rev. Lett. **102**, 044502 (2009).
- [9] Y. Liu and R. E. Ecke, Heat transport measurements in turbulent Rayleigh-Bénard convection, Phys. Rev. E 80, 036314 (2009).
- [10] S. Stellmach, M. Lischper, K. Julien, G. Vasil, J. S. Cheng, A. Ribeiro, E. M. King, and J. M. Aurnou, Approaching the Asymptotic Regime of Rapidly Rotating Convection: Boundary Layers Versus Interior Dynamics, Phys. Rev. Lett. 113, 254501 (2014).
- [11] K. Julien, S. Legg, J. McWilliams, and J. Werne, Rapidly rotating turbulent Rayleigh-Bénard convection, J. Fluid Mech. 322, 243 (1996).

- [12] J.E. Hart, S. Kittelman, and D.R. Ohlsen, Mean flow precession and temperature probability density functions in turbulent rotating convection, Phys. Fluids 14, 955 (2002).
- [13] P. Vorobieff and R. E. Ecke, Turbulent rotating convection: An experimental study, J. Fluid Mech. 458, 191 (2002).
- [14] R. P. J. Kunnen, H. J. H. Clercx, B. J. Geurts, L. J. A. van Bokhoven, R. A. D. Akkermans, and R. Verzicco, Numerical and experimental investigation of structure-function scaling in turbulent Rayleigh-Bénard convection, Phys. Rev. E 77, 016302 (2008).
- [15] K. Julien, E. Knobloch, A. M. Rubio, and G. M. Vasil, Heat Transport in Low-Rossby-Number Rayleigh-Bénard Convection, Phys. Rev. Lett. **109**, 254503 (2012).
- [16] Y. Liu and R. E. Ecke, Local temperature measurements in turbulent Rayleigh-Bénard convection, Phys. Rev. E 84, 016311 (2011).
- [17] S.-S. Ding, H.-M. Li, W.-D. Yan, and J.-Q. Zhong, Temperature fluctuations relevant to thermal-plume dynamics in turbulent rotating Rayleigh-Bénard convection, Phys. Rev. Fluids 4, 023501 (2019).
- [18] R. P. J. Kunnen, H. J. H. Clercx, and B. J. Geurts, Breakdown of large-scale circulation in turbulent rotating convection, Europhys. Lett. 84, 24001 (2008).
- [19] J.-Q. Zhong and G. Ahlers, Heat transport and the largescale circulation in rotating turbulent Rayleigh-Bénard convection, J. Fluid Mech. 665, 300 (2010).
- [20] S. Weiss and G. Ahlers, The large-scale flow structure in turbulent rotating Rayleigh-Bénard convection, J. Fluid Mech. 688, 461 (2011).
- [21] S. Chandrasekhar, *Hydrodynamic and Hydromagnetic Stability* (Oxford University Press, Oxford, 1961).
- [22] K. Julien, A. Rubio, I. Grooms, and E. Knobloch, Statistical and physical balances in low Rossby number Rayleigh-Bénard convection, Geophys. Astrophys. Fluid Dyn. 106, 392 (2012).
- [23] J. S. Cheng, J. M. Aurnou, K. Julien, and R. P. J. Kunnen, A heuristic framework for next-generation models of geostrophic convective turbulence, Geophys. Astrophys. Fluid Dyn. 112, 277 (2018).
- [24] R. Kunnen, R. Stevens, J. Overkamp, C. Sun, G. van Heijst, and H. Clercx, The role of Stewartson and Ekman layers in turbulent rotating Rayleigh-Bénard convection, J. Fluid Mech. 688, 422 (2011).
- [25] K. Stewartson, On almost rigid rotation, J. Fluid Mech. 3, 17 (1957).
- [26] K. Stewartson, On almost rigid rotation. part 2, J. Fluid Mech. 26, 131 (1966).
- [27] The experimental measurements were performed for $0.804 \le Pr \le 0.941$, $8.4 \times 10^{12} \le Ra \le 8.0 \times 10^{14}$, and $Ek \ge 1.0 \times 10^{-8}$. The DNS were conducted for Pr = 0.8, $Ra = 10^9$, and $Ek \ge 2.8 \times 10^{-6}$, see Fig. 5(c).
- [28] G. L. Kooij, M. A. Botchev, E. M. Frederix, B. J. Geurts, S. Horn, D. Lohse, E. P. van der Poel, O. Shishkina, R. J. A. M. Stevens, and R. Verzicco, Comparison of computational codes for direct numerical simulations of turbulent Rayleigh–Bénard convection, Comput. Fluids 166, 1 (2018).
- [29] O. Shishkina, S. Horn, S. Wagner, and E. S. C. Ching, Thermal Boundary Layer Equation for Turbulent Rayleigh-Bénard Convection, Phys. Rev. Lett. **114**, 114302 (2015).

- [30] G. Ahlers, D. Funfschilling, and E. Bodenschatz, Transitions in heat transport by turbulent convection at Rayleigh numbers up to 10¹⁵, New J. Phys. **11**, 123001 (2009).
- [31] S. Horn and O. Shishkina, Rotating non-Oberbeck–Boussinesq Rayleigh-Bénard convection in water, Phys. Fluids 26, 055111 (2014).
- [32] O. Shishkina, S. Weiss, and E. Bodenschatz, Conductive heat flux in measurements of the Nusselt number in turbulent Rayleigh-Bénard convection, Phys. Rev. Fluids 1, 062301(R) (2016).
- [33] S. Weiss, X. He, G. Ahlers, E. Bodenschatz, and O. Shishkina, Bulk temperature and heat transport in turbulent Rayleigh-Bénard convection of fluids with temperature-dependent properties, J. Fluid Mech. 851, 374 (2018).
- [34] S. Horn and O. Shishkina, Toroidal and poloidal energy in rotating Rayleigh-Bénard convection, J. Fluid Mech. 762, 232 (2015).
- [35] S. Horn and J. M. Aurnou, Regimes of Coriolis-Centrifugal Convection, Phys. Rev. Lett. **120**, 204502 (2018).
- [36] S. Cioni, S. Ciliberto, and J. Sommeria, Strongly turbulent Rayleigh-Bénard convection in mercury: Comparison with results at moderate Prandtl number, J. Fluid Mech. 335, 111 (1997).

- [37] E. Brown, A. Nikolaenko, and G. Ahlers, Reorientations of the Large-Scale Circulation in Turbulent Rayleigh-Bénard Convection, Phys. Rev. Lett. **95**, 084503 (2005).
- [38] T. Vogt, S. Horn, A. M. Grannan, and J. M. Aurnou, Jump rope vortex in liquid metal convection, Proc. Natl. Acad. Sci. U.S.A. 115, 12674 (2018).
- [39] O. Shishkina, S. Wagner, and S. Horn, Influence of the angle between the wind and the isothermal surfaces on the boundary layer structures in turbulent thermal convection, Phys. Rev. E 89, 033014 (2014).
- [40] J. Proudman, On the motion of solids in a liquid possessing vorticity, Proc. R. Soc. 92, 408 (1916).
- [41] G. I. Taylor, Experiments with rotating fluids, Proc. R. Soc. 100, 114 (1921).
- [42] R. Kunnen, H. Clercx, and G. van Heijst, The structure of sidewall boundary layers in confined rotating Rayleigh-Bénard convection, J. Fluid Mech. 727, 509 (2013).
- [43] See Supplemental Material at http://link.aps.org/ supplemental/10.1103/PhysRevLett.124.084505 for evolution of the vorticity and temperature fields of the BZF.

Correction: The license statement contained an omission and has been fixed.