Generalized Knudsen Number for Unsteady Fluid Flow

V. Kara, V. Yakhot, and K. L. Ekinci*

Department of Mechanical Engineering, Boston University, Boston, Massachusetts 02215, USA

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We explore the scaling behavior of an unsteady flow that is generated by an oscillating body of finite size in a gas. If the gas is gradually rarefied, the Navier-Stokes equations begin to fail and a kinetic description of the flow becomes more appropriate. The failure of the Navier-Stokes equations can be thought to take place via two different physical mechanisms: either the continuum hypothesis breaks down as a result of a finite size effect or local equilibrium is violated due to the high rate of strain. By independently tuning the relevant linear dimension and the frequency of the oscillating body, we can experimentally observe these two different physical mechanisms. All the experimental data, however, can be collapsed using a single dimensionless scaling parameter that combines the relevant linear dimension and the frequency of the body. This proposed Knudsen number for an unsteady flow is rooted in a fundamental symmetry principle, namely, Galilean invariance.

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The Navier-Stokes (NS) equations of hydrodynamics can be obtained perturbatively from the kinetic theory of gases in the limit of small Knudsen number, Kn = $(\lambda/\mathcal{L}) \to 0$ [1]. Here, λ is the mean free path in the gas, and \mathcal{L} represents a characteristic length scale of the flow. As $Kn \rightarrow 0$, it follows from statistical mechanics that density fluctuations in the gas vanish [2], leading to the notion of a "fluid particle." This continuum hypothesis becomes less accurate as Kn grows, eventually leading to the failure of the NS equations for Kn \gtrsim 0.1. Likewise, the NS equations break down if the local value of the strain rate, S_{ij} = $\frac{1}{2}[(\partial u_i/\partial x_i) + (\partial u_i/\partial x_i)],$ becomes so large that the condition $\tau S_{ij} \ll 1$ no longer holds. Here, u_i represents the velocity vector, and τ is the relaxation time that characterizes the rate of decay of a perturbation to thermodynamic equilibrium. As τS_{ij} grows, the fluid particle becomes deformed on shorter and shorter time scales, eventually violating the local equilibrium assumption. For a broad class of flows, breakdown of the continuum hypothesis and violation of local equilibrium can be thought to be equivalent, because $\tau S_{ij} \sim \tau(U/\mathcal{L}) \sim (\lambda/c)(U/\mathcal{L}) \sim$ Ma \times Kn. Here, the Mach number Ma = U/c compares the speed of sound c to the characteristic flow velocity U, and it is assumed to remain small and slowly varying. Thus, either Kn or τS_{ij} emerges as the relevant scaling parameter for determining the crossover from hydrodynamics to kinetic theory.

To demonstrate the limitations of the above-described widely accepted reasoning, we consider the canonical problem of an *infinite* plate oscillating at a *prescribed* angular frequency ω_0 in a gas (Stokes second problem) [3]. We assume the oscillation amplitude to be small and the geometry to be such that the velocity field is $u_x(x, y, 0) = U_0 \cos \omega_0 t$, $u_y = 0$, and $u_z = 0$. Since the plate is infinite $(l \to \infty)$, the "standard" size-based Knudsen number

 $Kn_l = \lambda/l$ remains zero at all limits and cannot be relevant. The scaling parameter here is the Weissenberg number, Wi = $\omega_0 \tau$ [4,5], and one can recover the correct Knudsen number, $Kn_{\delta} = \lambda/\delta$, using the boundary layer thickness, $\delta = \sqrt{2\nu_q/\omega_0}$. (Indeed, $Kn_\delta \sim \sqrt{Wi}$, given the kinematic viscosity is $\nu_q \sim \lambda^2/\tau$.) Regardless, $\tau S_{ij} \approx \tau(U_0/\delta) \sim$ $Ma \times Kn_{\delta}$. Thus, as above, the validity of the NS equations (and the scaling properties of the flow) is determined either by the flow length scale (Kn_{δ}) or by the flow time scale (τS_{ij}) or Wi), and both parameters lead to the same conclusion. While this analysis for an infinite plate is reasonable, it does not work for a finite plate (or a finitesized body). For a finite-sized body, Kn₁ may be nonzero at some limit and appear in the problem alongside Wi. This is because the oscillation frequency ω_0 is in general independent of the linear dimensions of the body and an externally prescribed parameter. Recent literature on scaling of such flows reflects this complexity: some reports suggest Kn_1 scaling [6–8] and others Wi scaling [4,9,10]. The purpose of the present work is to study this nontrivial limit and to recover, both experimentally and theoretically, the universal scaling hidden in the apparent contradictions.

Our experimental measurements are based on quartz crystals, and micro- and nanomechanical resonators. When driven to oscillations in a gas, these structures generate oscillatory flows and dissipate energy. The gases used are high-purity He, N₂, and Ar. The approximate equation of motion of a mechanical resonator (in any resonant mode) is that of a damped harmonic oscillator: $\ddot{\xi} + (\omega_0/Q_t)\dot{\xi} + \omega_0^2\xi = \mathcal{F}(t)/m_r$, where $\xi(t)$ is the amplitude, m_r is the mass, $1/Q_t$ is the total (dimensionless) dissipation, and $\omega_0 = 2\pi f_0$ is the angular frequency of the mode driven by the sinusoidal force $\mathcal{F}(t)$. In a typical experiment, the pressure p of the gas is changed, and $1/Q_t$ and ω_0 are measured. For all practical purposes, ω_0 stays constant

through p sweeps. To obtain the (dimensionless) gas dissipation $1/Q_g$, we calculate $1/Q_g = (1/Q_t) - (1/Q_0)$, where $1/Q_0$ is the intrinsic dissipation (obtained at the lowest p). Relevant parameters of our resonators and other details can be found in the Supplemental Material [11].

All our $1/Q_g$ vs p data possess similar features [Figs. 1(a), 2(a), 3(a), 3(b), and S2–S10]. At low p, $(1/Q_g) \propto p$. This is the kinetic limit [31,32], where the mean free path λ and the relaxation time τ of the gas are both large. At high p, the NS equations are to be used [3].

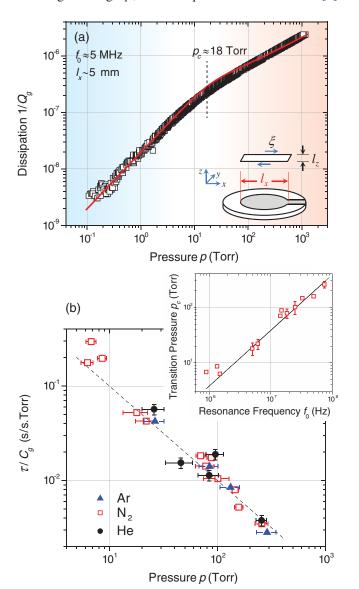


FIG. 1. (a) Dissipation in N_2 as a function of pressure for a quartz crystal (inset) oscillating in shear mode at $f_0 \approx 5$ MHz. Solid line is a fit to Eq. (1). Transition from the kinetic to viscous regime occurs at $p_c \approx 18$ Torr. (b) The inset shows p_c vs f_0 for different quartz crystals in N_2 (Kn $_l \approx 0$). The linear fit gives the empirical τ as a function of p. The main figure shows τ/\mathcal{C}_g for He, N_2 , and Ar as a function of p. Normalization by \mathcal{C}_g accounts for the differences between gases [11]. Dashed line is 1/p. Error bars are not shown when smaller than symbols.

The crossover between these two asymptotes (transitional flow regime) manifests itself as a slope change in the data. The pressure p_c , around which this transition occurs, is therefore a fundamentally important parameter and provides insight into how this flow scales. (p_c , τ_c , and λ_c henceforth indicate transition values.)

We first analyze the dissipation of a macroscopic quartz crystal resonator in shear-mode oscillations in N₂ (Fig. 1(a)). The resonance frequency is $f_0 = (\omega_0/2\pi) \approx$ 5 MHz, and the relevant linear dimension is roughly the diameter of the metal electrode on the quartz, $l_x \sim 5$ mm [Fig. 1(a), inset]. For the shown pressures, $Kn_l = \lambda/l_x$ is in the range $10^{-5} \lesssim \text{Kn}_l \lesssim 10^{-1}$, found using $\lambda \approx (k_B T)/(\sqrt{2\pi} d_g^2 p)$, where $k_B T$ is the thermal energy and d_q is the diameter of a N_2 molecule. Because Kn_l remains small, we treat the quartz as an infinite plate and Wi = $\omega_0 \tau$ is left as the only relevant scaling parameter. The transition from molecular flow ($\omega_0 \tau \gg 1$) to viscous flow $(\omega_0 \tau \ll 1)$ must take place at Wi = $\omega_0 \tau_c \approx 1$. Hence, we call this the "high-frequency limit." Next, we perform the same $1/Q_q$ vs p measurement on similarly large quartz resonators but with different f_0 . We determine p_c consistently for all by finding the pressure at which $1/Q_q$ deviates from the low-p asymptote by 25%. The inset of Fig. 1(b) shows the measured p_c values in N_2 as a function of f_0 . The data scale as $p_c = \text{constant} \times f_0$. This is consistent with the flow being scaled by Wi = $\omega_0 \tau$ and $\omega_0 \tau_c \approx 1$ determining the transition: $\tau = C_{\rm N_2}/p$ for a nearideal gas with $C_{\rm N_2}$ being a constant; $\omega_0 \tau_c \approx \omega_0 (C_{\rm N_2}/p_c) \approx 1$, and $p_c \approx 2\pi C_{\rm N_2} \times f_0$. The experiment provides the *empiri*cal value $C_{N_2} = 610 \pm 30 \times 10^{-9}$ in units of s·Torr. Repeating the same experiment for He and Ar, we find $\mathcal{C}_{He}=560\pm70\times10^{-9}$ and $\mathcal{C}_{Ar}=750\pm80\times10^{-9},$ both in units of s · Torr. Figure 1(b) (main) is a collapse plot of τ/\mathcal{C}_q for all three gases as a function of p, showing the degree of linearity. The measured values of C_q for all gases are a factor of ~5 larger than the kinetic theory predictions [11,33].

The data in Fig. 1(a) can be fit accurately [4]. For a large plate resonator (Kn_l \approx 0), the dissipation in a gas of viscosity μ_g and density ρ_g can be found as [11,30]

$$\frac{1}{Q_g} = \frac{S_r}{m_r} f(\omega_0 \tau) \sqrt{\frac{\mu_g \rho_g}{2\omega_0}}.$$
 (1)

Here, S_r is the surface area and m_r is the mass of the plate resonator, and f is the scaling function [30] found as $f(x) = [1/(1+x^2)^{3/4}]\{(1+x)\cos[(\tan^{-1}x)/2] - (1-x)\sin[(\tan^{-1}x)/2]\}$. The fit in Fig. 1(a) was obtained using the empirical relation $\tau = (610 \times 10^{-9} [\text{s} \cdot \text{Torr}])/p$ and experimental parameters [11].

Now, we turn to the "low-frequency limit" of $\omega_0 \tau \to 0$. Figure 2(a) shows the pressure-dependent dissipation of a low-frequency microcantilever with linear dimensions $l_x \times l_y \times l_z \approx 32 \times 350 \times 1~\mu\text{m}^3$ [inset Fig. 2(a)] and

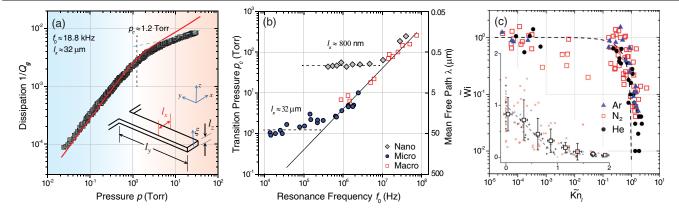


FIG. 2. (a) Dissipation vs pressure for a microcantilever (inset) with $l_x \times l_y \times l_z \approx 32 \times 350 \times 1~\mu\text{m}^3$ and $f_0 \approx 18.8~\text{kHz}$ in N_2 . Solid line is a fit to Eq. (4); dotted (blue) line is a fit to the cylinder solution; $p_c \approx 1.2~\text{Torr.}$ (b) $p_c \text{ vs } f_0$ in N_2 for three sets of devices with different characteristic dimensions. Diamonds are nanocantilevers from Ref. [29]; circles are microcantilevers; squares are macroscopic resonators from Fig. 1(b). (c) Wi and $\tilde{\text{Kn}}_l$ in He, N_2 , and Ar for all devices. Dashed line is $\tilde{\text{Wi}} + \tilde{\text{Kn}}_l = 1$. The inset shows the same data using linear axes; the large data points correspond to binned average values.

frequency $f_0=18.8$ kHz. We define $\mathrm{Kn}_l=\lambda/l_x$, as suggested in Refs. [7,34,35]. The transition in Fig. 2(a) takes place around $p_c\approx 1.2$ Torr, where $\mathrm{Kn}_l\approx 1$ and $\omega_0\tau\approx 0.06$. ($\mathrm{Kn}_l\approx 1$ indicates deviation from the low-p molecular asymptote.) The features in Fig. 2(a) are very similar to those in Fig. 1(a): two asymptotes with a well-defined p_c . Inspection of the ranges of Wi and Kn_l suggests that the transition cannot be tied to frequency (Wi) but must be due to the length scale (Kn_l). In other words, the transition from molecular flow ($\mathrm{Kn}_l\gg 1$) to viscous flow ($\mathrm{Kn}_l\ll 1$) appears to take place around $\mathrm{Kn}_l=\lambda_c/l_x\approx 1$. While the data trace in Fig. 2(a) looks similar to that in Fig. 1(a), the transitions observed in the two are due to different physical mechanisms.

In Fig. 2(b), we plot the consistently found p_c in N_2 for different sets of devices. Here, the relevant linear dimension l_x is kept constant for each set, but the frequency is varied: diamond nanocantilevers [29] with $l_x \approx 800 \text{ nm}$ and 0.4 MHz $\leq f_0 \leq$ 40 MHz; silicon microcantilevers with $l_x \approx 32 \ \mu \text{m}$ and $14 \ \text{kHz} \le f_0 \le 2.4 \ \text{MHz}$; and quartz crystals with $l_x \sim 5 \text{ mm}$ and $5 \text{ MHz} \leq f_0 \leq 75 \text{ MHz}$. Surprisingly, the linear trend between p_c and f_0 holds only for high frequencies, with a saturation at low frequencies. The saturation value of p_c is determined by the condition that $\lambda \sim l_x$ (dotted horizontal lines). The oscillation frequency (and Wi) becomes the relevant scaling parameter above a certain frequency; at low frequency, the length scale (Kn₁) takes over. Thus, the physics is determined by an interplay between the relevant length scale of the body and its oscillation frequency.

To gain more insight into the transition, we scrutinize $\tilde{\mathrm{Kn}}_l = \lambda_c/l_x$ and $\tilde{\mathrm{Wi}} = \omega_0 \tau_c$ for each device at its p_c . Figure 2(c) shows $\tilde{\mathrm{Kn}}_l$ and Wi plotted in the xy plane using logarithmic and linear axes (inset); the dashed lines are $\tilde{\mathrm{Wi}} + \tilde{\mathrm{Kn}}_l = 1$. The data suggest that the dissipation is a function of both Wi and Kn_l , and it approximately depends on $\mathrm{Wi} + \mathrm{Kn}_l$.

We now justify the observed scaling more rigorously by inspecting the stress tensor σ_{ij} obtained from the Chapman-Enskog expansion of the Boltzmann equation in the relaxation time approximation. To second order of smallness, the expansion is [36]

$$\sigma_{ij} \approx \sigma_{ij}^{(1)} + \sigma_{ij}^{(2)} = 2\rho_g \theta \left[\tau S_{ij} - \tau \left(\frac{\partial}{\partial t} + \mathbf{u} \cdot \nabla \right) (\tau S_{ij}) + 2\tau^2 \left(S_{ik} S_{kj} - \frac{\delta_{ij}}{3} S_{kl} S_{kl} \right) - 2\tau^2 (S_{ik} \Omega_{kj} + S_{jk} \Omega_{ki}) \right].$$
 (2)

As usual, $S_{ij} = \frac{1}{2} [(\partial u_i/\partial x_j) + (\partial u_j/\partial x_i)]$ and $\Omega_{ij} = \frac{1}{2} [(\partial u_i/\partial x_j) - (\partial u_j/\partial x_i)]$ are the strain rate and the vorticity tensors, respectively, with i, j = x, y, z, and $\theta = k_B T/m_g$. The last two terms of σ_{ij} are the second rank tensor $\xi_{ij}^{(2)}$ of order $(\tau \mathbf{S})^2$, where \mathbf{S} represents the strain rate tensor. There are two dimensionless groups in Eq. (2): the total time derivative $\tau(d/dt) = \tau[(\partial/\partial t) + \mathbf{u} \cdot \nabla]$ and $\tau \mathbf{S}$. One notices that these two dimensionless groups both remain invariant under Galilean transformations [11]. In order to satisfy Galilean invariance, therefore, the Chapman-Enskog expansion of kinetic equations must be in powers of these parameters only; powers of non-Galilean-invariant parameters, e.g., "bare" $\partial/\partial t$, are forbidden in a flow in an arbitrary geometry. Accordingly, one can formally write the Galilean-invariant stress tensor up to all orders as

$$\sigma_{ij} = 2\rho_g \theta \left\{ \tau S_{ij} + \sum_{n=2}^{\infty} \left[\alpha_{n-1} (-\tau)^{n-1} \left(\frac{\partial}{\partial t} + \mathbf{u} \cdot \nabla \right)^{n-1} (\tau S_{ij}) + \xi_{ij}^{(n)} \right] \right\}.$$
(3)

Here, α_{n-1} are constants, and the tensors $\xi_{ij}^{(n)} \sim (\tau \mathbf{S})^n$ are not necessarily zero [37].

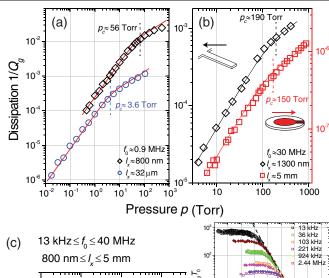
A closed form formula can be obtained for the dissipation of a finite-sized body oscillating in a fluid, if the deviations from the infinite plate solution [30] are assumed small. As in the infinite plate [11,30], we set all $\alpha_k \approx 1$ and all $\xi_{ij}^{(n)} \approx 0$ in Eq. (3). After nondimensionalization with $\hat{\bf u} = {\bf u}/c$, $\hat{t} = \omega_0 t$ and $\hat{\nabla} = l\nabla$, the stress tensor σ_{ij} for a finite-sized body becomes an expansion in powers of the operator $\tau(d/dt) = \omega_0 \tau(\partial/\partial \hat{t}) + {\rm Kn}_l \hat{\bf u} \cdot \hat{\nabla}$. The scaling parameter therefore becomes approximately $\omega_0 \tau + {\rm Kn}_l$, and the infinite plate solution in Eq. (1) can be generalized by replacing $\omega_0 \tau$ with $\omega_0 \tau + {\rm Kn}_l$. Thus, we deduce [11]

$$\frac{1}{Q_g} \approx \frac{S_r}{m_r} f\left(\omega_0 \tau + \frac{\lambda}{l_x}\right) \sqrt{\frac{\mu_g \rho_g \tau}{2(\omega_0 \tau + \frac{\lambda}{l_x})}} \tag{4}$$

for a finite-sized body oscillating in a fluid. Several points are noteworthy. First, Eq. (4) is valid in the asymptotic and the intermediate ranges. Second, the nondimensionalization above is eminently reasonable, because the only velocity scale in kinetic theory is the thermal velocity $\sim c$. Regardless, the dimensional solution is obtained only after imposing the boundary conditions. Finally, Galilean invariance dictates the form of d/dt and leads to a scaling parameter $\approx \text{Wi} + Kn_l$, instead of a more involved combination of Wi and Kn_l.

A number of fits to experimental data using Eq. (4) are shown in Figs. 2(a), 3(a), and 3(b), as well as in the Supplemental Material [11]. The data in Figs. 3(a) and 3(b) are examples of the low- and high-frequency limits, respectively. Here, different-sized but similar-frequency resonators are compared. All fits are obtained as follows. First, S_r/m_r is determined from linear dimensions or from separate measurements when necessary [11]. For each pressure, the value of $\omega_0 \tau + (\lambda/l_x)$ is computed using $\tau =$ C_q/p and $\lambda \approx 0.23(k_BT/d_q^2p)$ of the gas, and l_x and ω_0 of the resonator. Finally, the dissipation is found from Eq. (4) at each pressure using tabulated μ_g and ρ_g , and our empirical τ . To improve the fits, the theoretical prediction is multiplied by an $\mathcal{O}(1)$ constant \mathcal{Q}_p . The collapse plot in Fig. 3(c) is obtained by properly dividing the data by $(S_r/m_r)\sqrt{\{(\mu_q\rho_q\tau)/2[\omega_0\tau+(\lambda/l_x)]\}}Q_p$ and plotting the results as a function of $\omega_0 \tau + (\lambda/l_x)$. The thick solid line shows $f(Wi + Kn_l)$. There are no free parameters other than the fitting factors Q_p with mean $\bar{Q}_p \approx 2.6 \pm 0.5$ [11].

At the viscous limit Wi + $Kn_l \ll 1$, the cantilever data deviate from the plate solution and converge to a cylinder solution. The cylinder solution yields $1/Q_g \approx [\Gamma_I(\mathrm{Re}_\omega)]/[1/T_0 + \Gamma_R(\mathrm{Re}_\omega)]$ [12,13]. Here, $\Gamma(\mathrm{Re}_\omega) = \Gamma_R(\mathrm{Re}_\omega) + i\Gamma_I(\mathrm{Re}_\omega)$ is the complex hydrodynamic function for a cylinder and only depends upon the (oscillatory) Reynolds number $\mathrm{Re}_\omega = (\omega_0 l_x^2)/4\nu_g$; $T_0 = (\pi/4)(\rho_g l_x/\rho_r l_z)$ with ρ_r being the density of the solid



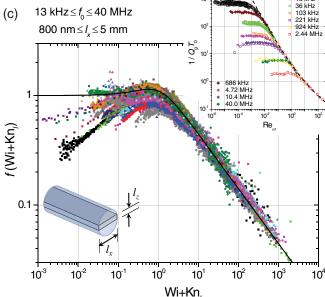


FIG. 3. (a) Dissipation vs p for two cantilevers with different length scales but similar frequencies ($l_x \approx 800\,\mathrm{nm}$, $f_0 \approx 894\,\mathrm{kHz}$; and $l_x \approx 32~\mu\mathrm{m}$, $f_0 \approx 924~\mathrm{kHz}$) in N₂. Transitions are determined by Kn_l ≈ 1 at $p_c \approx 56$ and 3.6 Torr, respectively. (b) Dissipation for a nanocantilever ($l_x \approx 1300\,\mathrm{nm}$, $f_0 \approx 28.6~\mathrm{MHz}$) and a macroscopic quartz crystal ($l_x \sim 5~\mathrm{mm}$, $f_0 \approx 32.7~\mathrm{MHz}$ [4]); the transitions take place around 190 and 150 Torr, respectively. (c) Collapse plot for all the data in different gases. The thick solid line shows the scaling function f. The inset is a collapse of select cantilever data based on the viscous cylinder solution. Squares and diamonds correspond to microcantilevers ($l_x \approx 32~\mu\mathrm{m}$) and nanocantilevers ($l_x \approx 800~\mathrm{nm}$), respectively. Dashed line shows the imaginary part of the complex hydrodynamic function for a cylinder. The lower inset shows the parameters of the model.

[Fig. 3(c) lower inset]. For our gas experiments, $1/T_0 \gtrsim 1000 \gg \Gamma_R$, and thus $1/Q_gT_0 \approx \Gamma_I(\mathrm{Re}_\omega)$. The upper inset of Fig. 3(c) shows $1/Q_gT_0$ from representative cantilevers with different parameters plotted against Re_ω ; the dashed line shows $\Gamma_I(\mathrm{Re}_\omega)$. In each case, a fitting constant Q_c with mean $\bar{Q}_c \approx 0.9 \pm 0.2$ is used [11]. The data converge to the cylinder solution in the viscous regime.

We conclude that the scaling parameter for an arbitrary time-dependent isothermal flow should be a function of both Wi and Kn_l . We show that a generalized Knudsen number in the form $Wi + Kn_l$ works well and can be justified by Galilean invariance.

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- ekinci@bu.edu
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