

Stability and Angular-Momentum Transport of Fluid Flows between Corotating Cylinders

M. Avila*

Max Planck Institute for Dynamics and Self-Organization (MPIDS), 37077 Göttingen, Germany and Institute of Fluid Mechanics, Friedrich-Alexander-Universität Erlangen-Nürnberg, Cauerstraße 4, 91058 Erlangen, Germany

(Received 5 October 2011; published 19 March 2012)

Turbulent transport of angular momentum is a necessary process to explain accretion in astrophysical disks. Although the hydrodynamic stability of disklike flows has been tested in experiments, results are contradictory and suggest either laminar or turbulent flow. Direct numerical simulations reported here show that currently investigated laboratory flows are hydrodynamically unstable and become turbulent at low Reynolds numbers. The underlying instabilities stem from the axial boundary conditions, affect the flow globally, and enhance angular-momentum transport.

DOI: 10.1103/PhysRevLett.108.124501

PACS numbers: 47.20.-k, 47.54.-r, 95.30.Lz

Accretion in astrophysical disks requires the flow of mass towards a central gravitating body. The ensuing loss of momentum must be balanced by outward angular-momentum transfer among gas particles [1]. If the motion of orbiting gas was laminar, molecular transport would be orders of magnitude too slow for accretion to take place, and so considering a turbulent viscosity becomes necessary [2]. However, in Keplerian disks the gas rotates as $\Omega \propto r^{-3/2}$ and laminar motion is linearly stable according to the Rayleigh criterion. Although axial magnetic fields can drive turbulence via the magnetorotational instability [3], it is not clear whether this operates in weakly ionized disks. On the other hand, it is well known that linearly stable shear flows (such as pipe flow) can become turbulent due to finite amplitude disturbances. Whether Keplerian flows are susceptible to such a transition scenario or remain stable despite the large Reynolds numbers (Re) is a topic of great interest and the source of much controversy [4].

The stability of disklike flows is typically probed in laboratory experiments of fluid between two concentric and independently rotating cylinders, Taylor-Couette flow (TCF). In the infinite-cylinder idealization, the Navier-Stokes equations admit a pure rotary solution

$$\Omega(r) = \frac{\Omega_2 r_2^2 - \Omega_1 r_1^2}{r_2^2 - r_1^2} + \frac{(\Omega_1 - \Omega_2)(r_1 r_2)^2}{r_2^2 - r_1^2} \frac{1}{r^2}, \quad (1)$$

commonly referred to as Couette flow. Here r_1 and r_2 are the radii of the inner and outer cylinders and Ω_1 and Ω_2 their angular velocities. When $(r_1/r_2)^2 < \Omega_2/\Omega_1 < 1$ the angular velocity decreases outward but the angular momentum increases (quasi-Keplerian flows). Accretion disks are stratified in the axial direction and are thus best modeled considering an unbounded domain, thereby avoiding artificial boundary conditions [5]. Most experiments and simulations focus, however, on the physics of the disk's midplane and neglect stratification. Under this assumption simulations typically employ axially periodic boundary conditions, whereas in experiments cylinders have a

finite-length h . Hence the degree to which (1) may be approximated is compromised by the axial boundary conditions and length-to-gap aspect ratio $\Gamma = h/(r_2 - r_1)$. In particular, solid axial boundaries result in a basic state with nonzero radial and axial velocity components (Ekman flow). Hence, producing Couette-like profiles in experiments poses an extraordinary challenge, which may be addressed by considering very tall cylinders or splitting the end walls in several rings that rotate at independent angular speeds [6]. The latter strategy has been implemented in the Princeton Taylor-Couette experiment [7], which has a short aspect ratio $\Gamma = 2.104$ but whose end walls are split into two independently rotating rings. From simultaneous laser Doppler velocity measurements of azimuthal and radial velocity components, Ji *et al.* [8] suggest that quasi-Keplerian flows at $\text{Re} = O(10^6)$ are essentially laminar. From this, they conclude that purely hydrodynamic mechanisms cannot transport angular momentum at the rates required for accretion to occur in disks.

This conclusion has been recently challenged in a new experimental study by Paoletti and Lathrop [9], who report from direct torque measurements at the inner cylinder that Keplerian flows at $\text{Re} \geq 10^6$ are fully turbulent. When extrapolated to astrophysical disks, their results indicate that transport occurs at accretion relevant rates, in agreement with previous estimations [10]. Despite having tall cylinders $\Gamma = 11.47$, the Maryland experiment has solid end walls that are attached to the outer cylinder and hence cannot be rotated independently. Although these boundary conditions are known to generate vigorous Ekman vortices and greatly increase the exerted torque, their contribution is discarded by dividing the inner cylinder into three sections and measuring torque only in the central one. Despite efforts in the Princeton and Maryland experimental setups to mitigate end wall effects, it is, however, unclear whether their results can be used to infer the stability of flows in astrophysical disks [4]. In this Letter it is shown that current laboratory experiments of quasi-Keplerian flows become turbulent already at $\text{Re} = O(10^3)$ due to

hydrodynamic instabilities stemming from the axial boundary conditions. Moreover, it is found that turbulence fills the entire flow domain and as a result the momentum transfer is globally enhanced.

Here direct numerical simulations of flows with the precise geometry and boundary conditions of the Princeton and Maryland experiments were performed. The Navier-Stokes equations for an incompressible Newtonian fluid of velocity \mathbf{v} ,

$$\partial_t \mathbf{v} + (\mathbf{v} \cdot \nabla) \mathbf{v} = -\nabla p + \Delta \mathbf{v}, \quad \nabla \cdot \mathbf{v} = 0, \quad (2)$$

were rendered dimensionless by scaling lengths and time with the gap width $d = r_2 - r_1$ and viscous time d^2/ν , where ν is the kinematic viscosity of the fluid. The solution of (2) was formulated in primitive variables in cylindrical coordinates (r, θ, z) and a second-order time-splitting method with consistent boundary conditions for the pressure was used [11]. The spatial discretization consists of Chebyshev collocation in (r, z) and a Galerkin-Fourier expansion in θ . The code converges spectrally in the three directions [12] and was validated against a Legendre-Fourier-Galerkin code [13]. Here the resolution was chosen to ensure that computed torque values were accurate to at least 1%.

The geometry of the Taylor-Couette system is specified by the radii ratio $\eta = r_1/r_2$ and the length-to-gap aspect ratio Γ . The dimensionless boundary conditions at the cylinders read $(v_r, v_\theta, v_z)[r_{1,2}, \theta, z] = (0, \text{Re}_{1,2}, 0)$, where $\text{Re}_1 = dr_1 \Omega_1/\nu$ ($\text{Re}_2 = dr_2 \Omega_2/\nu$) is the inner (outer) cylinder Reynolds number. Because of differential rotation the angular velocity changes abruptly at adjacent rotating boundaries. In the Princeton experiment, the end wall is split at midradius $r_m = (r_1 + r_2)/2$ into two independently rotating rings. Hence, there are four independent angular speeds: Ω_1 and Ω_2 for inner and outer cylinder and Ω_3 and Ω_4 for inner and outer rings. To preserve spectral convergence discontinuities in angular velocity were regularized, yielding the following boundary condition at the end walls $z = \pm\Gamma/2$:

$$\begin{aligned} \Omega(r) = & (\Omega_1 - \Omega_3) \exp[-(r - r_1)/\epsilon] \\ & + (\Omega_2 - \Omega_4) \exp[-(r_2 - r)/\epsilon] + \frac{\Omega_3 + \Omega_4}{2} \\ & + \frac{\Omega_4 - \Omega_3}{2} \tanh[(r - r_m)/\epsilon], \end{aligned}$$

with $\epsilon \in [5 \times 10^{-3}, 10^{-2}]$ (see Ref. [14]). The boundary condition modeling the Princeton experiment [8] is shown as circles in Fig. 1(a). Because of the sharp gradient $\partial\Omega/\partial r|_{r_m}$ and the clustering of Chebyshev points close to the boundaries, a large number of radial points ($n_r = 351$) was required to accurately simulate the split end wall. In the axial and azimuthal directions up to $n_z = 281$ Chebyshev points and $n_\theta = 256$ Fourier modes were used. The Maryland experiment has a single solid ring attached to the outer cylinder ($\Omega_4 = \Omega_3 = \Omega_2$) and

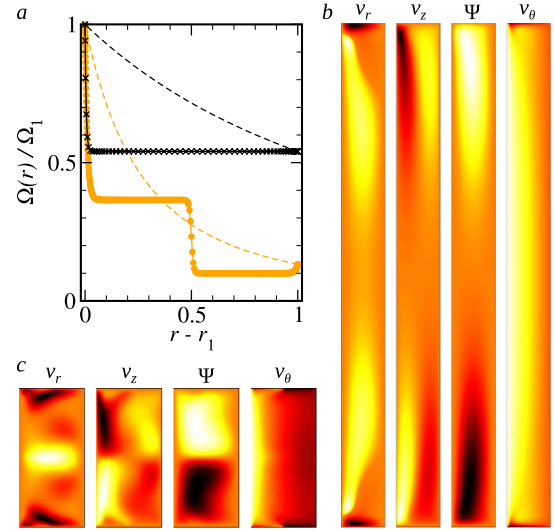


FIG. 1 (color online). (a) Angular velocity at the end walls for the Princeton (circles) and Maryland (crosses) experiments, and Couette flow (dashed lines). (b),(c) Steady basic states at $\text{Re} = 320$ (Maryland) and $\text{Re} = 772$ (Princeton). White (black) corresponds to maximum (minimum) velocity and the radial direction is horizontal with left (right) corresponding to inner (outer) cylinder.

there is only a strong gradient at r_1 [see crosses in Fig. 1(a)]. Here up to $n_z = 601$, $n_\theta = 384$, and $n_r = 61$ were used.

To put TCF in the wider context of rotating shear flows it is useful to define a shear Reynolds number $\text{Re} = 2/(1 + \eta)|\text{Re}_2 \eta - \text{Re}_1|$ and a rotation number $R_\Omega = (1 - \eta) \times (\text{Re}_1 + \text{Re}_2)/(\text{Re}_2 \eta - \text{Re}_1)$, which measure the ratio of shear to viscous forces and the ratio of mean rotation to shear [15], respectively. Here the sign of R_Ω distinguishes between cyclonic ($R_\Omega > 0$) and anticyclonic flows ($R_\Omega < 0$), with $-2 < R_\Omega < -1$ corresponding to quasi-Keplerian rotation. In the experiments of Ji *et al.* [8], $R_\Omega = -1.038$ and $\eta = 0.3478$, and the same values were used here, whereas Lathrop and Paoletti [9] have systematically studied both cyclonic and anticyclonic regimes at $\eta = 0.7245$. Here, $\eta = 0.7245$ and the value $R_\Omega = -1.047$ was chosen [corresponding to their Rossby number $\text{Ro} = \text{Re}_1/(\eta \text{Re}_2) - 1 = 0.85$].

The end wall influence in the Maryland experiment is illustrated in Fig. 1(b), showing the velocity field of the steady basic state at $\text{Re} = 320$. At the end walls there is a strong negative radial velocity inflow, which generates axial velocities pointing towards midheight along the inner cylinder and result in an axially dependent azimuthal velocity. Figure 1(c) shows the basic state of the Princeton experiment at $\text{Re} = 772$. Because of the small aspect ratio the meridional circulation generates a strong radial outward flow at midheight that increases the outward transport of azimuthal velocity. Were the angular speeds of the end wall rings selected according to the ideal Couette

profile [6,16] instead of the values used in experiments [8] and reproduced here, profiles with weaker meridional circulation and hence closer to Couette flow could be obtained. This approach was used in previous numerical simulations of the Princeton configuration [17].

The visualizations of Figs. 1(b) and 1(c) hint at the difficulty of realizing quasi-Keplerian profiles in a laboratory experiment even at very low Reynolds numbers. The end wall boundary conditions change the velocity field not just locally but globally across the domain. In fact, at slightly higher Reynolds numbers the flow becomes three dimensional and time dependent via supercritical Hopf bifurcations. In the Maryland configuration instability occurs first at $Re_c = 352$ to prograde rotating waves with azimuthal wave number $m = 5$ and localized at the end walls. Beyond Re_c multiplicity of states, with different symmetries and wave number $m \in [2, 5]$, was found, whereas for $Re \geq 1330$ only a global $m = 2$ mode remained stable and was obtained regardless of initial conditions [see Fig. 2(a)]. Further increasing the Reynolds number led to modulated waves and a quick transition to temporal chaos at about $Re \approx 1600$. Subsequently, spatial periodicity was lost and the spectra broadened as the flow became gradually turbulent; Fig. 2(b) shows a flow snapshot at $Re = 5328$. This transition picture is also representative of the simulations of the Princeton experiment. Here the basic steady state becomes unstable at $Re = 1448$ almost simultaneously to $m = 1$ and $m = 2$ rotating waves, which were found to coexist in space and time [see Fig. 2(c)]. By no means is this situation generic: changing the relative rotation of the cylinders one of $m = 1, 2$ was found to bifurcate first. Further increasing the Reynolds number led to very complex and strongly three-dimensional flow as shown in Fig. 2(d).

The stability of quasi-Keplerian TCF with end walls attached to the outer cylinder and $\eta = 0.7245$ is shown in Fig. 3(a). The minimum critical Reynolds number is

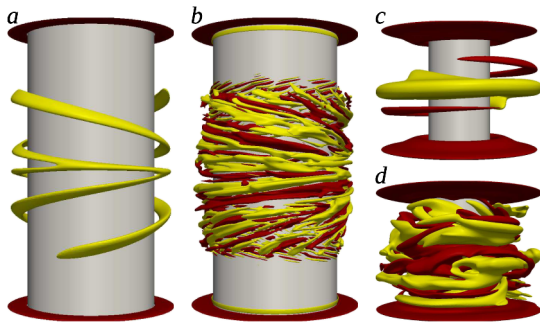


FIG. 2 (color online). Three-dimensional view of isosurfaces of negative [dark gray (red)] and positive [light gray (yellow)] radial velocity. (a) $m = 2$ rotating wave at $Re = 1332$ and (b) turbulent flow at $Re = 5328$ from simulations of the Maryland experiment. (c) Modulated rotating wave with $m = 1$ and $m = 2$ at $Re = 1545$ and (d) turbulent flow at $Re = 6437$ from simulations of the Princeton experiment.

attained at the Rayleigh line ($R_\Omega = -1$) and increases as differential rotation decreases, but with $Re_c < 10^4$ across the whole quasi-Keplerian regime. Close to the Rayleigh line modes localized to the end walls bifurcate first, whereas for $R_\Omega \lesssim -1.1$ global instability modes [as in Fig. 2(a)] dominate. The former are similar to those observed experimentally in Ref. [18] and the latter are similar to those reported by Avila *et al.* [12], who studied global boundary layer effects on flows between exactly corotating cylinders and stationary end walls. Figure 3(b) further shows that end wall instabilities depend weakly on geometry and hence generically govern the dynamics of quasi-Keplerian TCF. It is worth noting that end wall instabilities persist beyond the Rayleigh line and coexist with Taylor vortices close to the onset of centrifugal instability.

The onset of hydrodynamic instability and transition to turbulence are expected to radically change the radial transport of azimuthal momentum. The solid lines in Fig. 4(a) show normalized average azimuthal velocity profiles $\langle v_\theta \rangle_{\theta,t} / (r_1 \Omega_1)$ at midheight for simulations of the Maryland experiment at $Re = 5328$ (black solid line) and the Princeton experiment at $Re = 6437$ [gray (orange) solid line]. At the inner cylinder the profiles are steeper than laminar Couette flow (dashed lines), implying larger torques on the cylinder surface. It is worth noting that in TCF between infinite cylinders the transverse current of azimuthal motion $J^\Omega = r^3[\langle v_r \rangle_{\theta,z,t} - \nu \partial_r \langle \Omega \rangle_{\theta,z,t}]$ is a conserved quantity [19], and as a consequence the dimensionless torque $G = \nu^{-2} J^\Omega$ is the same at the inner and the outer cylinder. This does not hold, however, for flows confined by no-slip axial boundaries. Torque profiles along the inner cylinder, normalized with the laminar Couette torque, are shown in Fig. 4(b). Because of the sharp change in Ω occurring across a small gap between the inner cylinder and end walls [see Fig. 1(a)], the torque required to rotate the inner cylinder faster than the end wall is very large. This increase in local torque as the end walls $2z/\Gamma = \pm 1$ are approached can be seen in Fig. 4(b). Although the direct contribution of the end wall is largely avoided by the measurement technique in the experiments, the torque in

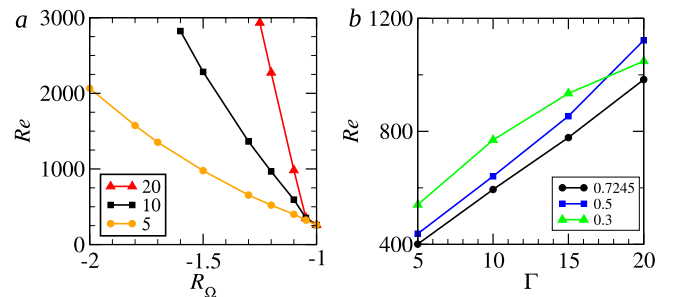


FIG. 3 (color online). Stability curves of quasi-Keplerian Taylor-Couette flows with end walls attached to the outer cylinder: (a) $\eta = 0.7245$ and Γ as in the legend, (b) $R_\Omega = -1.1$ and η as in the legend.

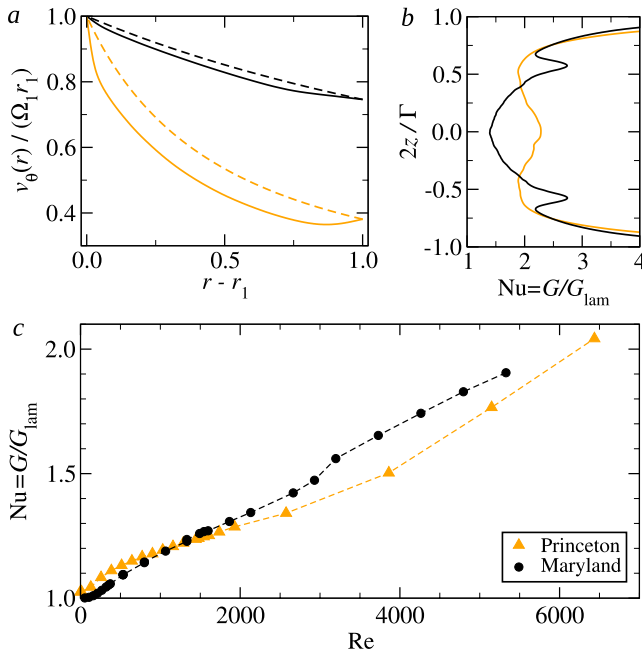


FIG. 4 (color online). (a) Average azimuthal velocity at mid-height for simulations of the Maryland ($Re = 5328$, black solid line) and Princeton [$Re = 6437$, gray (orange) solid line] experiments. The dashed lines are laminar Couette flow. (b) Torque $Nu = G/G_{lam}$ along the inner cylinder, solid lines as in (a). (c) Reynolds number dependence of the torque on the central section of the inner cylinder ($0.6 > |2z/\Gamma|$ and $0.4 > |2z/\Gamma|$ for simulations of the Maryland and Princeton experiments).

the central section remains well above laminar because of turbulent fluctuations.

The simulations of the Maryland experiment show a clear change in the torque behavior at about $Re = 3000$ [see black curve in Fig. 4(c)]. This is related to the appearance of the two torque peaks at $2z/\Gamma \sim \pm 0.5$ in Fig. 4(b) and is caused by the onset of small-scale vortices which have opposite spiral orientation to the structure of the primary rotating wave [see Fig. 2(b)]. In both experiments the torque has already doubled the laminar value at $Re \sim 6000$ due to the end-wall-driven instabilities.

In spite of the disparity in Reynolds numbers, it is tempting to compare these numerical results to experimental observations. van Gils *et al.* [20] have measured torque at $Re > 10^5$ and have observed an effective universal scaling law $Nu = G/G_{lam} = a(R_\Omega)Re^{0.76}$ that holds throughout the linearly unstable regime of TCF. Surprisingly, Paoletti and Lathrop [9] (see also Ref. [21]) have demonstrated that this law applies to linearly stable cyclonic and anti-cyclonic Rayleigh-stable regimes as well, that is including quasi-Keplerian rotation. It is then natural to ask how this universal behavior connects to the complex flows uncovered in this work. It is speculated here that a transition between end-wall-driven turbulence to the universal $Nu \propto Re^{0.76}$ scaling reported in experiments may take place at intermediate Re . Interestingly, in the case of a stationary

outer cylinder a crossover marking the transition from centrifugally to shear-driven turbulence at $Re \approx 13000$ was reported [22]. If an analogous crossover was found in quasi-Keplerian flows and shown to be independent of aspect ratio and end wall boundary condition, a strong case for the existence of hydrodynamic turbulence in astrophysical disks would be made. In fact, ingredients of shear-driven turbulence such as transient growth of disturbances are found also in quasi-Keplerian flows, although significantly only at $Re = O(10^6)$ [23]. On the other hand, it would be interesting to investigate the connection between the complex flows found here and the quiescent flows reported by Ji *et al.* [8] at large Re .

In conclusion, current laboratory experiments designed to approximate flow profiles expected from accretion disks become turbulent at moderate Reynolds number due to imposed boundary conditions. Although these instabilities are generic and hence cannot possibly be avoided, universal scaling suggests that shear mechanisms might overwhelm end wall effects at large Reynolds number. In order to probe this hypothesis new experiments with variable aspect ratio and different axial boundary conditions should be conducted. These would provide great insight on the physical mechanisms of rotating shear flows and might shed light on the origin of turbulence in astrophysical disks.

Support from the Max Planck Society and the Engineering and Physical Sciences Research Council (Grant No. EP/F017413/2) is acknowledged. The author is grateful to Kerstin Avila and Bjoern Hof for discussions.

*mavila@lstm.uni-erlangen.de

- [1] J. Pringle, *Annu. Rev. Astron. Astrophys.* **19**, 137 (1981).
- [2] N.I. Shakura and R.A. Sunyaev, *Astron. Astrophys.* **24**, 337 (1973).
- [3] S.A. Balbus and J.F. Hawley, *Rev. Mod. Phys.* **70**, 1 (1998); S.A. Balbus, *Annu. Rev. Astron. Astrophys.* **41**, 555 (2003).
- [4] S.A. Balbus, *Nature (London)* **470**, 475 (2011).
- [5] J.A. Barranco and P.S. Marcus, *Astrophys. J.* **623**, 1157 (2005).
- [6] R. Hollerbach and A. Fournier, *AIP Conf. Proc.* **733**, 114 (2004).
- [7] E. Schartman, H. Ji, and M.J. Burin, *Rev. Sci. Instrum.* **80**, 024501 (2009).
- [8] H. Ji, M. Burin, E. Schartman, and J. Goodman, *Nature (London)* **444**, 343 (2006).
- [9] M.S. Paoletti and D.P. Lathrop, *Phys. Rev. Lett.* **106**, 024501 (2011).
- [10] D. Richard and J.P. Zahn, *Astron. Astrophys.* **347**, 734 (1999).
- [11] S. Hughes and A. Randriamampianina, *Int. J. Numer. Methods Fluids* **28**, 501 (1998); I. Mercader, O. Batiste, and A. Alonso, *Comput. Fluids* **39**, 215 (2010).
- [12] M. Avila, M. Grimes, J.M. Lopez, and F. Marques, *Phys. Fluids* **20**, 104104 (2008).

- [13] F. Marques and J.M. Lopez, *J. Fluid Mech.* **561**, 255 (2006).
- [14] J.M. Lopez and J. Shen, *J. Comput. Phys.* **139**, 308 (1998).
- [15] B. Dubrulle, O. Dauchot, F. Daviaud, P. Y. Longaretti, D. Richard, and J.P. Zahn, *Phys. Fluids* **17**, 095103 (2005).
- [16] A. Kageyama, H. Ji, J. Goodman, F. Chen, and E. Shoshan, *J. Phys. Soc. Jpn.* **73**, 2424 (2004).
- [17] A. Obabko, F. Cattaneo, and P. Fischer, *Phys. Scr.* **T132**, 014029 (2008).
- [18] C.D. Andereck, S.S. Liu, and H.L. Swinney, *J. Fluid Mech.* **164**, 155 (1986).
- [19] B. Eckhardt, S. Grossmann, and D. Lohse, *J. Fluid Mech.* **581**, 221 (2007).
- [20] D.P.M. van Gils, S.G. Huisman, G.W. Bruggert, C. Sun, and D. Lohse, *Phys. Rev. Lett.* **106**, 024502 (2011).
- [21] M. Paoletti, D. van Gils, B. Dubrulle, C. Sun, D. Lohse, and D. Lathrop, [arXiv:1111.6915](https://arxiv.org/abs/1111.6915).
- [22] D.P. Lathrop, J. Fineberg, and H.L. Swinney, *Phys. Rev. Lett.* **68**, 1515 (1992); G.S. Lewis and H.L. Swinney, *Phys. Rev. E* **59**, 5457 (1999).
- [23] P.A. Yecko, *Astron. Astrophys.* **425**, 385 (2004).