

**Capsule distributions and flow properties in curved tubes**Huiyong Feng,<sup>1</sup> Jianguo Zheng,<sup>2</sup> Bei Wei,<sup>3,4</sup> Jian Hou,<sup>3,4</sup> and Haibo Huang<sup>1,\*</sup><sup>1</sup>*Department of Modern Mechanics, University of Science and Technology of China, Hefei, Anhui 230026, People's Republic of China*<sup>2</sup>*School of Aerospace Engineering, Huazhong University of Science and Technology, Wuhan, Hubei 430074, People's Republic of China*<sup>3</sup>*Key Laboratory of Unconventional Oil & Gas Development (China University of Petroleum (East China)), Ministry of Education, Qingdao 266580, People's Republic of China*<sup>4</sup>*School of Petroleum Engineering, China University of Petroleum (East China), Qingdao 266580, People's Republic of China*

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The distribution of capsules and rheological properties of suspensions in curved tubes are investigated by using an immersed-boundary lattice Boltzmann method. We mainly focus on the effective suspension viscosity and equilibrium positions of capsules as functions of Reynolds number ( $Re$ ), Capillary number ( $Ca$ ) and volume fraction of capsules ( $\phi$ ). We found that at limited inertia ( $Re \mathcal{O}(10)$ ), the effective viscosity decreases with increasing  $Re$ , which is different from the variation trend in straight tubes. Dean's vortices play an important role. When the fluid inertia increases, the vortices are strengthened. They greatly promote the capsules' circumferential transportation by trapping the capsules into their centers and making the location of maximum azimuthal velocity close to them. The curvature effect of the torus vessel is also investigated. When the curvature  $\kappa$  is large enough, e.g.,  $\kappa > 0.2$ , a scaling law for the effective viscosity as a function of a redefined  $Re$  is proposed. Furthermore, the distribution feature of multiple capsules in the torus vessel is revealed. Generally, for the semidilute regime, the capsules concentrate on the symmetrical plane at low  $Re$  but the center of Dean's vortex at high  $Re$ . In addition, for both the dilute and semidilute regime, the scaling law connecting the effective viscosity and the average location of capsules is proposed. Our data support the scaling well. This study may be useful in the design of tubes for capsule transportation.

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Transportation of capsule suspensions in curved tubes is an important problem in physiology and microfluidics. In many situations, capsule suspensions flow through a complicated network of tubes or curved channels, such as red blood cells transport in the human circulation and artificial capsules flow through microfluidic devices. In physiology, many blood diseases are usually accompanied by the curvature of blood vessels [1]. For example, atherosclerosis tends to occur in areas where these vessels bend and branch [2–5]. In microfluidics, curved tubes are often used for cell or capsule separation and aggregation. Several studies used unlabeled helical microfluidics to isolate circulating tumor cells from other normal cells [6–8]. Moreover, many experiments about the tube's curvature effect on the separation of particles of different sizes have been carried out. Kutaegowdanahalli *et al.* [9–11] first used spiral tubes for sorting microparticles with efficiency

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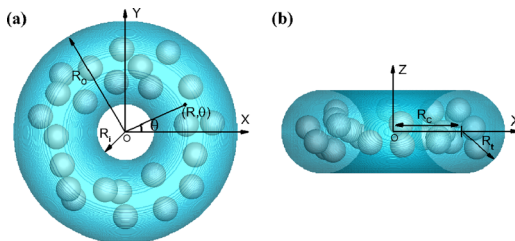


FIG. 1. Geometry for a torus vessel, (a) top view, (b) side view. The tube's radius is  $R_t$ . Here  $R_i$  and  $R_o$  are the radii of the inner and outer walls, respectively.  $Z = 0$  denotes the symmetrical plane. Curvature radius  $R_c$  is defined as the distance between the origin  $O$  and the vessel's centerline.

over 80%. Russom *et al.* [12] used a similar device with a low channel aspect ratio to separate microparticles and found that the focusing behavior was not affected by particle density. After these, there are many other similar separation experiments [13–15]. Besides the spiral tubes, the serpentine tubes (another type of curved tubes) are also used to separate microparticles by Di Carlo *et al.* [16]. It is necessary to further understand the flow properties in curved tubes.

In curved tubes, the flow undergoes outward centrifugal acceleration and forms secondary flow: a pair of counter-rotating vortices, i.e., Dean's vortices. Dean's vortices are first analyzed with the perturbation method in torus vessels by Dean [17]. The geometry of a torus vessel is shown in Fig 1. It is a spiral tube with zero spiral pitch. Because this kind of curved tube is simple and the curvature is uniform, numerous studies on the single-phase flow in the tube have been performed. Barua *et al.* [18–21] discussed the secondary flows of the circular cross section in torus vessels under different Dean numbers theoretically and numerically. Cheng *et al.* [22–24] obtained the velocity distribution in the torus vessels with triangular and rectangular cross sections through numerical calculation. Siggers and Waters [25] performed a comprehensive analysis and gave qualitative features of steady curved-tube flows.

In our study, torus vessels are adapted. For suspension flow in torus vessel, the capsules can reach an equilibrium state quickly. In reality, how to drive the flow inside the torus may be a technical issue. On the other hand, the flow inside an infinite spiral tubes with small spiral pitch  $p$  should be similar to that inside our torus vessels; this has been considered in previous work. [26]. In other words, if the flow in extending direction ( $z$  direction in Fig. 1) can be ignored, the flow in one coil of a spiral tube with periodic boundary on its inlet/outlet circular cross sections should be close to that inside our torus vessel.

For suspension flows in straight tubes, there are a large number of studies on the dynamics of either single-capsule suspensions [27–31] or multicapsule suspensions [32–36]. However, there are few studies, especially numerical studies, on the capsule suspension in curved tubes. Ye *et al.* [37,38] studied the movement and deformation of a single Red Blood Cell (RBC) in various curved tubes, e.g., spiral tube or U-shaped tube, and found that Dean's vortex can greatly affect the deformation of the capsules. Balogh and Bagchi [34,39] analyzed the cell-free-layer (CFL) profiles in microvascular networks within vivolike topologies. They showed that the vessel tortuosity causes the CFL to become more asymmetric along with the flow. Ebrahimi *et al.* [40,41] did comprehensive studies on the migration and deformation of a single capsule in torus vessels of circular and rectangular cross sections. However, in blood flows or industrial flows, e.g., the flow of coal water slurry, usually macroscopic viscosity of the suspension is more important and useful [33,42,43]. Here, we focus on the influence of the Dean's vortex on the apparent viscosity of the suspension. The ultimate goal is to understand the rheology of capsule suspensions and the distribution of capsules in curved tubes.

Inertia is an important factor that would affect the migration of rigid particles and deformable capsules. When inertia is present, particles can undergo cross-stream migration, and finally settle at an equilibrium radial position away from the tube center. For rigid particles, the equilibrium position is 0.6 times the tube radius from the tube center which was reported by Segré and Silberberg [44]

(Segré-Silberberg effect). Then Matas *et al.* [45–47] analyzed the shift of equilibrium position of rigid particles theoretically and experimentally. They found that depending on  $Re$  (the character length scale is the diameter of the tube, which is also applicable to the following  $Re$  in this paragraph), rigid particles are mainly distributed at two positions: Segré-Silberberg annulus and inner annulus. Different from rigid particles, the equilibrium position of soft capsules changes continuously with  $Re$ . The equilibrium position is off centered at low to medium  $Re$ , but shifts toward the centerline at high  $Re$  [32,33,35]. Inertia also plays an important role when curvatures are considered. Majji and Morris [48] found three regimes of particle distribution in Taylor-Couette flows at different  $Re$ . For curved tube flows, as the fluid inertia increases, the Dean’s vortex becomes stronger. It can be quantified by a dimensionless number, i.e., Dean number  $De = Re\sqrt{\frac{D_h}{2R}}$ , where  $D_h$  is the tube diameter,  $R$  is the radius of curvature [36]. When the size of curved tubes is fixed,  $De$  can be represented by  $Re$ . According to Ref. [40], when  $Re$  is small, the capsule stays on the symmetrical plane, while at high  $Re$ , it would move to the center of the vortex. This behavior can significantly affect the rheology of suspensions. In this paper, we mainly focus on the fluid inertia effect. Besides, the rheology properties and the distribution of multiple capsules under various  $\kappa$  and  $\phi$  in curved tubes are also explored.

To the best of our knowledge, there is so far no systematic and in-depth three-dimensional numerical study on the rheology and distribution of deformable capsules in the curved tubes. The change of effective viscosity with  $Re$  and  $\kappa$ , as well as the distribution of multiple capsules in curved tubes remains unclear. In this paper, not only the dilute regime with one single capsule but also the semidilute regime with multiple capsules are studied. The spatial distribution of capsules and the apparent viscosity of suspension in both curved and straight tubes are compared and analyzed. The role of Dean’s vortex is analyzed in detail.

The article is organized as follows. In Sec. II, the problem description, membrane model, and numerical method are introduced and validated. The results of single-capsule and multiple-capsule suspensions in torus vessels are presented in Sec. III. Conclusions are in Sec. IV.

## II. PROBLEM STATEMENT

The equation of the torus vessel is  $Z^2 = R_t^2 - (R_c - \sqrt{(X^2 + Y^2)})^2$ . Both the Cartesian coordinates  $(X, Y, Z)$  and the cylindrical coordinates  $(R, \theta, Z)$  are shown in Fig 1. The curvature radius  $R_c$  and tube radius  $R_t$  are defined in the caption of Fig 1. In our simulations, the fluid in the torus is driven by a body force or an azimuthal pressure gradient  $\frac{1}{R_c} \frac{dp}{d\theta} = -4\mu_0 U_0 / R_t^2$  [40], where  $\mu_0$  is the viscosity of the fluid and  $U_0$  is the maximum velocity in a straight tube with radius  $R_t$  if pressure gradient along the flow  $\frac{dp}{dx} = \frac{1}{R_c} \frac{dp}{d\theta}$  is applied.

### A. Methodology

In numerical simulations, each capsule is modeled as a liquid drop covered by a two-dimensional (zero-thickness) hyperelastic membrane. The membrane density is the same as the fluid. Also the mechanical properties of the fluid inside and outside of the capsule are identical. To simulate the migration and deformation of capsules, the capsule surface is discretized into thousands of triangular elements [49]. The force of the Lagrangian nodes of the triangular elements is obtained by solving the constitutive equation. Skalak’s (SK) law [50], which can model the shear deformation and area dilation of membrane is used

$$W^{\text{SK}} = \frac{1}{4}G_s(I_1^2 + 2I_1 - 2I_2) + \frac{1}{4}CG_sI_2^2, \quad (1)$$

where  $W$  is the strain energy density per unit membrane area.  $G_s$  is the surface shear modulus,  $C$  is the volume modulus ratio.  $I_1, I_2$  are the first and second strain invariants with  $I_1 = \lambda_1^2 + \lambda_2^2 - 2$  and  $I_2 = (\lambda_1\lambda_2)^2 - 1$ .  $\lambda_1$  and  $\lambda_2$  are the principal extension ratios in the plane of the membrane. For biological membranes,  $C$  must be large enough to ensure negligible area dilation. However for artificial capsules, taking a large value of  $C$  would cause numerical instability and lead to no

deformation. Therefore in the present study,  $C$  is set to be 10, which is identical to that in Ref. [40]. This value would not lead to numerical instability and can provide a large deformation.

The tensile forces acting on the Lagrangian nodes are obtained from the strain energy function  $W$  using the principal virtual work as

$$\mathbf{f}_t(\mathbf{s}) = -\frac{\partial W}{\partial \mathbf{v}}, \quad (2)$$

where  $\mathbf{v}$  is the displacement of the Lagrangian node  $\mathbf{s}$  on the capsule membrane.

The bending resistance of the membrane is modeled as a force density [51]:

$$\mathbf{f}_b = k_b[(2k_m + c_0)(2k_m^2 - 2k_g - c_0k_m) + 2\Delta_{\text{LB}}k_m]\mathbf{n}, \quad (3)$$

where  $k_m$ ,  $k_g$ ,  $c_0$  are the mean, Gaussian, and spontaneous curvatures, respectively.  $\Delta_{\text{LB}}$  is the Laplace-Beltrami operator and  $\mathbf{n}$  is the unit normal vector. Finally, the total force of the Lagrangian node on the membrane is the sum of the tensile force  $\mathbf{f}_t$  parallel to the membrane surface and the bending force  $\mathbf{f}_b$  perpendicular to the membrane surface,  $\mathbf{f}_m = \mathbf{f}_t + \mathbf{f}_b$ .

The fluid flow is simulated using the lattice Boltzmann method (LBM) [52]. The LBM originates from lattice gas automata. Macroscopically, it is able to recover the incompressible Navier-Stokes equations. The discretized lattice Boltzmann equation is

$$f_i(\mathbf{x} + c_i\Delta t, t + \Delta t) - f_i(\mathbf{x}, t) = \frac{1}{\tau}(f_i(\mathbf{x}, t) - f_i^{\text{eq}}(\mathbf{x}, t)) + \Delta t S_i, \quad (4)$$

where  $f_i(\mathbf{x}, t)$  is the distribution function at time  $t$  and position  $\mathbf{x}$ . Here  $D3Q19$  velocity model is adopted [53] and  $c_i$  is the  $i$ th discrete velocity, where  $i = 0 \dots, 18$ . Here  $\Delta t$  is time step and  $\tau$  is the relaxation time, which is related to the dynamic viscosity in the NS equation  $\mu = \rho c_s^2(\tau - 0.5)\Delta t$ . The equilibrium distribution function  $f_i^{\text{eq}}(\mathbf{x}, t)$  adopts the form

$$f_i^{\text{eq}}(\mathbf{x}, t) = \omega_i \rho \left[ 1 + \frac{\mathbf{e}_i \cdot \mathbf{u}}{c_s^2} + \frac{(\mathbf{e}_i \cdot \mathbf{u})^2}{2c_s^4} - \frac{(\mathbf{u})^2}{2c_s^2} \right], \quad (5)$$

where the weighting coefficients are  $\omega_0 = 1/3$ ,  $\omega_i = 1/18$  ( $i = 1 - 6$ ) and  $\omega_i = 1/36$  ( $i = 7 - 18$ ). The lattice sound speed  $c_s = \Delta x/(\sqrt{3}\Delta t)$ . The source term in the lattice Boltzmann equation is

$$S_i = \left(1 - \frac{1}{2\tau}\right) \omega_i \left[ \frac{\mathbf{e}_i - \mathbf{u}}{c_s^2} + \frac{\mathbf{e}_i \cdot \mathbf{u}}{c_s^4} \mathbf{e}_i \right] \mathbf{F}. \quad (6)$$

The macroscopic quantities of the fluid, such as density, velocity, and pressure can be calculated from the distribution function as

$$\rho = \sum_i f_i, \quad \mathbf{u} = \frac{1}{\rho} \sum_i f_i \mathbf{e}_i + \frac{1}{2\rho} \mathbf{F} \Delta t, \quad p = c_s^2 \rho. \quad (7)$$

The coupling between the solid and fluid is achieved by the immersed boundary method (IBM) [54] to ensure the nonslip boundary condition. In the implementation of the IBM, first the fluid velocity at the Lagrangian node  $\mathbf{s}$  on the capsule surface is interpolated from the velocities of surrounding fluid nodes, i.e.,

$$\mathbf{u}(\mathbf{s}) = \int_V \delta(\mathbf{x} - \mathbf{s}) \mathbf{u}(\mathbf{x}) dV, \quad (8)$$

where  $\mathbf{s}$  and  $\mathbf{x}$  are the locations of Lagrangian points and fluid nodes, respectively.  $V$  represents the fluid domain.  $\delta$  is the delta function

$$\delta(\mathbf{x} - \mathbf{s}) = \delta(x - x(\mathbf{s}))\delta(y - y(\mathbf{s}))\delta(z - z(\mathbf{s})), \quad (9)$$

where

$$\delta(r) = \begin{cases} \frac{1}{4} \left( 1 + \cos \left( \frac{\pi|r|}{2} \right) \right), & |r| \leq 2 \\ 0, & |r| > 2. \end{cases} \quad (10)$$

Then, according to Newton's second law, Lagrangian points may move  $\mathbf{u}(s)\Delta t$ . Due to the displacement or deformation, the force  $\mathbf{f}_m$  acting on the Lagrangian point can be calculated through constitutive law and bending equation. Finally,  $\mathbf{f}_m$  has to be spread to the surrounding fluid nodes through

$$\mathbf{F}(\mathbf{x}) = \int_V \delta(\mathbf{x} - \mathbf{s}) \mathbf{f}_m(\mathbf{s}) dV. \quad (11)$$

It is noted that this force should be added to the lattice Boltzmann equation as an external force.

The problem depends on several dimensionless parameters. The tube Reynolds number  $\text{Re} = (\rho U_0 R_t) / \mu_0$ . The capillary number  $Ca = (\mu_0 U_0) / G_s$  where  $G_s$  is the surface shear modulus.  $Ca$  measures the ratio between the viscous and elastic forces. The dimensionless bending stiffness  $E_b = k_b / (G_s a^2)$  where  $k_b$  is the bending modulus,  $a$  is the radius of capsule.  $E_b$  measures the relative importance of the membrane bending to shear elastic effects. Unless otherwise specified,  $E_b$  is kept at a constant 0.01, mainly to prevent the formation of membrane wrinkles [55]. The volume fraction of capsules  $\phi = \frac{\frac{4}{3}\pi a^3}{2\pi R_c \cdot \pi R_t^2}$ . The curvature of the torus vessel is defined as  $\kappa = R_t / R_c$ . The smaller the curvature is, the less bending the tube is. When the curvature approaches 0, the tube is straight. Unless otherwise specified, in the following descriptions, lengths, velocities, and time are normalized by  $a$ ,  $U_0$ ,  $a/U_0$ , respectively. For the polymeric capsules in linear flows, many studies focus on the deformation and inclination of capsules with a size of  $0.03 \sim 2$  mm. Usually, the viscosity of the fluid is  $0.1 \sim 1$  Pa s and channel width is  $5 \sim 80$  times the radius of capsules [56–59]. For the experimental feasibility of our typical cases, according to the above common choices, suppose  $a \sim 0.03$  mm,  $\mu_0 \sim 0.1$  Pa · s, through calculations of nondimensional parameters, the properties of capsules should be  $G_s = 1 \times 10^{-3} \sim 10$  N/m,  $E_b \sim 10^{-11}$  N m.

## B. Validation

We first simulate the motion and deformation of a large spherical capsule in a long straight tube with square cross section. In the simulations, tube length is 18, mesh resolution  $\Delta x = 0.04$ . The radius ratio between the capsule and the tube is  $a/R_t = 0.85$ . The three-dimensional capsule is discretized into 8192 triangular elements and 4098 Lagrangian nodes. The equilibrium deformed profiles are plotted in Fig 2. It can be seen that our equilibrium profiles agree well with that in the literature [60]. Second, the flow without capsules in a torus is simulated. For the fully developed flow without inertia in the torus vessel, the analytical solution of azimuthal velocity  $U$  is given by the well-known Dean's solution [17]:

$$U/U_0 = (1 - r^2) - \frac{3\kappa}{4} r(1 - r^2) \cos \theta + \frac{\kappa^2}{32} (1 - r^2)(-3 + 11r^2 + 10r^2 \cos 2\theta) + O(\kappa^3), \quad (12)$$

where  $r = R/R_t$  represents the radial position,  $(R, \theta)$  are the polar coordinates defined in Fig 1. The first term on the right-hand side of the equation represents the velocity profile of Poiseuille flow in a straight tube, while the remaining terms represent the curvature effect. Figure 3(a) shows the contours of  $U$ . The contours are symmetric about the dashed line (the symmetrical plane  $Z = 0$ ) but they are not left-right symmetric. The location of the maximum velocity  $U_m$ , i.e.,  $R'_{U_m}$  occurs in the symmetrical line but is closer to the inner wall, see Fig. 3(b). The location is a function of  $\kappa$  [40]:

$$R'_{U_m} = -\frac{3\kappa}{8} + O(\kappa^3), \quad (13)$$

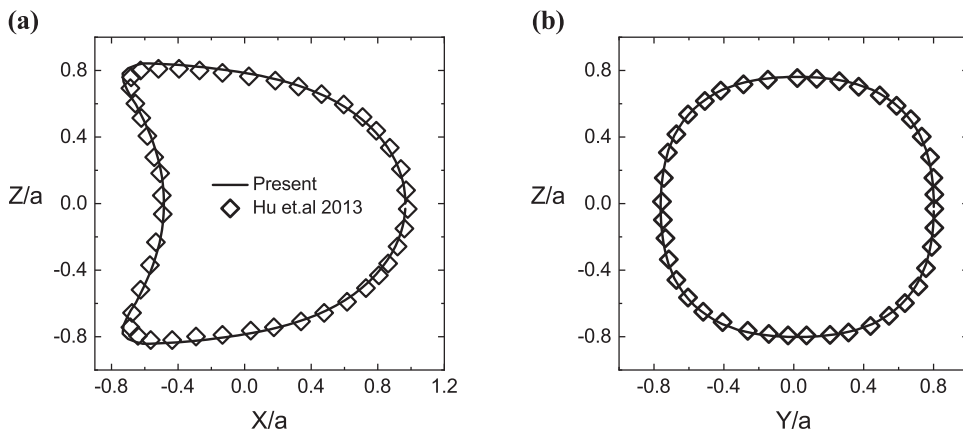


FIG. 2. Equilibrium profiles of an initially spherical capsule with SK membrane ( $Re = 0.125$ ,  $E_b = 0$ ,  $Ca = 0.1$ ,  $C = 1$ ) in a square section channel. (a) Profile in the  $XZ$  plane, (b) Cross profile in the  $YZ$  plane. The solid lines represent our results. The diamond symbols denote the results obtained by Hu *et al.* [60] using boundary element method.

where  $R'$  is the normalized radial location along the symmetrical plane, which is defined as  $R' = \frac{R-R_c}{R_i}$ . It is noted that  $R' = -1, 0, 1$  represent the innermost edge of the vessel on the plane of  $Z = 0$ , vessel centerline, and the outermost edge on the plane, respectively. The negative sign in the above equation indicates that  $R'_{U_m}$  is closer to the inner wall. As the curvature increases,  $R'_{U_m}$  further moves towards the inner wall. Figure 3(c) shows  $R'_{U_m}$  as a function of  $\kappa$ . In the cases,  $R_t = 2a$  and  $Re = 0.05$  are fixed but the curvature radius  $R_c$  varies. Here  $R'_{U_m}$ s that we obtained in our simulations are consistent with the analytic solution.

Then we conduct a grid independence study. In this case, a capsule moves in a torus flow. First, we fix the Lagrangian grids to 4098, which is fine enough, and change the Eulerian mesh

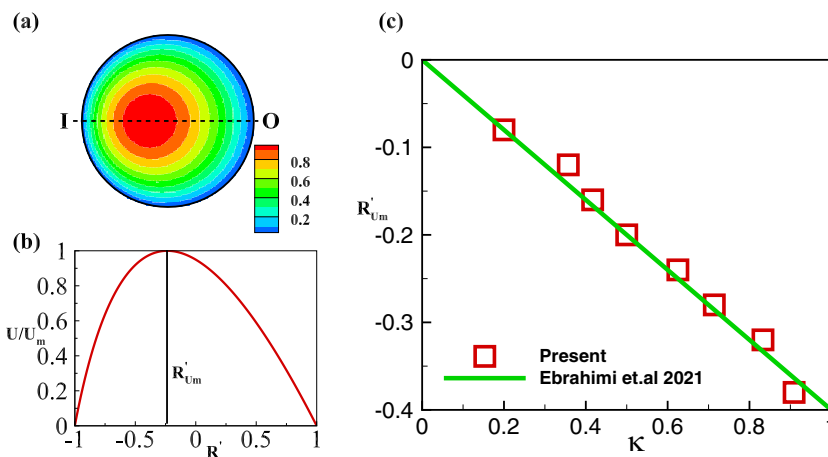


FIG. 3. (a) Azimuthal velocity (scaled by  $U_m$ ) contours in the case of  $Re = 0.05$ ,  $R_c = 3$ ,  $R_t = 2$ . The dashed line represents the symmetrical line  $Z = 0$ . I and O represent the innermost wall ( $R_i$ ) and outermost wall ( $R_o$ ) of the vessel, respectively. (b) Distribution of azimuthal velocity along the symmetrical line. The solid line represents the location of  $U_m$ . (c) Location of the maximum azimuthal velocity ( $R'_{U_{max}}$ ) on the symmetrical line in cases of  $\kappa : 0.2, 0.357, 0.417, 0.5, 0.625, 0.714, 0.833, 0.9091$  at  $Re = 0.05$ .

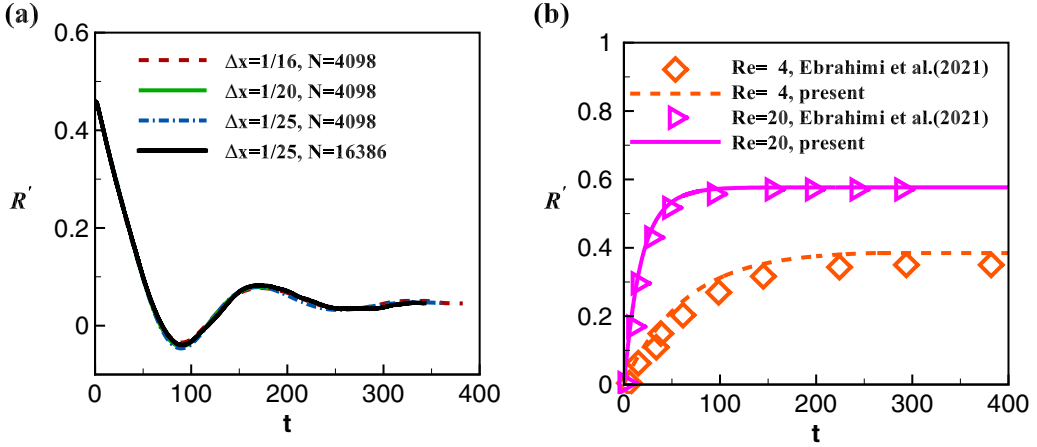


FIG. 4. (a) Radial location of the capsule as a function of time (grid independence study). The key parameters are  $Re = 10$ ,  $Ca = 1.0$ ,  $R_c = 5$ , and  $R_t = 3$ . (b) Results of the cases of  $Re = 4, 20$ . The other parameters are  $Ca = 1.0$ ,  $R_c = 5$ , and  $R_t = 4$ . Lines represent our results and symbols represent the results of Ref. [40].

size  $\Delta x = \frac{1}{16}, \frac{1}{20}, \frac{1}{25}$ , and track the radial location of the mass center of the capsule. Figure 4(a) shows that the results with three different Eulerian meshes collapse into a single curve. Then we used more Lagrangian grids to check the grid independence. It is seen from Fig. 4(a) that even more Lagrangian grids ( $N = 16386$ ) are used, the result almost collapses into the identical curve. Therefore Lagrangian grids  $N = 4098$  is enough. In our following simulations, the resolution ( $\Delta x = \frac{1}{25}, N = 4098$ ) is adopted.

In addition, using the resolution, we simulated another two cases of  $Re = 4$  and 20. Here a direct comparison [Fig. 4(b)] shows that our results of  $Re = 4, 20$  are consistent with those in Ref. [40].

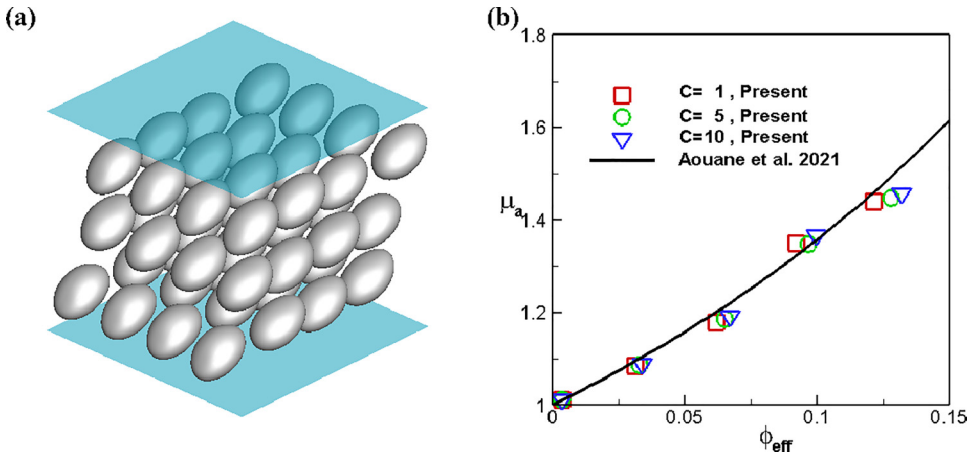


FIG. 5. (a) Typical configuration of capsules ( $\phi = 20.11\%$ ,  $Ca = 0.2$ ) in the computational domain where three lengths are  $10a \times 10a \times 10a$ . (b)  $\mu_a$  as a function of  $\phi_{eff}$  defined in Eq. (3.9) in Aouane *et al.* [61] at  $Re = 0.01$ . The mechanical properties of the capsules are  $C = 1, 5, 10$ ,  $Ca = 0.2$ ,  $\phi = 0.42\%, 3.77\%, 7.54\%, 11.31\%, 15.08\%, 20.11\%$ , where the cases of  $\phi = 0.42\%$  contain one capsule. The symbols are our results and the black line is the scaling law [Eq. (3.10)] of Aouane *et al.* [61].

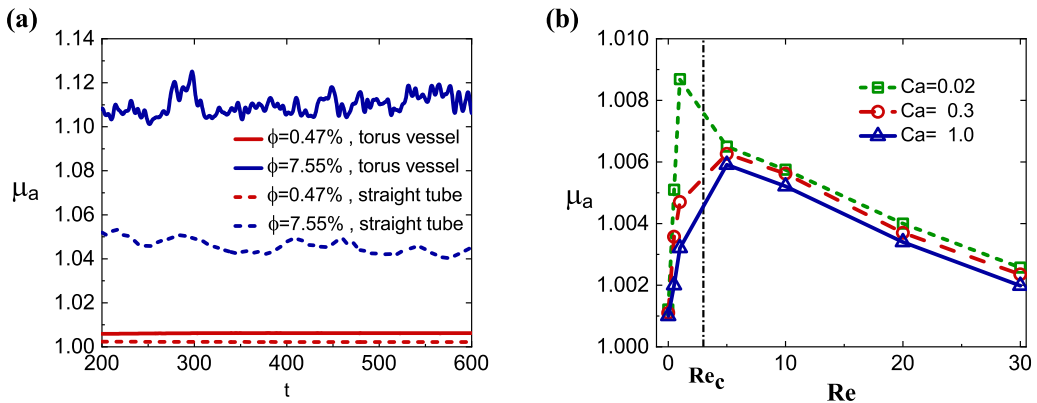


FIG. 6. (a) Evolution of instantaneous  $\mu_a$  for the cases with different  $\phi$  in torus vessel and straight tube ( $Re = 5$ ). (b)  $\mu_a$  as a function of  $Re$  for the cases of  $Ca = 0.02, 0.3, 1.0$  at  $\phi = 0.47\%$ .  $Re_c \approx 3.0$ .

At last, we validate the multiple capsule suspensions in a straight channel. Figure 5(a) shows the snapshot of a typical configuration of capsules in shear flow. The apparent viscosity of capsule suspension is compared with that of Aouane *et al.* [61]. From Fig. 5(b), it can be seen that our results are consistent with their scaling law. The code is accurate enough to simulate the flow of suspension in curved tubes.

### III. RESULTS AND DISCUSSION

In the following subsections, we simulate the migration of single and multiple capsules in torus vessels and straight tubes. Flow properties of capsule suspensions as functions of  $Re$ ,  $\kappa$ , and  $\phi$  are presented. Corresponding capsule distributions are also analyzed.

#### A. Single capsule flow in torus vessels

In this subsection,  $R_c = 5$  and  $R_t = 3$  are fixed. Since  $\kappa$  is a constant, the intensity of Dean's vortex only depends on fluid inertia  $Re$ . We mainly discuss the rheology of the most dilute suspension ( $\phi = 0.47\%$ ), which contains only one single capsule. For the pure fluid flow in the torus vessel without capsules, the volume flux  $Q_0$  under the driven of pressure gradient  $g$  is

$$Q_0 = \int_0^{R_t} \int_0^{2\pi} U(R, \theta) R dR d\theta = \left[ \frac{1}{2} + \frac{1}{96} k^2 + O(k^3) \right] \frac{g\pi R_t^4}{4\mu_0} = \frac{\lambda}{\mu_0}, \quad (14)$$

where  $U(R, \theta)$  is the theoretical azimuthal velocity in the limit of small  $Re$  [17].  $\lambda$  is a constant, which is only a function of  $g$ . If the capsule suspension flows in the same torus with pressure gradient  $g$ , we have volume flux of the capsule suspension  $Q = \frac{\lambda}{\mu}$ , where  $\mu$  is the effective viscosity of the suspension. Then the apparent viscosity is  $\mu_a = \frac{\mu}{\mu_0} = \frac{Q_0}{Q}$ . Similarly, the apparent viscosity depending on  $Re$  is introduced as [33,62]

$$\mu_a = \frac{\mu}{\mu_0} = \frac{Q_l}{Q}, \quad (15)$$

where  $Q_l$  and  $Q$  are the volume flux of the suspension with and without capsules at the same Reynolds number.

Figure 6(a) shows the instantaneous  $\mu_a$  in torus vessels and straight tubes under different volume fractions. For the most dilute regime (the red lines),  $\mu_a$  is almost a constant at the equilibrium state. For the semi dilute regime, the variations of the curves are quite small, which indicates that the equilibrium states are achieved. Here  $\mu_a$  as a function of  $Re$  for different  $Ca$  is shown in Fig. 6(b).



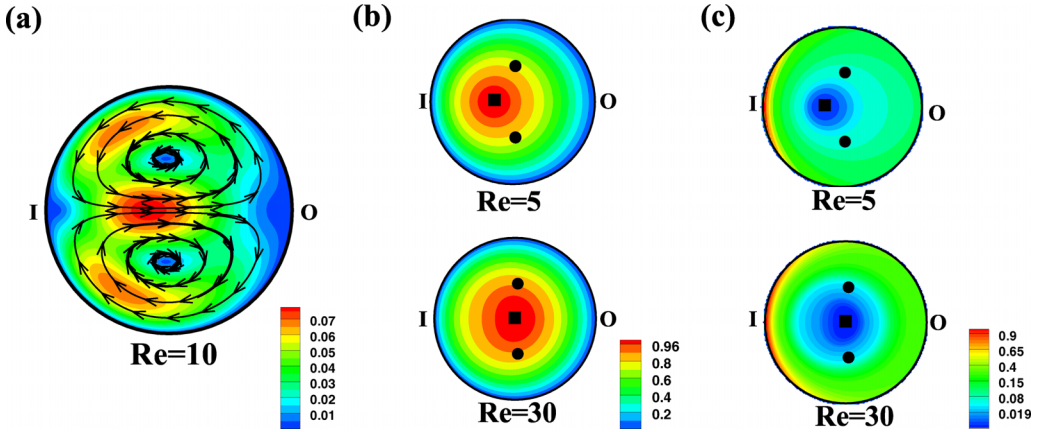


FIG. 7. (a) Contours of secondary velocity  $U_s = \sqrt{(u_R^2 + u_Z^2)}$  inside a cross section, where  $u_R, u_Z$  are  $R$  and  $Z$ -component velocities. Streamlines are also shown. (b) Contours of the azimuthal velocity  $U$ . It is perpendicular to the paper inward. (c) Contours of the dissipation  $D$  which is proportional to the product of the local shear rate  $\dot{\gamma}$  and fluid shear stress  $\sigma_f$ ,  $D \propto \dot{\gamma} \cdot \sigma_f$  [33]. Here  $U_s, U$  are scaled by  $U_m$ ,  $D$  is scaled by  $D_m$  (maximum of  $D$ ). Squares represent the locations of  $U_m$  in (b) and the locations of the lowest  $D$  in (c). The solid dots represent the central locations of the Dean's vortices. I and O represent the inner and outer walls of the vessel.

We can see that  $\mu_a$  as a function of  $Re$  can be divided into two regimes by a critical Reynolds number  $Re_c$  [it is defined as the location of the maximum of the curves  $\mu_a = f(Re)$ ], i.e.,  $Re_c \approx 3$ . For  $Re \leq Re_c$  (Regime I),  $\mu_a$  increases with  $Re$  and is strongly affected by  $Ca$ , which is similar to that in a straight tube. However, for  $Re > Re_c$  (Regime II),  $\mu_a$  decreases with  $Re$  and is not sensitive to  $Ca$ . This trend differs from that in a straight tube, where  $\mu_a$  increases with  $Re$  within the parameters we studied [32,33].

We would like to explain the significant difference at Regime II by showing the role of the secondary flow (Dean's vortex). Regime I would also be discussed. The Dean's vortex is shown in Fig. 7(a). There are two counter-rotating vortices on the cross section. The upper and lower vortices rotate counterclockwise and clockwise, respectively. A higher  $Re$  leads to a stronger Dean's vortex [19]. Due to Dean's vortices, the location of the maximum azimuthal velocity  $U_m$  is shifted. From Figs. 7(b) and 7(c), we can see that in tube flow, the region with larger azimuthal velocity has lower dissipation, which means that if capsules are in the region close to  $U_m$ ,  $\mu_a$  would be small. The contours of the azimuthal velocity  $U$  are shown in Fig. 7(b), the locations of  $U_m$  are labeled by the black squares. It is seen that when  $Re$  changes from 5 to 30, the location becomes closer to the centerline of the vessel. On the other hand, the locations of the centers of the Dean's vortex are almost unaffected by  $Re$  (see the solid dots). In other words, when  $Re$  is larger, the region around the center of the Dean's vortex would be closer to the location of  $U_m$ , thus the dissipation of the region becomes lower.

Meanwhile, we would like to discuss the equilibrium lateral locations of the capsules. In typical cases, such as the cases of  $Re = 5$  and 30, capsules all laterally migrate towards the center of Dean's vortex, and at equilibrium states, they stay close to the center [See upper semicircle in Fig. 8(a)]. Hence, as  $Re$  increases, at the equilibrium state, the capsule is closer to the low-dissipation region, which is favorable to decrease  $\mu_a$ . To further illustrate this point, we plot the normalized azimuthal velocities at the equilibrium location of the capsules ( $U/U_m$ ) [Fig. 8(b)]. For the cases of  $Re > Re_c$ , indeed,  $U/U_m$  increases with  $Re$ . This is expected because  $U_m$  becomes closer to the equilibrium position of the capsule. In addition, the effect of  $Ca$  on  $\mu_a$  is minor since all capsules with different rigidity are trapped by the center of the Dean's vortex.

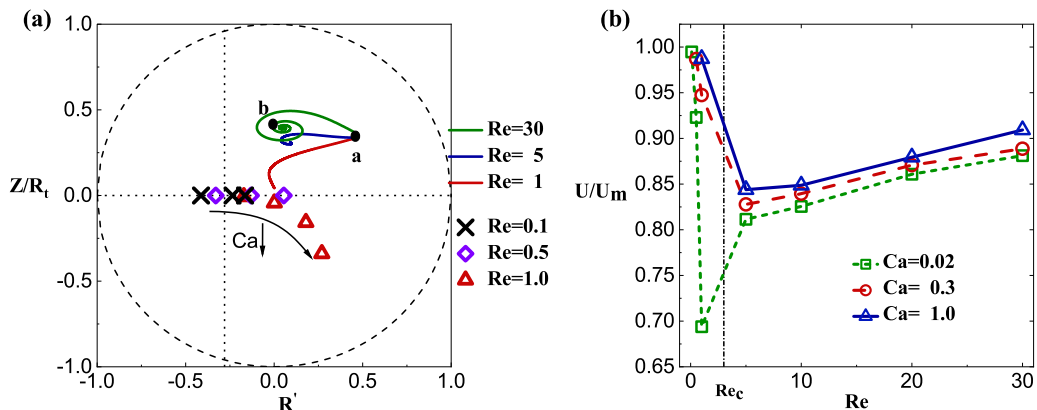


FIG. 8. (a) Trajectories and equilibrium locations of different cases in the cross-section. Upper semicircle: the trajectories of the capsule in the cases of  $Re \geq 1$  ( $Re = 1, 5$  and  $30$ ,  $Ca = 0.3$ ). The capsules are all released from the initial position a. Point b denotes the center of Dean's vortex at  $Re = 30$ . Lower semicircle: equilibrium locations of the capsule in the cases with  $Re \leq 1$ . At the same  $Re$ , from left to right,  $Ca = 1.0, 0.3, 0.02$ , respectively (the case  $Re = 1$  and  $Ca = 0.1$  is also presented). The intersection of the two dotted lines denotes the location of  $U_m$  when  $Re \leq 1$ . Ebrahimi *et al.* [40] also got similar results. (b) The normalized azimuthal velocity  $U/U_m$  at the equilibrium position of capsules as a function of  $Re$ . In this figure, all cases are calculated with one capsule.

For  $Re \leq Re_c$ , the intensity of Dean's vortex is weak.  $R'_{U_m}$  is closer to the inner wall and is not affected by Dean's vortex [see the intersection of the two dotted lines in Fig. 8(a)]. At a low  $Re$ , e.g.,  $Re = 0.1$ , the capsules laterally stay almost at the location of  $U_m$  and  $U/U_m \approx 1$  [see Fig. 8(b)], therefore  $\mu_a$  is small. As  $Re$  increases, they gradually move away from the location of  $U_m$  and migrate to a higher-dissipation region [see the equilibrium locations in the lower semicircle of Fig. 8(a)], thus  $\mu_a$  increases. In addition, at  $Re \approx Re_c$ , when the capsule is rigid (smaller  $Ca$ ), the capsule may further move away from the location of  $U_m$  [e.g., see the red triangles in Fig. 8(a)]. The behavior would greatly influence  $\mu_a$ . Then, we performed a quantitative analysis of the relationship between the equilibrium position and  $Ca$  at  $Re \approx Re_c$ , see Fig. 9. Through data fitting, the equilibrium position of capsules as a function of  $Ca$  is described quantitatively.

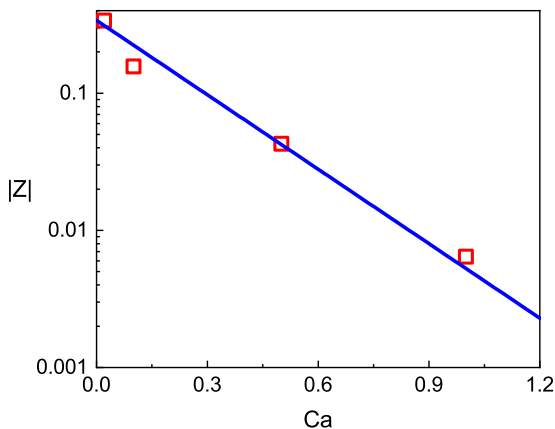


FIG. 9. Equilibrium position in  $z$  direction ( $|Z|$ ) as a function of  $Ca$  at  $\kappa = 0.6$ ,  $Re_c = 3.0$ . Red squares are our data, blue line is the exponential fitting formula  $|Z| = 0.35e^{-Ca/0.2}$ . The  $y$  axis is in log scale.

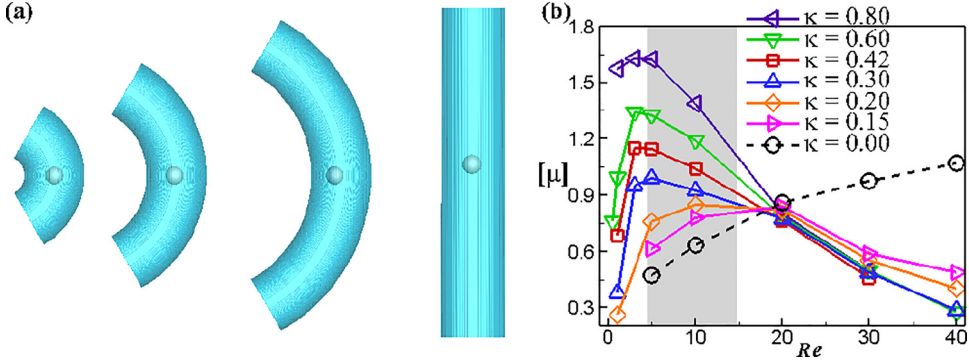


FIG. 10. (a) Configurations of the curved tubes with curvature  $\kappa = 0.6, 0.3, 0.2$  and straight tube (from left to right). (b) Intrinsic viscosity as a function of  $Re$  in torus vessels with various curvatures and the straight tubes. Each case contains one capsule.

When  $Ca$  is small, the equilibrium position is near the Dean's vortices. With the increase of  $Ca$ , the capsule would approach the position of maximum azimuthal velocity in an exponential trend. It is noticed that when  $\kappa \rightarrow 0$  (straight tube),  $Re_c \rightarrow \infty$ , there is no Dean's vortices and subsequently no  $|Z| - Ca$  curve. However, the exponential relationship between equilibrium radius and  $Ca$  is still valid [63].

### B. Curvature $\kappa$ effect

We now turn to the effect of curvature  $\kappa = R_t/R_c$  on the viscosity of capsule suspension with limited inertia. We mainly focus on the cases of  $Re > Re_c$ , because in such a regime,  $\mu_a$  decreases with  $Re$ , which differs from common perception. The above subsection shows that in the regime,  $\mu_a$  does not depend too much on the  $Ca$  at  $\kappa = 0.6$ . This is also the case for all the other configurations but is not shown here, so we keep  $Ca = 0.3$ . The fluid properties ( $\rho$  and  $\mu_0$ ) and the radius  $a$  of capsules are also fixed. Different curvature cases with  $\kappa$  ranging from 0.15 to 0.8 are achieved by changing the curvature radius and tube radius. Segments of the vessels are shown in Fig. 10(a). For comparison, the cases in a straight tube ( $\kappa = 0$ ) are also simulated. To eliminate the effect of  $\phi$ , we define the intrinsic viscosity  $[\mu]$  as

$$[\mu] = \frac{\mu - \mu_0}{\mu_0 \phi} = \frac{\mu_a - 1}{\phi}. \quad (16)$$

The intrinsic viscosity  $[\mu]$  as a function of  $Re$  for different  $\kappa$  is shown in Fig. 10(b). For the cases  $\kappa = 0.42, 0.3, 0.2$ , overall  $[\mu]$  decreases with  $Re$ . The variation trend looks similar to that of cases  $\kappa = 0.6$ , which we discussed above. On the other hand, we also find that the critical Reynolds number  $Re_c$  seems associated with  $\kappa$ . For example,  $Re_c \approx 3$  for cases of  $\kappa = 0.6$ , while  $Re_c \approx 10$  for  $\kappa = 0.2$ . Then  $Re_c$  as a function of  $\kappa$  is plotted in Fig 11(a). Through fitting the data points, we found that  $Re_c$  decreases exponentially with  $\kappa$ . When  $\kappa$  approaches 0 (the cases of straight tubes),  $Re_c$  approaches infinity. The possible reason is that in a straight tube, there is laminar flow and the viscosity always increases with  $Re$  and the transition occurs at  $Re \rightarrow \infty$ . When  $\kappa$  is large, the Dean's vortices become dominant and  $\mu_a$  begins to decrease with inertia at a small  $Re$ . Therefore  $Re_c$  would decrease with  $\kappa$ .

In addition, we can see that the variation trends of torus vessels are opposite to that in a straight tube ( $\kappa = 0$ ). The increasing trend of  $[\mu]$  in straight tubes can be explained in the following way. As  $Re$  increases, the capsules may migrate away from the centerline of the tube (low dissipation region) to the wall (high dissipation region), which enhances  $[\mu]$  [33]. The mechanism is similar to that in the torus vessel at small  $Re$ , which is also mentioned in the next subsection.

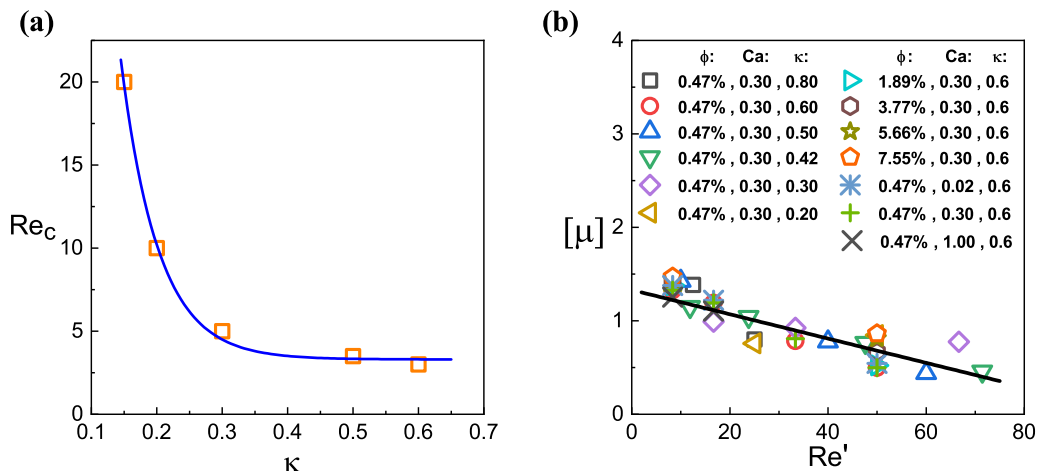


FIG. 11. (a) Transition Reynolds number  $Re_c$  as a function of  $\kappa$ . All cases contains one capsule with  $Ca = 0.3$ . The orange squares are our results, blue line is the fitting line  $Re_c = 3.30 + 227.45e^{-17.48\kappa}$ . (b) Intrinsic viscosity  $[\mu]$  as a function of  $Re'$  at various  $Ca$  and  $\phi$ . The line represents a data fitting curve  $[\mu] = -0.013Re' + 1.3$ .

Now we focus on the effect of  $\kappa$  on the cases of torus vessels when  $Re$  is at the shaded region. It is seen from Fig. 10(b) that  $[\mu]$  increases continuously with  $\kappa$ , e.g.,  $Re = 5$ , the suspension in the vessel with a larger  $\kappa$  appears more viscous. It can be understood as follows. As  $\kappa$  increases, the location of maximum azimuthal velocity  $R'_{U_m}$  becomes closer to the inner wall [see Fig. 3(c) and Eq. (13)], but due to the inertia of the capsules, the lateral locations of the capsules change little. In this way, the capsule becomes closer to the high-dissipation region, and  $[\mu]$  increases with  $\kappa$ . Here a scaling law for  $[\mu]$  in the shaded region of Fig. 10(b) is proposed. If Reynolds number in the region is redefined as  $Re' = Re/\kappa$ , then all the data in the shaded region can almost collapse into a single line  $[\mu] = -0.013Re' + 1.3$  [see Fig. 11(b)]. It is noted that the length that matters for the apparent viscosity is the radius of curvature of the torus  $R_c$  instead of  $R_t$ .

On the other hand, from Fig. 10(b) we also see that at large  $Re$ , when  $\kappa > 0.2$ ,  $[\mu]$  is a decreasing function of  $Re$ , which is not sensitive to  $\kappa$ . The possible reason is that when  $Re$  is large, the strength of Dean's vortices is so strong that the migration of capsules is dominated by them. When  $\kappa < 0.2$ , as the  $\kappa$  decreases, the strength of Dean's vortices become weak, so the curves decrease slower and slower, see the region outside the shaded one in Fig. 10(b). Here, the cases with very small  $\kappa$  were not performed for the following reasons. First, the cases of smaller  $\kappa$  (large radius of torus), require a large amount of computing resources due to the large computational domain size. Second, our focus is the flow inside significantly curved tubes, where Dean's vortices play an important role. The flow with a small  $\kappa$  (Dean's vortices are very weak) should be close to that of the straight tube ( $\kappa = 0$ , no Dean's vortices).

### C. Flow of semidilute suspensions in torus vessels

In the following, the semidilute regime with multiple capsules are investigated. In this subsection, the size of the torus is fixed,  $R_c = 5$ ,  $R_t = 3$ . For the semidilute regime, in addition to the above mechanism, the interaction between capsules may further affect the spatial distribution of capsules as well as the apparent viscosity [64]. The evolution of the instantaneous apparent viscosity  $\mu_a$  for the case of  $\phi = 7.55\%$  is shown in Fig. 6(a).  $\mu_a$  oscillates around an average value  $\bar{\mu}_a$  with a small amplitude  $\epsilon$ . When  $\epsilon < 1\% \bar{\mu}_a$ , we suppose the flow reaches an equilibrium state.

Typical snapshots of capsule distributions inside the vessel at equilibrium state for small [left column, (a) and (c)] and large  $Re$  [right column, (b) and (d)] are shown in Fig. 12. At small  $Re$ ,

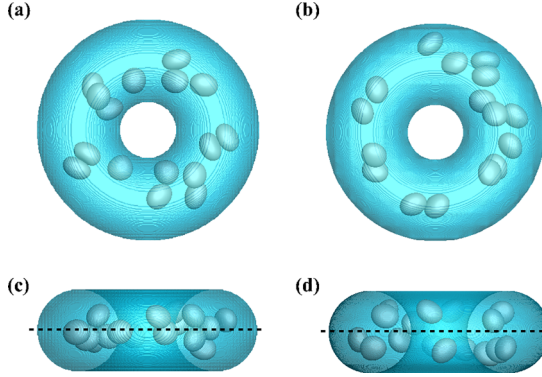


FIG. 12. Typical snapshots of configurations of capsules at equilibrium states ( $\phi = 7.55\%$ ,  $Ca = 0.3$ ). Top view (a), (b) and front view (c), (d) for the cases of  $Re = 1.0$ (a, c) and  $Re = 30$ (b, d). Dashed lines represent the symmetrical plane.

e.g.,  $Re = 1$ , because the Dean's vortices are weak, overall, capsules are close to the inner wall [see Fig. 12(a)] and concentrated close to the symmetric plane [see Fig. 12(c)]. While at large Reynolds number ( $Re = 30$ ), the Dean's vortex is strong and the capsules tend to migrate laterally to the vortices' centers. The capsules approximately form a two-file structure. The two files seem close to the centers of the upper and lower semicircles [see Figs. 12(b) and 12(d)].

To quantify the distribution of capsules in the torus, we give the temporal and spatial-averaged probability density of capsules, which is defined as

$$P(R, Z) = \frac{1}{2\pi R} \left\langle \int_0^{2\pi} I(R, \theta, Z, t) d\theta \right\rangle, \quad (17)$$

where  $I(R, \theta, Z, t)$  is a scalar function and if the lattice node  $(R, \theta, Z)$  is occupied by the capsules,  $I = 1$ , otherwise  $I = 0$ . The symbol  $\langle \dots \rangle$  denotes an averaging over time after reaching an equilibrium state. The time that used to compute the temporal average of the pdf is 400 (dimensionless time) which is long enough. The contours of  $P(R, Z)$  at different representative  $Re$  are shown in Fig. 13. From Fig. 13(a), we can see that at  $Re = 1$ , capsules concentrate on the symmetry plane and are close to the inner wall. At  $Re = 30$ , capsules stay close to the centers of vortices [see Fig. 13(b)], which is similar to the behavior in the single-capsule case. The distribution of capsules is consistent with what we have seen in Fig. 12. Moreover, quantitatively, the temporal and spatial-averaged centroid location of all capsules can be calculated through  $(\bar{R}, \bar{Z}) = (\iint P(R, Z)R dRdZ, \iint P(R, Z)Z dRdZ)$ . The temporal and spatial-averaged centroid is labeled as a cross symbol in Fig. 13(a). We can see that the centroid is close to the equilibrium location in the corresponding case of single capsule (solid dots). In addition, a similar phenomenon is also observed in the cases of other volume fractions (not shown). Therefore, it is reasonable to conjecture that the variation of  $\mu_a$  for semidilute regime is similar to that for the most dilute regime. Figure 14(a) confirms this point. From Fig. 14(a), we can see that at a specific  $\phi$ ,  $\mu_a$  as a function of  $Re$  is similar to that of the single-capsule cases in the torus vessels (increasing at small  $Re$ , decreasing at large  $Re$ ).

As a comparison,  $P(R, Z)$  in the straight tube is also investigated. The parameters  $Ca, R_t$  are the same as those in Sec. III B. Because there is no secondary flow in the straight tube, the distribution of capsules is different from that in the torus vessel. It can be seen that at  $Re = 1$ , the capsules are concentrated in the center of the tube, see Fig. 13(c). Then at  $Re = 30$ , the capsules are mainly located inside an annulus between the center and wall [Fig. 13(d)], i.e., inertial focusing phenomenon [65]. The annulus region has a larger shear rate with higher dissipation, which results

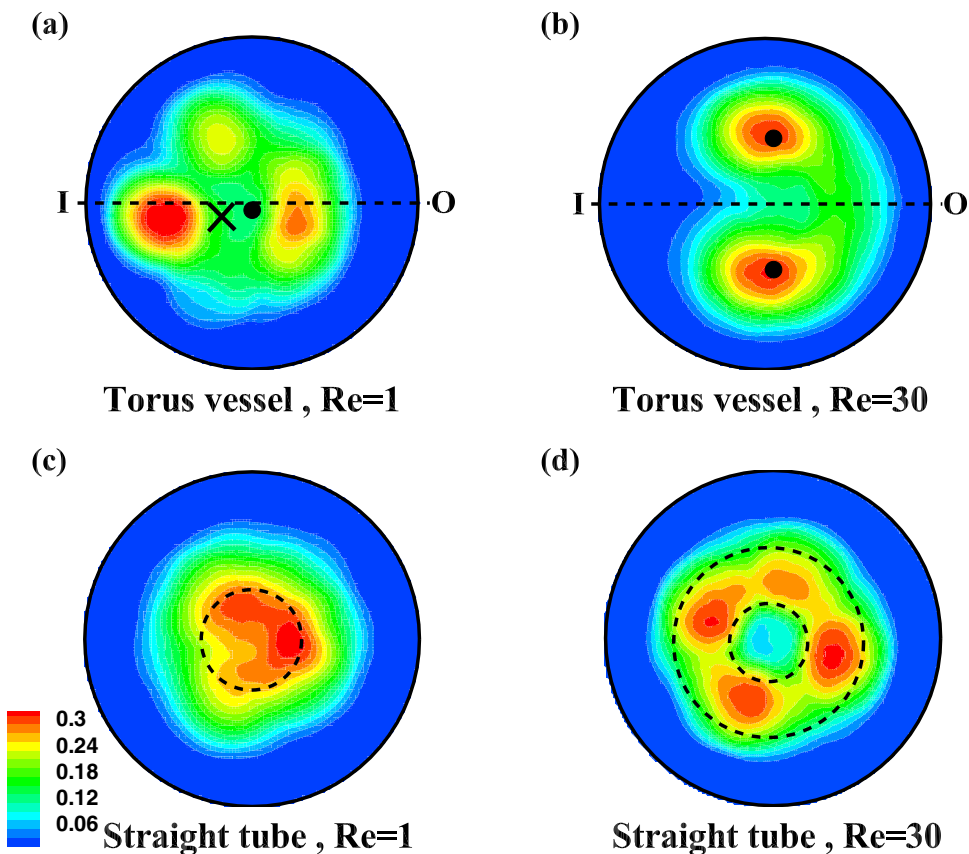


FIG. 13. The temporal and spatial-averaged probability density of capsules  $P$  in a cross-section. In all cases  $\phi = 11.325\%$ . The dashed line in (a) and (b) is the symmetrical line. The dashed circle in (c) and annulus in (d) denote the regions where the capsules are concentrated. The solid dot and cross represent the equilibrium position in the case of a single capsule and the average position of multiple capsules, respectively.

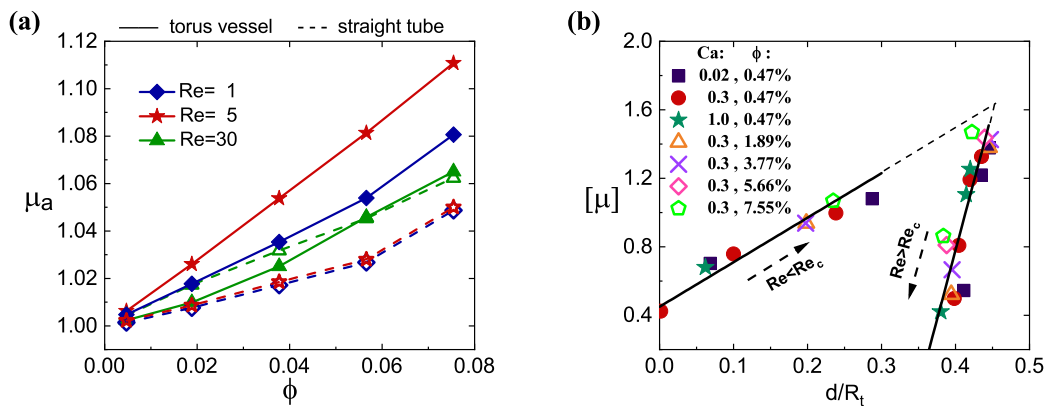


FIG. 14. (a)  $\mu_a$  as a function of  $\phi$  (0.47%, 1.89%, 3.77%, 5.66%, 7.55%) in torus vessel (solid lines) and straight tube (dotted lines) at  $Re = 1, 5, 30$  and  $Ca = 0.3$ . (b) Intrinsic viscosity  $[\mu]$  as a function of  $d$ . The directions of the arrows represent an increase of  $Re$ . The dashed line represents a transitional regime  $Re \approx Re_c$ .

in a larger  $\mu_a$  [see the dashed lines of  $Re = 1, 30$  in Fig. 14(a)]. Therefore in the straight tube,  $\mu_a$  generally increases with  $Re$ .

In both dilute and semi dilute results, effective viscosity seems directly relevant to the average location of capsules. In the following, we would like to quantify the relationship. The temporal and spatial-averaged distance  $d$  between the location of capsules and  $U_m$ , is defined as  $d = \langle \frac{1}{N} \sum_{i=1}^N R_i - R_{U_m} \rangle$ , where  $R_i$  and  $R_{U_m}$  are the positions of the  $i$ th capsule and  $U_m$  on the cross section, separately. We found that the intrinsic viscosity  $[\mu]$  is a piece linear function of  $d$  [see Fig. 14(b)]. We can see that  $[\mu]$  increases with  $d$  slowly when  $Re < Re_c$ , but decreases dramatically with  $d$  when  $Re > Re_c$ . Because when  $Re < Re_c$ , roughly capsules stay close to the location of  $U_m$  ( $d < 0.25$ ). In the region, the variation of the gradient of  $U$  (or viscous dissipation) is small. In other words, although  $d$  changes a lot,  $[\mu]$  increases gently with  $d$ . When  $Re > Re_c$ , capsules stay significantly far away from the location of  $U_m$  ( $d \approx 0.45$ ), where the variation of the gradient of  $U$  is high. Hence,  $[\mu]$  changes dramatically with  $d$ .

#### IV. CONCLUSION

In the present study, the rheology of capsule suspensions in torus vessels is studied by using an immersed-boundary lattice Boltzmann method. We first studied the apparent viscosity  $\mu_a$  as a function of  $Re$  and  $Ca$ . When the inertia is small ( $Re < Re_c$ ), the capsules tend to stay close to the  $Z = 0$  plane.  $\mu_a$  would increase with  $Re$ , because as  $Re$  increases, the capsule moves away from the location of  $U_m$  and migrates to the high-dissipation region. In addition,  $\mu_a$  decreases with  $Ca$  because the softer capsules (larger  $Ca$ ) are closer to the location of  $U_m$  where the viscous dissipation is low. When the inertia is large ( $Re > Re_c$ ),  $\mu_a$  decreases with  $Re$ , which is different from the trend in the straight tube. The Dean's vortices play an important role in the variation trend of  $\mu_a$ . On one hand, the stronger Dean's vortices at higher  $Re$  trap the capsules into their centers. On the other hand, they make the location of  $U_m$  closer to their centers. In this way, at higher  $Re$ , the capsules become closer to the low-dissipation region, which results in a smaller  $\mu_a$ . At this regime,  $\mu_a$  weakly depends on  $Ca$ .

Then the effect of curvature  $\kappa$  on the effective viscosity is investigated. When  $\kappa$  is large enough, e.g.,  $\kappa > 0.2$ , the variation trends of torus vessels are opposite to that in a straight tube ( $\kappa = 0$ ). When  $Re$  is small, e.g.,  $Re = 5$ , the suspension in the vessel with a larger  $\kappa$  appears more viscous because the inertia of the capsules makes them stay in the higher-dissipation region as  $\kappa$  increases. A scaling law for the effective viscosity at the small- $Re$  regime is proposed. All data of different  $\kappa$ ,  $Ca$ , and  $\phi$  in our study support the scaling law. On the other hand, at higher  $Re$ , Dean's vortices are dominant, which promotes the capsules' circumferential transportation. Therefore the effective viscosity is a decreasing function of  $Re$ , which is not sensitive to  $\kappa$ .

Furthermore, the distributions of multiple capsules in torus vessels and straight tubes are investigated. In torus vessels, at higher  $\phi$ , the capsules tend to concentrate in the symmetrical plane at low  $Re$  and tend to be trapped by the Dean's vortices at high  $Re$ . Besides, for both the dilute and semidilute regime, the scaling law connecting the effective viscosity and the average location of capsules is proposed. Our data well support the scaling. Our findings may be useful to guide tube design for capsule transportation.

#### ACKNOWLEDGMENTS

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