

## High-order moments of streamwise fluctuations in a turbulent channel flow with spanwise rotation

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It is well known that the spanwise rotation in turbulent channel flow alters the mean velocity distribution to a linear law. In the present work, we have studied the higher-order moments of the streamwise fluctuations in a turbulent channel flow with spanwise rotation. Our results show that in a significant part of the channel the  $2p$ -order moments, raised by the power  $1/p$  with  $p = 1, 2, \dots, 6$ , also follow linear behavior according to  $((u'^+)^{2p})^{1/p} = a_p(y/h) + b_p$ . Here,  $u'^+$  is the streamwise velocity fluctuation normalized by the global friction velocity,  $h$  is the channel half width, and  $b_p$  and  $a_p$  are the intercept and the slope, respectively, which vary with Reynolds and rotation numbers. The linear regions can be extended by introducing a self-similar scaling, that is,  $2p$ -order moments as a function of  $2q$ -order moments. The slopes in the self-similar scaling  $a_p/a_1$  do not reveal sub-Gaussian behavior as in nonrotating wall-bounded flows, but rather Gaussian or super-Gaussian behaviors.

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Wall-bounded turbulence is ubiquitous in nature as well as engineering applications, and it has been studied extensively. In wall-bounded turbulence without separations or system rotation, the most well-known feature is the universal logarithmic relation between the mean streamwise velocity and the distance to the wall ( $y$ ) in the inertial region [1–3], that is,

$$\langle u \rangle / u_\tau \equiv \langle u \rangle^+ = \frac{1}{\kappa} \ln \left( \frac{u_\tau y}{\nu} \right) + C.$$

Here,  $u_\tau = \sqrt{\tau_w/\rho}$  is the friction velocity based on the wall friction stress  $\tau_w$  and the fluid density  $\rho$ ,  $\nu$  is the kinematic viscosity,  $\kappa$  is the von Kármán constant, and  $C$  is the intercept of the logarithmic law, which is also a constant at high Reynolds number. Many experimental and numerical studies of turbulent boundary layer, turbulent pipe flow, and turbulent channel flows have shown evidence of the above logarithmic law (see the reviews [4–6], and references therein).

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In addition to the mean streamwise velocity, the velocity fluctuations are very important quantities for wall-bounded turbulence, and another logarithmic behavior, motivated by the “attached eddy hypothesis” [7–9], was reported for the variance of the streamwise velocity fluctuations in the inertial region [5, 10–14],

$$\langle (u'^+)^2 \rangle = B_1 - A_1 \ln(y/\delta).$$

Here,  $u'^+ = (u - \langle u \rangle)/u_\tau$  is the normalized streamwise velocity fluctuation,  $\delta$  is the outer length scale, which is half of the channel width for a channel flow, the radius for a pipe flow, or the boundary layer thickness for a boundary layer flow.  $A_1 \approx 1.25$  is the Townsend-Perry constant [5, 10, 12–15] while  $B_1$  is a parameter which depends on flow conditions and geometry. More recently, Meneveau and Marusic [16] found that the  $p$ th root of the  $2p$ -order moments of the streamwise velocity fluctuations also follow a generalized logarithmic law,

$$\langle (u'^+)^{2p} \rangle^{1/p} = B_p - A_p \ln(y/\delta) = D_p(\text{Re}_\tau) - A_p \ln(y^+), \quad (1)$$

where  $y^+ = y/l_v = yu_\tau/\nu$  is the wall distance in viscous units,  $\text{Re}_\tau = u_\tau\delta/\nu$  is the friction Reynolds number,  $D_p = B_p + A_p \ln \text{Re}_\tau$  is a Reynolds-number-dependent offset, and  $A_p$  is the slope of the logarithmic law. They reported that the slopes  $A_p$ , which are lower than the Gaussian prediction  $A_1[(2p-1)!!]^{1/p}$ , appear quite insensitive to the Reynolds number, suggesting the existence of universal behavior in wall-bounded turbulence. This generalized logarithmic law for high-order moments can also be explained by the attached eddy hypothesis, assuming that the eddy population density is inversely proportional to the wall distance while the length scales of the eddies are directly proportional to the wall distance [16–18]. Furthermore, by using this attached eddy hypothesis, other logarithmic scalings of generalized high-order two-point correlations can be motivated. These were confirmed for turbulent boundary layer [18, 19]. These logarithmic scalings can also be predicted using momentum generating functions [20, 21] and have been observed in large-eddy simulation [22]. Generalized high-order statistics can be used as a diagnostic tool in numerical simulations as well as experimental measurements [23, 24].

Wall-bounded turbulence with system rotation is of great interest to engineering applications, geophysics, as well as astrophysics. As one typical well-defined canonical flow problem, turbulent channel flow subject to spanwise rotation has attracted a lot of attention from theoretical, experimental and numerical points of view [25–46]. It is generally accepted for a spanwise-rotating turbulent channel flow (RTCF) that turbulence is augmented in the part of the channel where the mean flow vorticity has an opposite sign to the system rotation (anticyclonic rotation), while it is suppressed in the other part where the two quantities are of the same sign (cyclonic rotation). Spanwise rotation alters the flow structures and flow statistics strongly in turbulent channel flow, and the most famous result concerns the mean velocity profile. In RTCF, the mean velocity exhibits a linear profile with a slope  $dU/dy \approx 2\Omega$  (here,  $U$  is the mean streamwise velocity and  $\Omega$  is the spanwise angular velocity), which implies a nearly zero absolute mean vorticity. This linear mean velocity profile was first reported in the experimental results by Johnston *et al.* [25], and then verified by others by experiments and numerical simulations. Recently, Xia *et al.* [42] reported other linear profiles in RTCF by using their direct numerical simulation (DNS) results at low friction Reynolds number  $\text{Re}_\tau = 180$  ( $\text{Re}_\tau = u_\tau\delta/\nu$  with  $u_\tau$  being the global friction velocity and  $\nu$  being the kinematic viscosity), such as the Reynolds shear stress and the production of turbulent kinetic energy. The newly reported linear profiles were verified by Brethouwer [46] in DNS at higher Reynolds numbers.

Although RTCF has been intensively studied, very few results are reported on high-order statistics. In this Rapid Communication, the high-order moments of streamwise fluctuations will be studied using DNS data from Xia *et al.* [42] and Brethouwer [46]. It is interesting to see that these high-order moments also follow linear behaviors in a certain part of the channel although the Reynolds number is quite low. The dependence of the slopes on rotation and Reynolds numbers will also be discussed.

The present work is based on an analysis of the DNS data presented in Refs. [42, 46]. Figure 1 shows the profiles of  $\langle (u'^+)^2 \rangle$  for different cases. In the DNS of Ref. [42]  $\text{Re}_\tau$  was kept constant and the bulk Reynolds number  $\text{Re}_b = U_b\delta/\nu$ , where  $U_b$  is the bulk mean velocity, varies with the

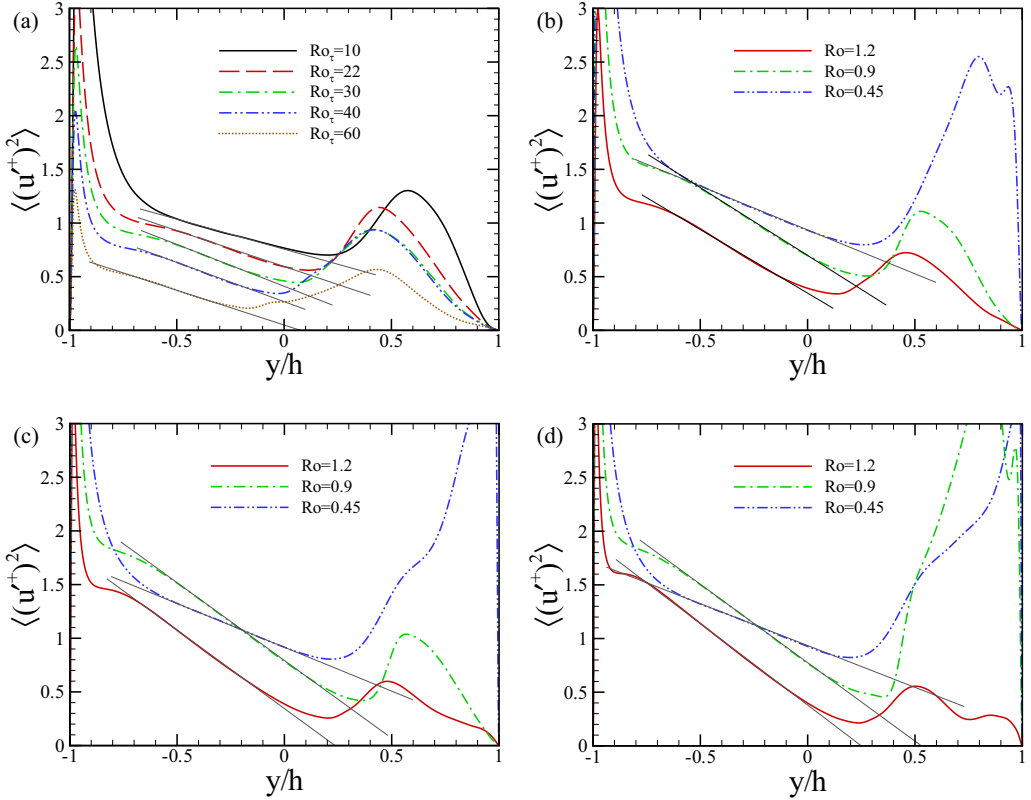


FIG. 1. Profiles of  $\langle (u'^+)^2 \rangle$ . (a) results at  $Re_\tau = 180$ ; (b) results at  $Re_b = 10000$ ; (c) results at  $Re_b = 20000$ , and (d) results at  $Re_b = 31600$ . The data are from Ref. [42] (a) and Ref. [46] (b)–(d). The thin solid lines show linear profiles with different slopes.

rotation rate, whereas in the DNS of [46] it was the other way around. When we refer to data from the first source we mention therefore  $Re_\tau$ , whereas when we refer to data from the second source we mention  $Re_b$ . Tables with  $Re_\tau$  together with  $Re_b$ ,  $Ro_\tau = 2\Omega\delta/u_\tau$  and  $Ro = 2\Omega\delta/U_b$  are given in both Refs. [42,46]. At lower Reynolds number cases as shown in Fig. 1(a), where the global friction Reynolds number is fixed at  $Re_\tau = 180$ ,  $\langle (u'^+)^2 \rangle$  decays with increasing rotation number  $Ro_\tau$  on the unstable pressure side. Away from the wall on the pressure side, a linear relation

$$\langle (u'^+)^2 \rangle = a_1(y/h) + b_1 \quad (2)$$

can be identified for all five  $Ro_\tau$ . This is within the region where the mean velocity profile displays a linear slope. However, the slopes at different  $Ro_\tau$  are not the same, but vary slightly. At higher Reynolds numbers, as displayed in Figs. 1(b)–1(d), where the bulk Reynolds numbers are fixed at  $Re_b = 10000$ ,  $20000$ , and  $31600$ , respectively,  $\langle (u'^+)^2 \rangle$  also has a linear slope away from the wall on the pressure side for all cases. The slopes vary more apparently than at lower Reynolds numbers. The linear slopes  $a_1$  of  $\langle (u'^+)^2 \rangle$  are plotted as function of  $Ro$  and  $Ro_\tau$  in Fig. 2. It is seen that  $a_1$  first decreases with  $Ro$  or  $Ro_\tau$  and then increases with  $Ro$  or  $Ro_\tau$  at all Reynolds numbers considered. At lower Reynolds numbers,  $a_1$  is above  $-1$  and varies slowly with  $Ro$ . At higher Reynolds numbers,  $a_1$  varies considerably with  $Ro$ , from a value around  $-0.8$  at  $Ro = 0.45$  to a value around  $-1.5$  at  $Ro = 0.9$  and  $1.2$  for  $Re_b \geq 20000$ . Note that in the DNS of Ref. [46] with a constant  $Re_b$ ,  $Re_\tau \leq 180$  for  $Re_b = 5000$  and  $Ro \geq 1.2$ , whereas in all other DNS from that source  $Re_\tau > 180$ . From the figure, we can infer that the slope varies more strongly with  $Ro$  at higher Reynolds numbers.

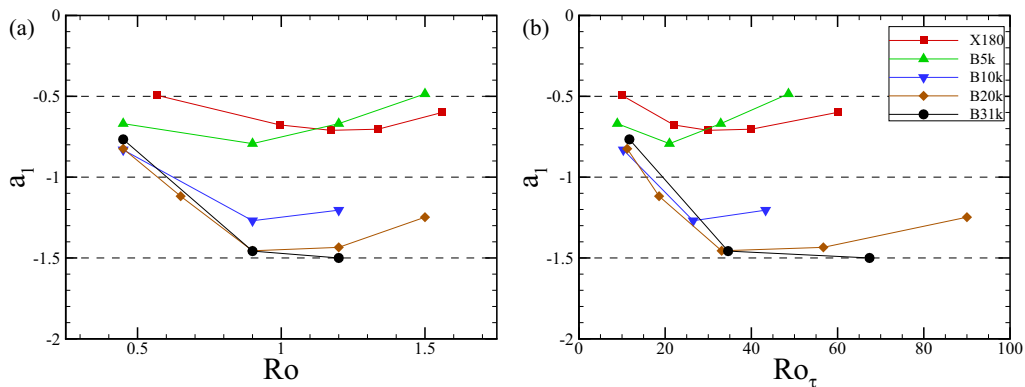


FIG. 2. Slopes  $a_1$  for the linear fitting of  $\langle(u'^+)^2\rangle$  as a function of (a)  $Ro$  and (b)  $Ro_\tau$ . “X180” refers to the data from Ref. [42] at  $Re_\tau = 180$ , while “B5k,” “B10k,” “B20k,” and “B31k” refer to the data from Ref. [46] at  $Re_b = 5000, 10\,000, 20\,000,$  and  $31\,600$ , respectively.

In nonrotating wall-bounded turbulence, the higher-order moments follow the same logarithmic behavior as the variance of the streamwise fluctuations  $\langle(u'^+)^2\rangle$ . In the present work, we have demonstrated that  $\langle(u'^+)^2\rangle$  follows a linear law at different Reynolds and rotation numbers. Now, we are going to show that the higher-order moments also follow a linear behavior. Figures 3(a) and 4(a) show the profiles of  $\langle(u'^+)^{2p}\rangle^{1/p}$  for  $p = 1, \dots, 6$  at  $Re_\tau = 180, Ro_\tau = 22,$  and  $Re_b = 31\,600, Ro = 1.2,$  respectively, the latter case corresponding to  $Re_\tau = 562$  and  $Ro_\tau = 67$ . The linear fitting

$$\langle(u'^+)^{2p}\rangle^{1/p} = a_p(y/h) + b_p \quad (3)$$

is also shown in the figure. The corresponding fitting coefficients  $a_p$  and  $b_p$  and the adjusted  $R$  square for  $p = 1, \dots, 6$  are listed in Tables I and II, respectively. It is clearly seen that linear profiles can be discerned for  $\langle(u'^+)^{2p}\rangle^{1/p}$  with  $p = 1, \dots, 6$  on the pressure side, again in the region where the mean velocity profile displays a linear slope. The higher-order moments of the two other velocity components (not shown) do not display a linear slope. In order to show more clearly how well is this linear fitting, the relative error of the linear fittings, which is defined as

$$\varepsilon = (\langle(u'^+)^{2p}\rangle^{1/p} - [a_p(y/h) + b_p]) / \langle(u'^+)^{2p}\rangle^{1/p} = 1 - [a_p(y/h) + b_p] / \langle(u'^+)^{2p}\rangle^{1/p}, \quad (4)$$

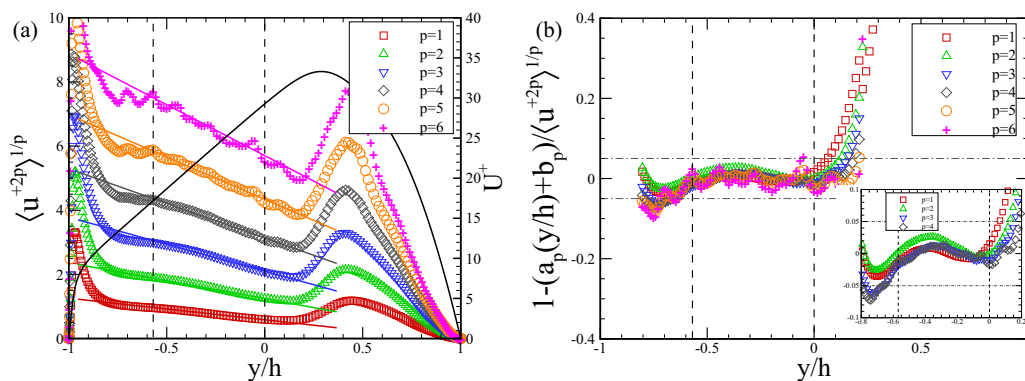


FIG. 3. (a) Profiles of  $\langle(u'^+)^{2p}\rangle^{1/p}$  and their linear fittings  $\langle(u'^+)^{2p}\rangle^{1/p} = a_p(y/h) + b_p$  for  $p = 1$  to  $6$ .  $U^+$  is also plotted to show the linear region for mean velocity (black solid line). (b) Relative error of the linear fitting  $1 - [a_p(y/h) + b_p] / \langle(u'^+)^{2p}\rangle^{1/p}$  for  $p = 1$  to  $6$ . The data is from Ref. [42] at  $Ro_\tau = 22$  and fitting coefficients are listed in Table I. Inset in (b) shows the zoomed-in behavior for  $p = 1$  to  $4$ .

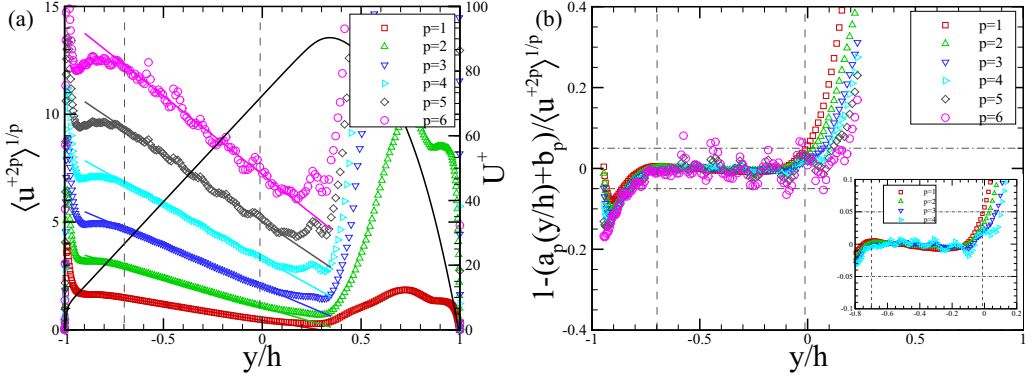


FIG. 4. (a) Profiles of  $\langle (u^+)^{2p} \rangle^{1/p}$  and their linear fittings  $\langle (u^+)^{2p} \rangle^{1/p} = a_p(y/h) + b_p$  for  $p = 1$  to 6.  $U^+$  is also plotted to show the linear region for mean velocity (black solid line). (b) Relative error of the linear fitting  $1 - [a_p(y/h) + b_p] / \langle (u^+)^{2p} \rangle^{1/p}$  for  $p = 1$  to 6. The data is from Ref. [46] at  $Re_b = 31\,600$  and  $Ro = 1.2$ . Inset in (b) shows the zoomed-in behavior for  $p = 1$  to 4.

is depicted in Figs. 3(b) and 4(b). It shows that the relative errors of these linear fittings are quite low; they are within 5% in the region  $-0.7 \lesssim y/h \lesssim 0$  for most cases, which demonstrates the present good linear fittings. This is consistent with the adjusted  $R$ -square values listed in Tables I and II, which are larger than 0.94 for  $p = 1$  to 6. It is seen in Figs. 3 and 4 that larger deviations exist for higher-order moments with  $p \geq 5$ ; for lower-order moments with  $p \leq 4$  the deviations are much smaller. In order to show the good fittings for lower-order moments, the relative errors for  $p = 1, \dots, 4$  are enlarged in the insets shown in Figs. 3(b) and 4(b). For the lower Reynolds number case, the relative errors are 3%, while in the higher Reynolds number case the errors are within 1% for  $p \leq 4$ . We may infer that the linear behavior is more apparent at high Reynolds numbers.

Figures 3(a) and 4(a) also include the mean velocity profiles  $U^+$ . This shows that the linear region for  $\langle (u^+)^{2p} \rangle^{1/p}$  is smaller than the linear region for the mean velocity profile. This is in contrast to the nonrotating cases. In wall-bounded turbulence without system rotation, Marusic *et al.* [14] identified the logarithmic law for the mean velocity and  $\langle (u^+)^2 \rangle$  within the range  $3Re_\tau^{1/2} < y^+ < 0.15 Re_\tau$ , while Meneveau and Marusic [16] found in the range  $y^+ > 400$  and  $y/\delta < 0.3$  ( $\delta$  is the boundary layer thickness) a fit to the logarithmic laws for  $\langle (u^+)^{2p} \rangle^{1/p}$ . Stevens *et al.* [22], on the other hand, used the range  $0.04 \leq y/\delta \leq 0.23$  to fit their large-eddy simulation data for  $\langle (u^+)^{2p} \rangle^{1/p}$  to the logarithmic laws. In other words, in wall-bounded turbulence without system rotation, the logarithmic regions for the mean velocity and  $\langle (u^+)^{2p} \rangle^{1/p}$  match and are within  $y/\delta < 0.3$ . In contrast, in spanwise-rotating turbulent channel flow, the linear profile for the mean velocity is quite wide (its extent is wider than  $0.5h$  in some cases), while the linear regions for  $\langle (u^+)^{2p} \rangle^{1/p}$  are much narrower. Furthermore, the extent of the region with these linear profiles depends on the Reynolds and rotation numbers.

In order to further verify the linear regions, plots of  $\langle (u^+)^{2p} \rangle^{1/p}$  against  $\langle (u^+)^2 \rangle$  and  $\langle (u^+)^6 \rangle^{1/3}$  for  $p = 1$  to 6 at  $Re_b = 31\,600$  and  $Ro = 1.2$  are depicted in Fig. 5. From Eq. (3), it is easy to obtain

TABLE I. Linear fitting information for the case with  $Ro_\tau = 22$  from Ref. [42]. The fitting region is  $-0.5691 \leq y/h \leq 0$ .

$y = a_p x + b_p$	$p = 1$	$p = 2$	$p = 3$	$p = 4$	$p = 5$	$p = 6$
$a_p$	-0.6736	-1.188	-1.682	-2.158	-2.637	-3.176
$b_p$	0.5843	1.244	2.099	3.135	4.342	5.703
Adjusted $R$ square	0.9978	0.9954	0.9923	0.9921	0.9890	0.9417

TABLE II. Linear fitting information for the case with  $Ro = 1.2$  and  $Re_b = 31\,600$  from Ref. [46]. The fitting region is  $-0.6984 \leq y/h \leq -0.0123$ .

$y = a_p x + b_p$	$p = 1$	$p = 2$	$p = 3$	$p = 4$	$p = 5$	$p = 6$
$a_p$	-1.500	-2.680	-3.889	-5.065	-6.174	-7.286
$b_p$	0.4405	1.054	1.985	3.296	5.035	7.206
Adjusted $R$ square	0.9995	0.9998	0.9998	0.9991	0.9939	0.9616

a relation between  $\langle(u'^+)^{2p}\rangle^{1/p}$  and  $\langle(u'^+)^{2q}\rangle^{1/q}$ ,

$$\langle(u'^+)^{2p}\rangle^{1/p} = \frac{a_p}{a_q} \langle(u'^+)^{2q}\rangle^{1/q} + \frac{a_q b_p - a_p b_q}{a_q}, \quad (5)$$

where  $q$  is a positive integer that differs from  $p$ . This relation implies that a linear region can also be identified in the plot of  $\langle(u'^+)^{2p}\rangle^{1/p}$  against  $\langle(u'^+)^{2q}\rangle^{1/q}$  with the slope being  $a_p/a_q$ . A similar relation between high-order moments and mean velocity profiles was proposed by Örlü *et al.* [24] as a diagnostic scaling. It is apparent from Fig. 5 that linear regions can be obtained for  $\langle(u'^+)^{2p}\rangle^{1/p}$  against  $\langle(u'^+)^2\rangle$  and  $\langle(u'^+)^6\rangle^{1/3}$  for  $p = 1$  to 6 with slopes  $a_p/a_1$  and  $a_p/a_3$ , respectively. More importantly, the linear regions with this new self-similar scaling are wider than the original ones with high-order moments profiles as a function of the wall distance, whose extent is marked by the two vertical dash-dotted lines in Fig. 5. Yang *et al.* [21] showed that the power-law scaling of the momentum generation functions (MGFs) for the streamwise velocity fluctuations can be extended to include the bulk and viscosity-affected regions  $30 < y^+$ ,  $y < \delta$  if the data were interpreted with a self-similar scaling of the MGFs as a function of one reference value, implying extended self-similarity [47–53]. Accordingly, our results also reveal extended self-similarity.

In fact, for wall-bounded turbulent flows without system rotation, we can also derive a self-similar scaling from Eq. (1)

$$\langle(u'^+)^{2p}\rangle^{1/p} = \frac{A_p}{A_q} \langle(u'^+)^{2q}\rangle^{1/q} + \frac{A_q B_p - A_p B_q}{A_q}, \quad (6)$$

which has the same form as Eq. (5). That is, the self-similar scaling of  $\langle(u'^+)^{2p}\rangle^{1/p}$  as a function of  $\langle(u'^+)^{2q}\rangle^{1/q}$  follows a linear law with the slope  $A_p/A_q$  in nonrotating wall-bounded turbulence, although  $\langle(u'^+)^{2p}\rangle^{1/p}$  itself exhibits a logarithmic law. Specifically, if we set  $q = 1$ , then the slope

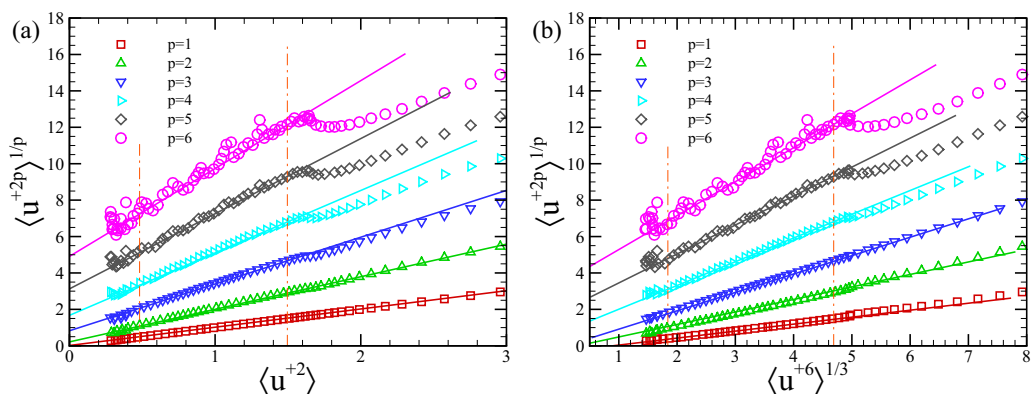


FIG. 5. (a) Plot of  $\langle(u'^+)^{2p}\rangle^{1/p}$  against  $\langle(u'^+)^2\rangle$  for  $p = 1$  to 6. The solid lines indicate the slopes  $a_p/a_1$ . (b) Plot of  $\langle(u'^+)^{2p}\rangle^{1/p}$  against  $\langle(u'^+)^6\rangle^{1/3}$  for  $p = 1$  to 6. The solid lines indicate the slopes  $a_p/a_3$ , while the vertical dash-dotted lines show the bounds of the linear regions depicted in Fig. 4 with  $-0.6984 \leq y/h \leq -0.0123$ . The data is from Ref. [46] at  $Re_b = 31\,600$  and  $Ro = 1.2$ .

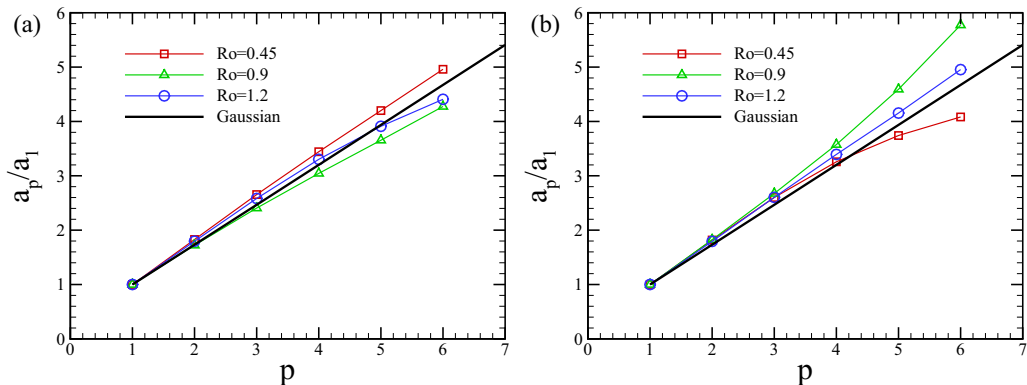


FIG. 6. Slopes of the self-scaling  $a_p/a_1$  at different Reynolds number and rotation number. (a)  $Re_b = 20\,000$  and (b)  $Re_b = 31\,600$ . The Gaussian values are also shown for comparison.

becomes  $A_p/A_1$ . As shown by Meneveau and Marusic [16], if the streamwise fluctuations have a Gaussian behavior, then  $A_p/A_1 = [(2p - 1)!!]^{1/p}$ . However, in the wall-bounded turbulent flows without system rotation, the streamwise fluctuations display sub-Gaussian statistics and the slope  $A_p/A_1$  falls below the Gaussian values  $[(2p - 1)!!]^{1/p}$ . In spanwise-rotating turbulent channel flows, the system rotation complicates the flow statistics. In Fig. 6, the self-scaling slopes  $a_p/a_1$  at  $Re_b = 20\,000$  and  $31\,600$  are shown with reference to Gaussian values. Different from the nonrotating cases, the statistics are no longer sub-Gaussian for most cases considered, but approximately Gaussian or super-Gaussian. However, there is no clear agreement for the results at the two  $Re_b$ , implying that we cannot draw strong conclusions regarding the Gaussian or super-Gaussian behavior.

As presented above, the attached eddy hypothesis can be used to predict the logarithmic scaling of the single-point high-order moments of the streamwise velocity fluctuations in wall-bounded turbulent flows without system rotation. Brethouwer [46] found evidence that attached eddies also populate spanwise-rotating channel flow at lower rotation rates, but at higher rotation rates (e.g.,  $Ro \gtrsim 0.9$  and  $Re_b = 20\,000$ ), they appear much less prominent. Therefore, we cannot use the attached eddy hypothesis to explain the linear profiles observed in spanwise-rotating channel flows. Nevertheless, if we adopted the attached eddy hypothesis but assumed that the eddy population density is constant in a certain region, it would lead to a linear profile for the even-order moments of the streamwise velocity fluctuations. However, a constant eddy population is very unlikely, therefore, this proposition is unlikely to give the correct physical explanation. As for the zero-absolute-mean-vorticity region of the mean velocity profile, more work needs to be done to find a proper physical explanation.

In conclusion, we studied the high-order moments of the streamwise fluctuations in spanwise-rotating turbulent channel flows. Our results showed that the high-order moments follow linear laws like the mean streamwise velocity. The linear regions for the higher-order moments are narrower than that for the mean velocity, but the linear regions can be extended if a self-similar scaling is used. Furthermore, our analysis showed that the flow statistics are no longer sub-Gaussian if the system rotates about the spanwise direction, but are approximately Gaussian or super-Gaussian. However, the Reynolds numbers studied are quite low. Data at higher Reynolds numbers should be used to further investigate the Reynolds and rotation number effect. Further studies are also required to find a proper physical explanation for the observed scaling behaviors.

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