

Diverging, but negligible power at Carnot efficiency: Theory and experimentViktor Holubec^{1,2,*} and Artem Ryabov²¹*Institut für Theoretische Physik, Universität Leipzig, Postfach 100 920, D-04009 Leipzig, Germany*²*Charles University, Faculty of Mathematics and Physics, Department of Macromolecular Physics, V Holešovičkách 2, CZ-180 00 Praha, Czech Republic*

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We discuss the possibility of reaching the Carnot efficiency by heat engines (HEs) out of quasistatic conditions at nonzero power output. We focus on several models widely used to describe the performance of actual HEs. These models comprise quantum thermoelectric devices, linear irreversible HEs, minimally nonlinear irreversible HEs, HEs working in the regime of low-dissipation, overdamped stochastic HEs and an underdamped stochastic HE. Although some of these HEs can reach the Carnot efficiency at nonzero and even diverging power, the magnitude of this power is always negligible compared to the maximum power attainable in these systems. We provide conditions for attaining the Carnot efficiency in the individual models and explain practical aspects connected with reaching the Carnot efficiency at large power output. Furthermore, we show how our findings can be tested in practice using a standard Brownian HE realizable with available micromanipulation techniques.

DOI: [10.1103/PhysRevE.96.062107](https://doi.org/10.1103/PhysRevE.96.062107)**I. INTRODUCTION**

Ever since first heat engines (HEs) appeared, engineers and physicists optimized their output power and efficiency [1]. The most influential theoretical result in this field was achieved by Carnot already in the beginning of 19th century [2]. Consider a HE which can communicate with heat baths at temperatures ranging from T_c to T_h . Then, regardless of the details of the machine, the ratio $\eta = W/Q_h$ of work done by the engine to heat accepted from surroundings is bounded from above by the Carnot efficiency $\eta_C = 1 - T_c/T_h$. Recently, a lot of studies discussed whether and how this efficiency can (or cannot) be actually attained in practice [3–13].

Two general classes of systems where the Carnot efficiency can be reached according to the second law are depicted in Fig. 1. These comprise HEs coupled simultaneously to two reservoirs at temperatures T_h and T_c (steady-state HEs, left) and HEs which operate periodically and are always coupled to a single bath at a time (periodic HEs, right). The Carnot efficiency can be attained also in mutations of these two classes, for example, by HEs coupled simultaneously to two baths and operated periodically [14,15]. All these machines share one important feature: they can communicate with reservoirs at the boundary temperatures T_h and T_c only.

Let us now focus on the periodic HEs of Fig. 1(b). Assuming that the both baths are ideal thermodynamic reservoirs (of infinite size and infinitely fast relaxation), the second law of thermodynamics states that the total amount of entropy produced per cycle, ΔS_{tot} , fulfills the inequality $\Delta S_{\text{tot}} = Q_c/T_c - Q_h/T_h \geq 0$, which leads to the relation $-Q_c/Q_h = -\Delta S_{\text{tot}} T_c/Q_h - T_c/T_h \leq -T_c/T_h$. Using further the definition $W = Q_h - Q_c > 0$ for the output work of the engine, $P = W/t_p$ for the output power and $\sigma = \Delta S_{\text{tot}}/t_p$ for the average amount of entropy produced per unit time during the cycle, we obtain the following expression for the efficiency

$$\eta = W/Q_h:$$

$$\eta = \frac{\eta_C}{1 + T_c \sigma / P} \leq \eta_C. \quad (1)$$

The same result (only with $\sigma = q_c/T_c - q_h/T_h$) holds also for the steady-state HEs, Fig. 1(a).

The inequality (1) shows that the Carnot efficiency can be reached if and only if $T_c \sigma / P \rightarrow 0$. A standard example where this occurs is the quasistatic limit of infinitely slow driving ($t_p \rightarrow \infty$). Then the system is in thermodynamic equilibrium during the whole cycle and $\sigma = 0$. However, in this limit, the output power of the engine vanishes. It was suggested only recently [12,13,16–18] that there exist other ways to achieve η_C .

First, Campisi and Fazio showed that η_C at nonzero output power can be attained in a HE working close to a critical point [12,16,17]. Second, the Carnot efficiency can be reached in the limit of infinitely fast dynamics [11,18]. It is important to note that both these suggestions lead to diverging heat flows through the system. In the critical HE, this is caused by the diverging heat capacity of the working fluid, for the infinitely fast dynamics, by diverging rates for processes of heat exchange with reservoirs. Such diverging energy currents represent an intuitive hallmark of devices reaching η_C out of equilibrium. Indeed, for a nonequilibrium process, one naturally assumes that $T_c \sigma > 0$. Then Eq. (1) implies that η_C can be reached only in the limit $P \rightarrow \infty$.

It should be noted that these suggestions for reaching the Carnot efficiency are based on idealized setups. In practice, these machines may work close to η_C , but they can never reach it, as discussed in Refs. [9,10,19,20]. The critical HE proposed in Ref. [16] exhibits diverging fluctuations of output work and power [20]. In a more general study, Shiraishi and Tajima [9] show that η_C cannot be reached once finite reservoir relaxation times are taken into account (see also Refs. [21,22]).

In the present paper, we answer three basic questions concerning attainability of the Carnot efficiency out of quasistatic conditions: (i) What is the magnitude of the output power of a HE operating with η_C ? (ii) For what parameters can the Carnot efficiency at nonzero power be attained in widely used models?

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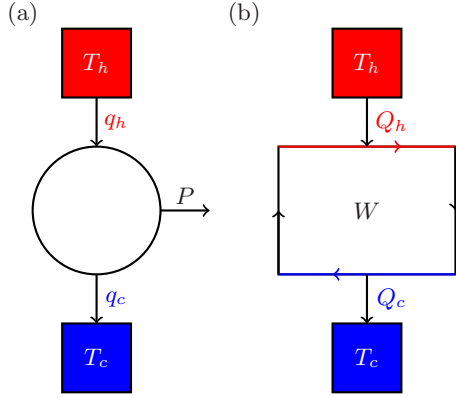


FIG. 1. Two major classes of HEs which can achieve the Carnot efficiency. These HEs communicate with reservoirs at temperatures T_h and T_c only. Steady-state HEs [panel (a)] are coupled simultaneously to both reservoirs and operate under time-independent conditions. They transform the difference between the steady heat influx q_h and the steady heat outflow q_c into the output power P . Periodically driven HEs coupled always to a single bath at a time [panel (b)] operate in a time-periodic nonequilibrium steady state. These engines accept the total amount of heat Q_h ($-Q_c$) from the hot bath (cold bath) per cycle of the duration t_p and deliver the average output power $P = W/t_p$. Black branches of the cycle are adiabats, and other branches are isotherms.

(iii) Can an actual HE operating close to η_C at large output power be constructed using currently available experimental techniques?

In Sec. II, we answer the first question for models where the upper bound on efficiency at given power is known. These comprise quantum thermoelectric HEs [23,24], linear irreversible HEs [25,26], HEs working in the regime of low-dissipation and overdamped stochastic HEs [27], minimally nonlinear irreversible HEs [28], and an underdamped stochastic HE [29]. The result is quite surprising. In all these models, η_C can be reached only at output powers which are vanishingly small as compared to the maximum power P^* attainable by the device. Reaching the Carnot efficiency thus may not be the most frequent goal in engineering practice where the magnitude of the output power often represents an important component of the figure of merit.

This allows us to generalize the well-known textbook wisdom that η_C can be reached only at vanishingly small power P to the following conjecture: the Carnot efficiency can be attained only at a vanishingly small ratio P/P^* . First, this fraction vanishes in the quasistatic limit ($P \rightarrow 0$). Second, it vanishes for a fixed nonzero output power P and diverging maximum power $P^* \rightarrow \infty$.

In Sec. III we answer the second question for the models mentioned above. The Carnot efficiency at nonzero power can be reached for reasonable parameter values just in linear response HEs, low-dissipation HEs, overdamped stochastic HEs, and minimally irreversible HEs. For these models, we propose a specific scaling of parameters inspired by Ref. [18] which can bring the efficiency arbitrarily close to η_C at $P > 0$. It turns out that this is possible both for a positive average entropy production σ where the power at η_C diverges and for a vanishing σ where the power can be both finite and diverging

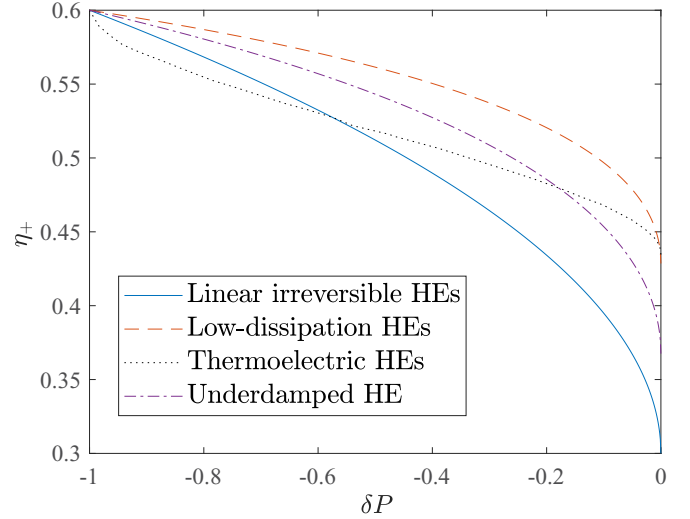


FIG. 2. Upper bound on the efficiency at given power for quantum thermoelectric HEs (dotted black line), for linear irreversible HEs (5) (full blue line), for low-dissipation HEs and minimally nonlinear irreversible HEs (11) (dashed orange line), and for HEs based on an underdamped Brownian particle in a breathing parabola potential (20) (dot-dashed purple line); $\eta_C = 3/5$.

(see Secs. IIIB and IIID, a similar result was obtained in Ref. [11]).

The answer to our third question is given in Sec. IV, where we describe how our results can be tested using a stochastic Brownian HE [30–38]. The HE consists of a Brownian particle diffusing in a harmonic trap with time-dependent stiffness and was already realized experimentally [30–32,38]. We present a realistic scaling of the stiffness under which the power and efficiency of the HE increase at the same time. Moreover, in contrast to the critical HE [16,20], this scaling leads to bounded relative power fluctuations and hence the suggested stochastic HE can operate efficiently with a well-defined output power.

II. BOUNDS ON MAXIMUM EFFICIENCY AT GIVEN POWER

Upper bounds on efficiency at given power obtained in the studies [23–25,27–29] can be effectively written using the variable [39,40]

$$\delta P = \frac{P - P^*}{P^*}, \quad \delta P \in [-1, 0], \quad (2)$$

which measures relative change in power P with respect to the maximum power P^* achievable in a given setup.

Let us denote the maximum efficiency attainable at given power as $\eta_+(\delta P)$. The individual upper bounds for all the models mentioned in the Introduction are plotted in Fig. 2. All the curves monotonously increase with decreasing δP from the efficiency at maximum power ($\delta P = 0$) to the Carnot efficiency ($\delta P = -1$). In all the models, η_C can thus be reached under the condition $P/P^* \rightarrow 0$ only. Hence we have $\eta = \eta_C + f(P/P^*)$, $\lim_{x \rightarrow +0} f(x) = 0$. For the models considered here, the function $f(x)$ is given by the power law $f(x) = cx^\theta$ with a negative constant c ($\eta \leq \eta_C$) and a

positive exponent θ . Close to η_C , it follows from Eq. (1) that $\eta = \eta_C - \eta_C \sigma T_c / P$, and we thus have

$$\eta - \eta_C \approx -\eta_C \frac{\sigma T_c}{P} = f\left(\frac{P}{P^*}\right) = c\left(\frac{P}{P^*}\right)^\theta \leq 0. \quad (3)$$

In case the maximum power P^* is finite, η_C can be reached only at vanishingly small power P and thus at quasistatic conditions. The possibility to attain the Carnot efficiency at nonzero or even diverging power opens only when the maximum power diverges faster than P .

This suggests that reaching η_C at $P > 0$ may not be the holy grail of engineering practice where a tradeoff between power and efficiency is often optimized [41–48]. Consider, for example, the target function $\xi = \eta^\alpha P^\beta$, $\alpha \geq 0$, $\beta \geq 0$, which should be optimized. If we denote as ξ_C and ξ^* the values of this function at maximum efficiency and at maximum power, respectively, we find that the condition $P_C/P^* = 0$ yields $\xi_C/\xi^* = 0$ whenever $\beta \neq 0$. Our findings thus encourage the struggle for reaching η_C only if the ultimate performance goal is the maximum efficiency (for example at a fixed value of output power). In such a case, the advantage of HEs working close to η_C under nonequilibrium conditions is their typically large power output (which is still negligible as compared to P^*). A practical disadvantage resides in preparation of a working medium and/or operational cycle for such engines, which must be tuned in a special way.

Having described the general formulation of our findings, let us now turn to particular examples of HEs. Below, we review the bounds obtained for the individual model HEs and discuss the conditions under which these HEs can achieve the Carnot efficiency. In Sec. III A we discuss short thermoelectric HEs and Sec. III B contains an extensive discussion of linear response HEs. In Sec. III C we discuss in detail low-dissipation HEs in general, and Sec. III D is devoted to a HE based on an underdamped particle diffusing in an externally controlled parabolic potential. Finally, in Sec. IV, we propose a specific setup where η_C at $P > 0$ can be reached in experiments with Brownian particles diffusing in an externally controlled potential. We give a detailed model study including all experimentally relevant parameters.

From a practical point of view, the scalings presented in Secs. III B, III C, and IV must be understood as recipes on how to set model parameters in order to attain efficiencies close to η_C at large output power, as it is presented in Figs. 4 and 5. For $\eta = \eta_C$, the parameters in the individual models either diverge (Onsager coefficients for linear response HEs) or vanish (the cycle duration for low-dissipation HEs and for the Brownian HE). Such extreme values are experimentally inaccessible and challenge the validity of basic assumptions underlying the individual models.

III. EXAMPLES

A. Quantum thermoelectric heat engines

Thermoelectric HEs are connected to two thermodynamic reservoirs at different chemical potentials and temperatures (see Ref. [49] for the latest review). They are operating in a nonequilibrium steady state [Fig. 1(a)] using the temperature gradient for pumping electrons against the gradient of chemical

potential. For quantum thermoelectric HEs operating under vanishing magnetic fields, the upper bound on efficiency can be written analytically only for $\delta P \rightarrow -1$, where it reads [23,24]

$$\eta_+(\delta P) = \eta_C [1 - 0.478 \sqrt{(1 - \eta_C)(1 + \delta P)}]. \quad (4)$$

For the remaining values of δP , the curve can be obtained numerically [23,24]. The resulting curve depicted in Fig. 2 always exhibits the general features described above.

This bound was derived by Whitney [23,24] using non-linear Landauer-Büttiker scattering theory, and it is generally valid for all systems which can be modeled by this theory. In particular, the bound may not be valid once the time-reversal symmetry of the underlying dynamics is broken, for example, by introducing magnetic fields into the system.

The maximum power corresponding to the bound (4) is given by $P^* = A_0 \pi^2 N k_B^2 / h (T_h - T_c)^2$, where $A_0 \pi^2 k_B^2 / h$ is a constant and N is the number of transverse modes in the narrowest part of the quantum system. Assuming that N is finite, the bound (4) suggests that η_C at $P > 0$ may be reached in the unrealistic case of diverging temperature difference, $T_h - T_c \rightarrow \infty$, only.

When the efficiency of the thermoelectric reaches η_C , the entropy production scales as $\sigma \propto P^{3/2}$ [23,24]. The ratio σ/P thus scales as $\sigma/P \propto \sqrt{P}$, leading to the exponent $\theta = 1/2$ in Eq. (3). This can be checked by the direct calculation from Eq. (4).

B. Linear irreversible heat engines

Let us now focus on HEs working in the linear response regime. The linear response formalism is valid for an arbitrary system if (thermodynamic) forces acting on it are small enough that the system operates close to thermal equilibrium. The formalism can be applied both to HEs operating in a nonequilibrium steady state caused by their simultaneous coupling to two (or more) reservoirs at different temperatures [Fig. 1(a)] and to cyclic HEs, which are connected only to a single bath at a time [Fig. 1(b)]. Because the two classes of models were shown to be equivalent [6,50], we limit our discussion to steady-state models only.

Assuming that the Onsager matrix describing the linear model is symmetric, the upper bound on efficiency at given power is given by [25]

$$\eta_+(\delta P) = \frac{\eta_C}{2} (1 + \sqrt{-\delta P}). \quad (5)$$

For nonsymmetric Onsager matrices, the Carnot efficiency can be reached in principle even at maximum power [3] in the limit $L_{12}/L_{21} \rightarrow \infty$ if one considers just the restriction imposed by the second law. However, for example, for thermoelectric HEs a detailed analysis of Onsager coefficients strongly suggests that the power vanishes at least linearly when η reaches η_C [51]. This result narrows the way to a thermoelectric working in the linear response regime at $\eta = \eta_C$ and $P > 0$. Nevertheless, such a working regime may be still attainable even for a symmetric Onsager matrix based on considerations of Ref. [18].

We demonstrate this possibility on the simple model comprising just two thermodynamic forces X_1 , X_2 and fluxes J_1 , J_2 related by a symmetric Onsager matrix:

$$J_1 = L_{11}X_1 + L_{12}X_2, \quad (6)$$

$$J_2 = L_{12}X_1 + L_{22}X_2. \quad (7)$$

The first thermodynamic force $X_1 = F/T$ determines the output power $P = -J_1X_1T$ (F is the load attached to the engine and T is the system temperature). The second thermodynamic force $X_2 = (T_h - T_c)/T^2$ impels the heat flux J_2 from the hot reservoir to the system. The engine efficiency is thus defined as $\eta = P/J_2$.

The maximum power $P^* = \eta_C^2 L_{22} q^2 T/4$ is in this system attained for the load $X_1^* = -L_{12}X_2/(2L_{11})$ at the efficiency $\eta^* = 0.5\eta_C q^2/(2 - q^2)$ [25,52]. The constant, $q^2 = L_{12}^2/(L_{11}L_{22})$, characterizes coupling between the fluxes J_1 and J_2 , which become proportional for $q^2 = 1$. The definition of the entropy production in the system, $\sigma = J_1X_1 + J_2X_2$, together with the second law, $\sigma > 0$, implies the following limitations for the Onsager coefficients: $L_{11} \geq 0$, $L_{22} \geq 0$, and $L_{11}L_{22} - L_{12}^2 \geq 0$. These restrictions give the bounds $-1 \leq q \leq 1$ for the coupling constant q .

With these definitions and results, the maximum efficiency at given power can be written as [25]

$$\eta = \eta^*(1 + \delta P) \frac{2 - q^2}{2 - q^2(1 + \sqrt{-\delta P})}. \quad (8)$$

Optimization of this result with respect to q^2 gives the formula (5) and the optimal value of the parameter $q = 1$. To sum up, η_C can be in this model attained only when $q = 1$ and $\delta P = -1$.

The ratio P/P^* can be written as $P/P^* = (2 - X_1/X_1^*)X_1/X_1^*$. Thus the condition $\delta P = -1$ implies either $X_1 = 0$ or $X_1/X_1^* = -(2L_{11}X_1)/(L_{12}X_2) = 2$, and from $q^2 = 1$ it follows that $L_{12}/L_{11} = L_{22}/L_{12}$. When both these conditions are strictly satisfied, the output power P as well as the entropy production σ vanish, meaning that η_C is reached under quasistatic conditions.

However, the way is not completely closed yet, as suggested in Refs. [8] and [18]. This is because we can set $q^2 = 1$ and ensure that δP approaches -1 asymptotically such that neither P nor σ vanish and, at the same time, η approaches η_C . Setting $L_{12}/L_{11} = L_{22}/L_{12}$ ($q^2 = 1$), output power and entropy production can be written as

$$P = -L_{12}X_1X_2 \left(1 + \frac{L_{11}X_1}{L_{12}X_2} \right) T, \quad (9)$$

$$\sigma = L_{12}X_1X_2 \left(\frac{L_{11}X_1}{L_{12}X_2} + 2 + \frac{L_{12}X_2}{L_{11}X_1} \right). \quad (10)$$

For $q^2 = 1$, only two free parameters remain, e.g., L_{11} and L_{12} . We now let them diverge and assume that thermodynamic forces X_1 and X_2 are bounded, which is reasonable in the linear response regime. Instead of working directly with the two Onsager coefficients, it is convenient to split the flux J_1 such that $L_{12}X_2 = (J_\infty + J_1)$ and $L_{11}X_1 = -J_\infty$. Assuming that $J_\infty \gg J_1 > 0$, the condition $-L_{11}/L_{12} = X_2/X_1$ (or $\delta P = -1$) is strictly obeyed in the limit $J_\infty \rightarrow \infty$ only. Inserting

these expressions into Eqs. (9) and (10), we determine asymptotic behavior of σ , P , and P^* as $J_\infty \rightarrow \infty$. We obtain $\sigma \approx -X_1J_1^2/J_\infty$, $P \approx X_2J_1$, and $\sigma/P \approx J_1/J_\infty$ for their ratio, which must vanish as $\eta \rightarrow \eta_C$. The ratio P/P^* reads $P/P^* \approx 4J_1/(T^2J_\infty)$, and hence it vanishes in the same way as $\sigma T/P$. The exponent θ in Eq. (3) thus equals 1.

The Carnot efficiency at nonzero power can thus be reached for $J_\infty \rightarrow \infty$. Let us assume that $J_1 \propto J_\infty^\kappa$, $\kappa < 1$. Then the power diverges as $J_\infty \rightarrow \infty$ whenever $\kappa > 0$ and is finite otherwise. In the same limit, the entropy production σ vanishes whenever $\kappa < 1/2$, is constant for $\kappa = 1/2$, and diverges otherwise. By a proper choice of Onsager coefficients, one can hence increase the output power, reach efficiencies arbitrarily close to η_C , and at the same time, suppress the entropy production.

To conclude, the Carnot efficiency can be reached by properly tuning parameters of the working medium encoded in Onsager coefficients. To get close to η_C , these coefficients must attain relatively large values. In practice, large Onsager coefficients can be obtained for binary mixtures near their critical point (see Ref. [53] and the references therein). Our results thus provide further evidence that η_C at $P > 0$ can be attained by HEs working close to a critical point, as suggested in Ref. [16].

C. Low-dissipation and minimally nonlinear irreversible heat engines

For low-dissipation HEs [27] and for minimally nonlinear irreversible HEs [28], the upper bound on efficiency at given power reads

$$\eta_+(\delta P) = \eta_C \frac{1 + \sqrt{-\delta P}}{2 - (1 - \sqrt{-\delta P})\eta_C}. \quad (11)$$

The minimally nonlinear irreversible HEs represent straightforward generalization of linear irreversible HEs discussed in the preceding section. Within this generalization, it is assumed that the formulas (6) and (7) for the currents also contain quadratic terms of the type $\kappa_i J_i^2$. The setup used to derive the bound (11) for minimally nonlinear irreversible HEs [28] is mathematically equivalent to the low-dissipation model [54]. The latter is discussed in next paragraphs, including a straightforward physical interpretation of its basic assumptions.

The low-dissipation model used to derive the bound (11) is depicted schematically in Fig. 1(b), and it can be mapped onto the Brownian HE depicted in Fig. 3, see Refs. [27,39,40,55–57]. In Fig. 3, the filled blue Gaussian represents the probability density function (PDF) for the position of a Brownian particle driven by a time-dependent potential (black line). During the cycle, the system is first attached for time t_h to the hot bath and the potential widens (the isothermal expansion). Then the bath temperature changes from T_h to T_c , and the potential further opens. We assume that this happens so fast that the PDF remains unchanged (the adiabatic step). After that, the system is attached for time t_c to the cold bath and the potential shrinks (the isothermal compression). Finally, the nonequilibrium Carnot cycle is closed as the temperature and the potential jump to their initial values (the adiabatic step).

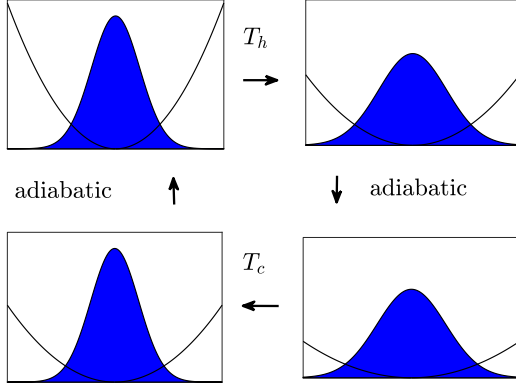


FIG. 3. Sketch of the operational cycle of HEs based on a particle diffusing in a time-dependent potential considered in Secs. III C, III D, and IV. The filled blue curve stands for the probability density for the position of the particle. The black parabola represents the potential at the beginning of the individual branches.

The system performs work during the first two strokes when the potential opens and consumes work in the rest of the cycle. The described cycle was already realized experimentally using a colloidal particle manipulated by optical tweezers by Blickle and Bechinger [31]. We refer to Refs. [58–61] for a discussion of different adiabatic and isothermal processes on microscale.

The main assumption of the low-dissipation model [55,62,63] is that the heat exchanged with the individual reservoirs during the isotherms can be written as

$$Q_h = T_h \Delta S - A_h/t_h, \quad (12)$$

$$Q_c = T_c \Delta S + A_c/t_c, \quad (13)$$

leading to the entropy produced per cycle,

$$\Delta S_{\text{tot}} = \frac{A_h}{t_h T_h} + \frac{A_c}{t_c T_c}. \quad (14)$$

Here, A_h and A_c are positive parameters independent of the times t_h and t_c , and ΔS is the change of the system entropy during the isothermal expansion. Low-dissipation HEs thus work at vanishing entropy production under quasistatic conditions ($t_h \rightarrow \infty$ and $t_c \rightarrow \infty$).

Although the assumptions (12)–(13) may look oversimplified, there exist several real systems which fit into the scheme. Examples are nanomotors based on two-level quantum systems [64] and various overdamped Brownian HEs [32,39,40,56,65]. In general, jumps in the temperature at the ends of the isotherms bring the system far from equilibrium, leading to additional terms in the total entropy production (14) which would not vanish in the long-time limit. To avoid this, one should drive the system in such a way that it is in equilibrium both before and after the jump. While this can be relatively easily achieved for large systems, it might require a precise control of the system dynamics on the microscale [32,39,40,56,57,64–66].

Let us now investigate if low-dissipation HEs can operate with the Carnot efficiency at nonzero power. In Sec. IV, we will exemplify the obtained results using an exactly

solvable Brownian HE [39,40,56,57] which can be realized experimentally [30–32,38].

Because the adiabatic branches are (infinitely) faster than the isotherms, the total duration of the cycle is given by $t_p \approx t_h + t_c$. Introducing the parameter α through the formulas $t_h = \alpha t_p$ and $t_c = (1 - \alpha)t_p$, the power of a low-dissipation HE is given by [56]

$$P = \frac{(T_h - T_c)\Delta S}{t_p} - \frac{(1 - \alpha)A_h + \alpha A_c}{t_p^2 \alpha(1 - \alpha)}. \quad (15)$$

Optimization of the power with respect to t_p and α gives

$$\alpha^* = \frac{A_h - \sqrt{A_h A_c}}{A_h - A_c}, \quad (16)$$

$$t_p^* = \frac{2}{T_h \eta_C \Delta S} (\sqrt{A_h} + \sqrt{A_c})^2, \quad (17)$$

$$P^* = \frac{1}{4} \left(\frac{T_h \eta_C \Delta S}{\sqrt{A_h} + \sqrt{A_c}} \right)^2, \quad (18)$$

$$\eta^* = \frac{\eta_C (1 + \sqrt{A_c/A_h})}{2(1 + \sqrt{A_c/A_h}) - \eta_C}. \quad (19)$$

The upper bound for efficiency at maximum power, $\eta_+(0) = \eta_C/(2 - \eta_C)$, and also the bound (11) are obtained in the limit $A_c/A_h \rightarrow 0$. Nevertheless, qualitatively similar bounds as (11) apply for arbitrary A_c and A_h (see Fig. 2(b) in Ref. [27]).

The Carnot efficiency at nonzero power can be attained only if the maximum power diverges, which occurs for $(\sqrt{A_h} + \sqrt{A_c})^2 \rightarrow 0$. To approach this condition asymptotically, we assume that the coefficients scale as $A_c \propto A_\infty^{-\phi}$ and $A_h \propto A_\infty^{-\delta}$, $A_\infty \rightarrow \infty$. Then both $A_c/A_h \rightarrow 0$ and $(\sqrt{A_h} + \sqrt{A_c})^2 \rightarrow 0$ whenever $0 < \delta < \phi$. The condition $(\sqrt{A_h} + \sqrt{A_c})^2 \rightarrow 0$ alone is fulfilled for $0 < \phi \leq \delta$.

Vanishing coefficients A_c and A_h lead to a vanishing entropy production (14) unless the cycle duration t_p also goes to zero. This occurs in the regime of maximum power, where t_p (17) scales as $t_p^* \propto A_h$. Larger efficiencies than η^* are obtained for $t_p > t_p^*$ [27]. Let us thus assume that $t_p = t_p^* A_h^{-\kappa}$, $\kappa > 0$, and $\alpha = \alpha^*$.

Using this scaling we obtain the relation $P \approx (T_h \eta_C \Delta S)^2 A_h^{\kappa-1}/2$ for power and the formula $\sigma/P = A_h^\kappa/(2T_h)$ for the ratio, which must vanish at $\eta = \eta_C$. The power at the Carnot efficiency is constant for $\kappa = 1$ and diverges for $0 < \kappa < 1$. The entropy produced per cycle scales as $\Delta S_{\text{tot}} \approx 2A_h^\kappa$, and the average entropy produced per unit time is given by $\sigma = T_h (\eta_C \Delta S)^2 A_h^{2\kappa-1}/2$. We hence see that the average entropy produced per cycle vanishes whenever $\eta = \eta_C$ at $P > 0$. The entropy produced per unit time vanishes for $\kappa < 1/2$, is nonzero for $\kappa = 1/2$, and diverges otherwise. We obtain the same surprising result as in the linear response: By a proper choice of parameters one can increase the power, achieve efficiencies arbitrary close to η_C , and, at the same time, suppress the entropy production. Finally, the ratio P/P^* reads $P/P^* \approx 2A_h^\kappa$, and thus it scales in the same way as ΔS_{tot} and σ/P [exponent θ in Eq. (3) equals 1].

The discussed setting represents another example of reaching η_C at $P > 0$ by fast driving, as suggested in Ref. [18] for a different model. In Sec. IV, we show that a HE based on an overdamped Brownian particle driven by an optimally

controlled parabolic potential is a low-dissipation HE. Interestingly enough, this is not the case for an underdamped particle.

D. Underdamped Brownian heat engine

For a stochastic HE based on an underdamped Brownian particle driven by optimally controlled parabolic potential, the bound on efficiency at fixed power is [29]

$$\eta_+(\delta P) = \eta_{CA}(1 + \delta P) - \frac{\eta_C}{2}\delta P + \frac{1}{2}\sqrt{-\delta P}\sqrt{\eta_C^2 - \eta_{CA}^4(1 + \delta P)}, \quad (20)$$

where $\eta_{CA} = 1 - \sqrt{T_c/T_h}$ stands for the famous Curzon-Ahlborn efficiency [67–70]. The model used for the derivation is based on the cycle depicted in Fig. 3.

The maximum power is given by $P^* = \gamma_h\gamma_c(\sqrt{T_h} - \sqrt{T_c})^2/(\sqrt{\gamma_h} + \sqrt{\gamma_c})^2$, and thus it diverges for diverging temperature differences or for diverging friction constants γ_h and γ_c , in complete contradiction to what is found in the overdamped limit [see Eqs. (18) and (27)]. The former regime of operation is unphysical, and the latter one breaks the basic assumption of the model. The formula (20) has been derived assuming a small friction coefficient as compared to the frequency ω of the parabolic potential ωx^2 . Under this assumption, η_C can be reached only for very strong potentials.

From Eqs. (3) and (20) it follows that $\sigma/P = \eta_{CA}^2/(T_h\eta_C)P/P^*$ and thus the exponent θ in Eq. (3) is for the present model equal to 1.

IV. PROPOSED EXPERIMENT: BROWNIAN HEAT ENGINE

Brownian HEs are frequently used to demonstrate and verify the latest results in stochastic thermodynamics [30,31,31–37,39,40,56]. In this section, we show how to tune the control parameters of a realistic overdamped HE such that it works close to η_C at large output power.

We consider the one-dimensional Brownian HE with the working cycle depicted in Fig. 3. The probability density for the particle position satisfies the Fokker-Planck equation [56,71]:

$$\frac{\partial}{\partial t} p(x,t) = -\frac{\partial}{\partial x} j(x,t), \quad (21)$$

$$j(x,t) = -\frac{1}{\gamma(t)} \left\{ k_B T(t) \frac{\partial}{\partial x} + \left[\frac{\partial U(x,t)}{\partial x} \right] \right\} p(x,t), \quad (22)$$

supplemented by the periodicity condition $p(x,t+t_p) = p(x,t)$. Above, k_B is the Boltzmann constant, $\gamma(t)$ denotes the friction constant, $T(t)$ stands for the actual temperature of the reservoir coupled to the particle, and $U(x,t)$ denotes the externally controlled potential. The friction constant depends on temperature. Thus we have $T(t) = T_h$, $\gamma(t) = \gamma_h$ along the hot isotherm and $T(t) = T_c$, $\gamma(t) = \gamma_c$ along the cold one.

The thermodynamics of the Brownian HE is described by Eqs. (12) and (13) with the parameters [56] (see also Eqs. (8)

and (10) in Ref. [72])

$$A_{h,c} = \gamma_{h,c} t_{h,c} \int_{t_i^{h,c}}^{t_f^{h,c}} dt \int_{-\infty}^{\infty} dx \frac{j^2(x,t)}{p(x,t)}. \quad (23)$$

Here t_i^h (t_f^h) denote the initial (final) time of the hot isotherm of the cycle and t_i^c (t_f^c) denote the same for the cold one. In general, the parameters $A_{h,c}$ depend on durations of the two isotherms in a nontrivial way. However, once the time dependence of the potential is optimized to yield maximum output work, the parameters $A_{h,c}$ become independent of $t_{h,c}$. Then the Brownian HE is a low-dissipation HE.

Further, we assume the specific potential

$$U(x,t) = \frac{k(t)}{2} x^2, \quad (24)$$

for which the optimization procedure can be performed analytically [56,57] and which is easily created by optical tweezers [30–32,38]. The resulting optimal driving $k(t)$ contains a discontinuity connected with the instantaneous change of temperature during the adiabatic branches [57]. Let us denote as $k_h(t)$ ($k_c(t)$) the optimal protocol during the hot (cold) isotherm. These functions read

$$k_h(t) = \frac{T_+}{2w_0} \frac{1}{(1 + b_1 t)^2} - \frac{b_1}{1 + b_1 t}, \quad (25)$$

$$k_c(t) = \frac{T_-}{2w_f} \frac{1}{[1 + b_2(t - t_h)]^2} - \frac{b_2}{1 + b_2(t - t_+)}. \quad (26)$$

Here, the parameter w_0 (w_f) stands for the variance of the particle position at the beginning (end) of the hot isotherm (see Fig. 3). The constants b_1 and b_2 are given by $b_1 = (\sqrt{w_f/w_0} - 1)/t_h$ and $b_2 = (\sqrt{w_0/w_f} - 1)/t_c$ [73].

Using this driving, the parameters A_h and A_c are given by

$$A_{h,c} = \gamma_{h,c} (\sqrt{w_f} - \sqrt{w_0})^2. \quad (27)$$

The scaling $A_c \propto A_\infty^{-\phi}$ and $A_h \propto A_\infty^{-\delta}$ proposed in Sec. III C to reach the Carnot efficiency in the limit $A_\infty \rightarrow \infty$ thus implies that η_C can be reached either for vanishing friction constants $\gamma_{h,c}$ or for the vanishing bracket $(\sqrt{w_f} - \sqrt{w_0})$. The assumption of small friction constants contradicts conditions of the overdamped limit. The ratio w_f/w_0 determines the increase in system entropy during the hot isotherm, $\Delta S = k_B \log(w_f/w_0)/2$, and thus the reversible work done by the system. In order to achieve small parameters $A_{h,c}$, we thus assume that the particle is during the whole cycle strongly localized, i.e., $w_{0,f} \rightarrow 0$, while we keep constant w_f/w_0 .

In micromanipulation experiments with Brownian particles, the Carnot efficiency can be achieved as follows. We set $\delta = \phi$ and $\gamma_h = \gamma_c$ and thus $A_h = A_c$. The assumption of equal friction coefficients is realistic for changing the temperature in accord with the study [32]. We also take equal durations of the two isothermal branches, $t_h = t_c$ (thus $\alpha = 1/2$). Finally, we assume that the largest variance during the cycle scales as $w_f = w_\infty^{-\xi}$, $\xi > 0$, and thus the coefficients $A_{h,c}$ are given by $A_h = A_c \propto w_\infty^{-\xi}$, i.e., $\delta = \phi = \xi$.

In the numerical illustration shown in Figs. 4–6, we consider a very light colloidal particle of the diameter $R = 10^{-6}$ m diffusing in water with the friction coefficient given by Stokes' law: $\gamma_h = \gamma_c = \gamma = 6\pi R\mu$. Here $\mu = 1.002 \times 10^{-3}$ Pa s is

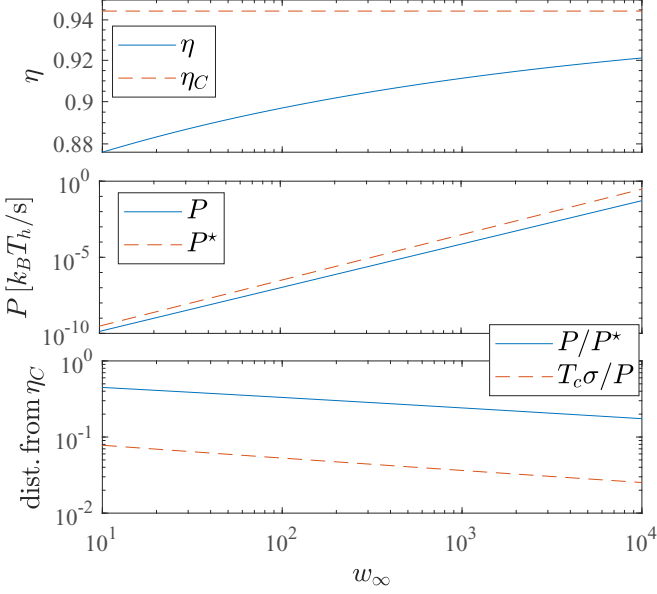


FIG. 4. Efficiency η (upper panel), power P and maximum power P^* (middle panel), and two variables (3) measuring the distance from the Carnot efficiency of the engine described in Sec. IV as functions of the scaling parameter w_∞ . The increasing efficiency is accompanied with the increasing power in direct contradiction with the quasistatic limit. The detailed description and the parameters used are given in Sec. IV.

the dynamic viscosity of water at the room temperature, 293.15 K. We assume that the real bath temperature during both branches is $T_c = 293.15$ K and that during the hot isotherm this temperature is effectively increased by an additional external noise to $T_h = 5273.15$ K, in accord with the recent experimental work [32]. The corresponding Carnot efficiency is $\eta_C \approx 0.945$. We use the exponent $\kappa = 0.05$ for the cycle time t_p and $\xi = -3$ for the maximum variance w_f , i.e., we take $t_p = t_p^* A_h^{-0.05}$ and $w_f = w_\infty^{-3}$. Finally, we fix the ratio of the maximum and minimum variance to be $w_f/w_0 = 2$. According to Sec. III C, this choice leads to the following scaling of the thermodynamic variables in question: $P \propto w_\infty^{2.85}$, $P^* \propto w_\infty^3$, $t_p \propto w_\infty^{-2.85}$, $t_p^* \propto w_\infty^{-3}$, $\eta_C - \eta \propto P/P^* \propto \sigma/P \propto \Delta S_{\text{tot}} \propto w_\infty^{-0.15}$, and $\sigma \propto w_\infty^{2.7}$.

In Fig. 4 we show the behavior of thermodynamic variables of the system with increasing parameter w_∞ . The engine efficiency η converges to $\eta_C \approx 0.945$ (upper panel). In contrast to the quasistatic limit, this increase in η is accompanied with an increase in power (middle panel). The lower panel shows the convergence of the ratio of power to maximum power, P/P^* , and of the product $T_c \sigma / P$ to zero as $\eta \rightarrow \eta_C$. The two lines are parallel, as predicted by Eq. (3) for η close to η_C .

In Fig. 5 we show experimentally controlled variables as functions of the parameter w_∞ . Both the cycle duration t_p and the optimal cycle duration t_p^* goes from experimentally inaccessible values (years) for small w_∞ to reasonable values (seconds) for large w_∞ . Similarly, the minimum and maximum particle variance (middle panel) are very large for small w_∞ and attain realistic values for large w_∞ . The corresponding maximum spring constant $k_{\text{max}} = k_h(0)$ is plotted in the lower panel. The whole range of the spring constant shown in the

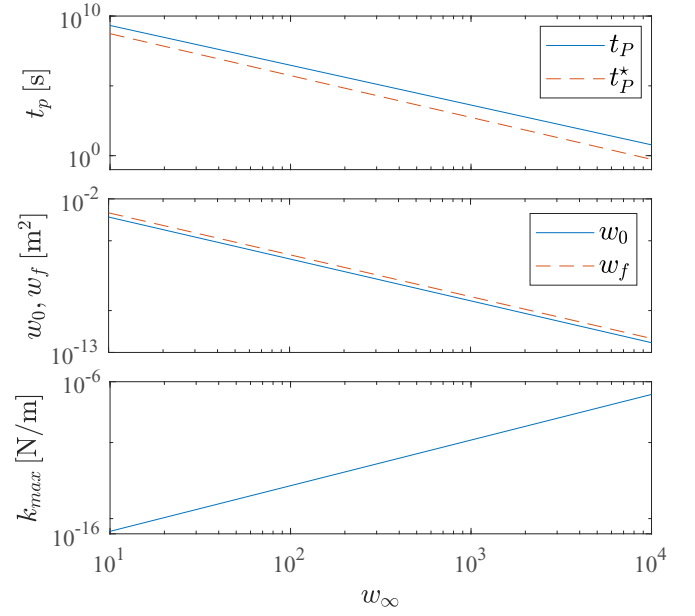


FIG. 5. Total cycle duration t_p and duration of the cycle at maximum power t_p^* (upper panel), minimum and maximum variances of the particle position during the cycle w_0 and w_f (middle panel), and the maximum value of the trap stiffness (lower panel) for the engine described in Sec. IV as functions of the scaling parameter w_∞ . The region accessible in experiments is roughly $w_\infty \in (10^3, 10^4)$, where the cycle time decreases from 1 h to 5 s. A detailed description and the parameters used are given in Sec. IV.

figure, especially the part for large w_∞ , can be readily achieved in experiments either by using optical tweezers [30–32] or a feedback or anti-Brownian electrokinetic trap [33–37].

In Fig. 6 we show the relative fluctuation of power, $\sqrt{\langle P^2 \rangle - \langle P \rangle^2} / \langle P \rangle$. The calculation has been performed numerically using the procedure described in Ref. [57], Sec. 3.2. The power fluctuation increases with increasing scaling parameter w_∞ and saturates at a relatively small value in the

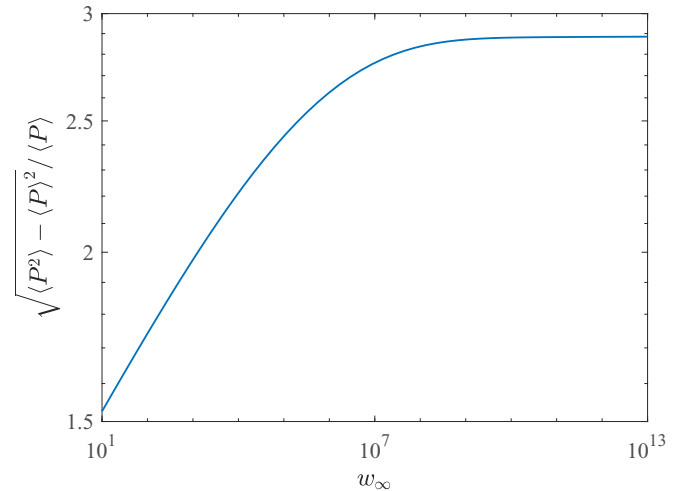


FIG. 6. The relative fluctuation of power for the Brownian HE described in Sec. IV as a function of the scaling parameter w_∞ . A detailed description and the parameters used are given in Sec. IV.

limit $w_\infty \rightarrow \infty$. In contrast to the critical HE introduced in Ref. [16] (see Ref. [20]), the proposed Brownian HE delivers a relatively stable output power.

V. CONCLUSION AND OUTLOOK

The struggle to reach the Carnot efficiency at nonzero output power is an exciting part of the current research in nonequilibrium thermodynamics. In the present work, we have added another piece into the mosaic: For various models widely used to describe the performance of actual HEs, the output power at which η_C can be possibly reached is doomed to be negligibly small as compared to the maximum power achievable in these models. This is best visible from Eq. (3), which shows that both the ratio of entropy production to the output power, σ/P , and the ratio of output power to the maximum power, P/P^* , must vanish when the Carnot limit is attained.

Besides that, we have investigated conditions for reaching divergent maximum power and thus η_C at $P > 0$ in the individual models. These settings seem to be unrealistic for thermoelectric HEs (infinite temperature gradient) and for an underdamped Brownian HE (the required conditions break assumptions of the model). More realistic conditions were found for linear response HEs, minimally nonlinear irreversible HEs, and low-dissipation HEs.

In the linear response regime, η_C may be attained for diverging Onsager coefficients, as also suggested in Ref. [18]. The open question is whether such conditions can be achieved in a real system. A suitable candidate is the critical HE proposed by Campisi and Fazio [16].

In the low-dissipation regime, η_C can be achieved for fast cycles with vanishing dissipation coefficients A_h and A_c . In practice, these conditions can be (nearly) satisfied by Brownian HEs, which might be constructed using current micromanipulation experimental techniques. Concrete parameters and the driving protocol for a Brownian HE operating near η_C at large P are discussed in Sec. IV. We believe that this detailed analysis will stimulate experimental verification of our findings. In connection with the experiment, it would be very interesting to investigate the behavior of probability densities

for work, heat, and fluctuating efficiency [5] for η close to η_C at large power.

At first glance, the condition $\sigma/P \rightarrow 0$ implies that reaching η_C at $P > 0$ should require diverging power P , because one naturally assumes that $\sigma > 0$. For linear response HEs and low-dissipation HEs, we have found certain scalings which allow reaching η_C at $P > 0$ and vanishing entropy production at the same time. The corresponding power can both diverge and attain a finite nonzero value. To the best of our knowledge, such a scenario has not been discussed in previous works. In practice, this allows constructing a HE operating close to the Carnot efficiency at a large output power and with a small entropy production.

The sensitivity of the individual models to the precise form of the scaling which must be chosen in order to achieve η_C at $P > 0$ represents the biggest qualitative difference between the present approach and the quasistatic limit. In order to realize a quasistatic cycle, it is enough to make it very slow, regardless of the details of the system. On the contrary, the scalings leading to the Carnot efficiency at nonzero power must be engineered in a model-dependent manner and, moreover, their practical usage requires precise control of the system dynamics.

Our present knowledge suggests that the limit $\eta \rightarrow \eta_C$ always incurs negative effects. In the quasistatic limit, the power at η_C vanishes. For critical HEs [16,17,20], approaching η_C at $P > 0$ is accompanied by diverging power fluctuations (even in the macroscopic limit). For HEs studied in this work we observe a large loss in power as compared to the maximum power regime. Interestingly, such negative effects are not present in machines working under isothermal conditions in a steady state driven by chemical or external forces [74]. Some of these machines can reach the second law upper bound on efficiency under maximum power conditions.

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