

## Propulsion of Bubble-Based Acoustic Microswimmers

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Acoustic microswimmers present a great potential for microfluidic applications and targeted drug delivery. Here, we introduce armored microbubbles (size range, 10–20  $\mu\text{m}$ ) made by three-dimensional microfabrication, which allows the bubbles to last for hours even under forced oscillations. The acoustic resonance of the armored microbubbles is found to be dictated by capillary forces and not by gas volume, and its measurements agree with a theoretical calculation. We further measure experimentally and predict theoretically the net propulsive flow generated by the bubble vibration. This flow, due to steady streaming in the fluid, can reach 100 mm/s, and is affected by the presence of nearby walls. Finally, microswimmers in motion are shown, either as spinning devices or free swimmers.

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### I. INTRODUCTION

Using microswimmers for transport and mixing purposes has been central to numerous research projects over the past decade [1–3]. Microswimmers act as propulsion devices that can carry a payload, and the propulsion jet can be used for mixing. Artificial microswimmers have recently been made possible because of two specific developments: on one side microfabrication and nanofabrication [4,5], and on the other contactless biocompatible devices such as magnetic fields [6–8], chemical engines [9,10], or acoustically excited bubbles [5,11–13], which are used to move objects at the microscopic scale.

One very promising actuation method exploits contactless acoustic sources. Sound waves can travel through biological tissues if their acoustic impedance is similar to the impedance of water, so reflections are minimized [14]. A bubble subjected to ultrasound will pulsate intensely [15], especially when the driving frequency is near the resonance of the bubble. The pulsating bubble generates powerful acoustic streaming [16] that can then be harnessed for propulsion. This mechanism has been used with some success at the 50–100- $\mu\text{m}$  scale, in two-dimensional (2D) microchannels, with a contact disc transducer that has sharp resonance frequencies [5,12,13,17].

In this paper, we demonstrate how to use three-dimensional (3D) microfabrication in order to build objects in the 10–20- $\mu\text{m}$  range, that contain a microbubble—a device we call *armored microbubble* (AMB). Unlike free air bubbles in water at room temperature [18], the AMBs are protected from dissolution, and have a long life even

under acoustic forcing. We then show how to create a microswimmer by exciting an AMB with a contactless focused broadband transducer. Experiments and theory are used jointly to measure and predict resonance frequencies and strong streaming flows, with excellent agreement. We finally demonstrate how actuated AMBs may be used to induce swimming, in a manner which is affected by their environment and their shape.

### II. FABRICATION, SETUP, AND BUBBLE LIFE SPAN

In order to fabricate 3D microobjects, we use a two-photon polymerization setup (TEEM Photonics). The precision is on the order of the Nd:YAG microchip laser wavelength, 532 nm. The setup is mounted on the epifluorescence port of an inverted microscope. Fabrication parameters are described in detail in Ref. [19]. The AMBs are designed on FreeCAD and fabricated on a glass coverslip (approximately 30 min per AMB) in the shape of a hollow capsule mounted on a pole [see Figs. 1(a) and 1(b)]. The solid capsule is surrounded by liquid polymer which is ultimately washed away with acetone. The process generates 0.5- $\mu\text{m}$ -thick shells, creating hollow objects that can be reinforced inside with plates or bars. The material used is OrmoComp<sup>®</sup>, which is biocompatible but not biodegradable. A crack at the top of the capsules is visible on scanning-electron-microscopy (SEM) images after fabrication [see Fig. 1(b)], but using a double-shell design prevents leaks. To simulate an acoustic wave traveling through a microchannel, the coverslip is placed in a polydimethylsiloxane (PDMS) cell filled with a 25-wt % NaCl water solution containing 2- $\mu\text{m}$  spherical particles to observe the streamlines [see Figs. 1(c)–1(e)]. The cell

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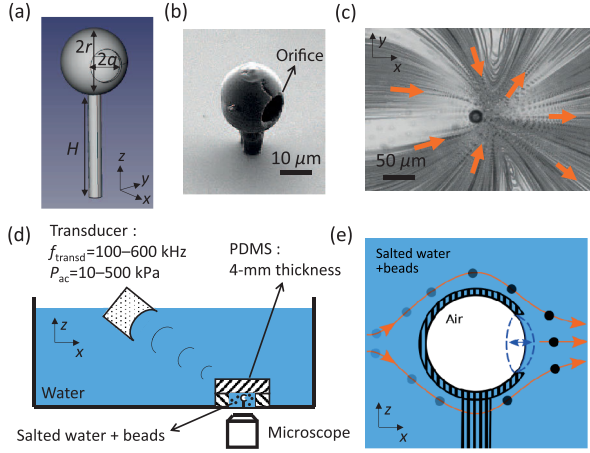


FIG. 1. (a) FreeCAD design of a capsule with  $r = 9\ \mu\text{m}$  and  $a = 5\ \mu\text{m}$  on a pole of length  $H = 30\ \mu\text{m}$ . (b) SEM image of the device illustrated in (a). (c) Streamlines generated by device (b) submerged in water (viewed from below) at transducer’s frequency  $f_{\text{transd}} = 320\ \text{kHz}$  and acoustic pressure  $P_{\text{ac}} = 9.2\ \text{kPa}$ . (d) Experimental setup. (e) Schematic of the bubble-first vibration mode.

is then placed inside a 1-L water tank [see Fig. 1(d)]. Films are recorded with a Phantom v2511 high-speed camera, with frame rates up to 10 000 frames per second, mounted on a IX73 Olympus microscope with  $\times 20$  or  $\times 40$  lenses. For sound emission, a contactless Olympus v381 transducer (500 kHz, focused, 330-kHz half-amplitude bandwidth) is used. Its broadband emission will allow us to detect the true resonance of AMBs by scanning frequencies.

A challenging task when studying the acoustic streaming of AMBs is to prolong the life of the bubbles. Air in an AMB [see Fig. 1(b)] lasts for about 15 min in deionized water, and about 1 min when exposed to ultrasound. Condensation droplets appear inside the capsule, filling it gradually. We solve the problem by using a 25-wt % NaCl water solution, leading to an AMB lifetime of hours under ultrasound (a phosphate-buffered saline containing only a few percent of salt produces the same effect).

### III. STREAMING

Having overcome the dissolution and lifetime challenge, we can now study the flow generated by AMBs for hours at a time. The vibration of the bubble interface under ultrasonic forcing results in a strong “acoustic-streaming” jet [20]. Passive particles suspended in the fluid are seen flowing from the back and sides of fixed AMBs toward their opening before being pushed forwards away from the AMB [see Figs. 1(c) and 1(e)]. The fluid jet directed away from the AMB provides the basis for a propulsion mechanism, and gives the order of magnitude of the velocity for a free swimming device.

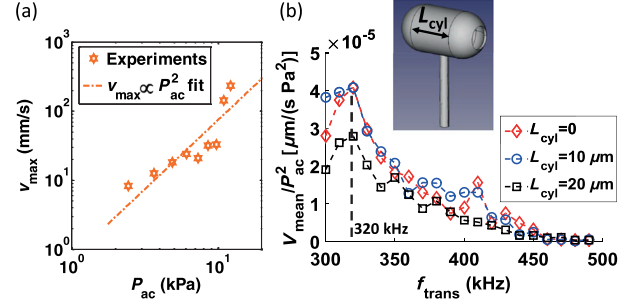


FIG. 2. (a) Maximum flow velocity  $v_{\text{max}}$  as a function of the acoustic pressure  $P_{\text{ac}}$ . (b) Normalized mean flow velocity  $v_{\text{mean}}/P_{\text{ac}}^2$  as a function of the transducer frequency  $f_{\text{transd}}$  for a voltage  $U_{\text{transd}} = 7.6 V_{\text{pp}}$  and three different values of  $L_{\text{cyl}}$ . The resonance frequency is measured to be 320 kHz.

We measure the maximum fluid velocity  $v_{\text{max}}$  of all the particles outside on the front side of the AMB. The particle trajectories are analyzed with a predictive tracking algorithm [21]. A strong streaming flow is achieved with velocities up to 100 mm/s near the aperture [see Fig. 2(a)]. For a 20- $\mu\text{m}$  AMB, this represents 5000 body lengths per second, despite the PDMS cell which diminishes the acoustic pressure by 30% according to our measurements. This loss is higher than, for example, losses due to ultrasound crossing organs (1% to 10% [14]). Moreover, the experimental points are in good agreement with a scaling  $v_{\text{max}} \propto P_{\text{ac}}^2$ , reflecting the quadratic nonlinearity of acoustic streaming.

### IV. RESONANCE

In order to optimally control and actuate AMBs, we have to quantify and predict their first important characteristics, namely, their resonance frequency. Free bubble have well predicted resonances [15]. Recently, Wang *et al.* [22] studied streaming from a 50- $\mu\text{m}$ -diameter bubble and noted that it generated a streaming flow of comparable velocities when excited in the range of 1–100 kHz. However, since they used a disc transducer with sharp resonances, they could not characterize the dependence of the streaming velocity on the pressure amplitude.

In our case, the broadband transducer allows us to measure the resonance of the AMB. The mean velocity of the particles,  $v_{\text{mean}}$ , is measured in a  $10 \times 100\text{-}\mu\text{m}$  rectangle outside the opening, for three different AMBs with identical apertures but different volumes. Changing the volume is achieved by adding a cylindrical part of length  $L_{\text{cyl}}$  to the AMB [Fig. 2(b)]. The case  $L_{\text{cyl}} = 0$  is the regular spherical AMB. A clear resonance is noted around 320 kHz for all shapes, with a 3-dB bandwidth of approximately 30 kHz [see Fig. 2(b)]. The resonance bandwidth means that a significant streaming ( $v_{\text{mean}} \geq 100\ \mu\text{m}/\text{s}$ ) is generated for a large range of frequencies, which may be used for activating AMBs of different shapes with the same sound wave.

Can this measured resonance frequency of 320 kHz be rationalized? The origin of the resonance is related to that of a Helmholtz resonator, with the additional ingredient of the capillary restoring force. For a Helmholtz resonator the resonance is illustrated by a mass and spring system, with the mass that of the gas oscillating in the entrance and the elasticity due to the gas compressibility in the cavity. Here, the mass is mainly given by the liquid oscillating near the entrance, scaling like  $m \sim \rho_l \pi a^2 l$ , where  $\rho_l$  is the liquid density and  $l$  an effective length on the order of the entrance length. The elasticity is due to gas compressibility and additionally here to surface tension  $\gamma$ . Therefore, two restoring forces act on the liquid mass,  $F_{\text{gas}} = -k_{\text{gas}}(x - x_0)$  and  $F_{\text{cap}} = -k_{\text{cap}}(x - x_0)$ , with  $x$  the typical position of the apex of a surface deformation shown on Fig. 1(e). The gas and capillary stiffness are, respectively,  $k_{\text{gas}} \sim \kappa P(\pi a^2)^2/2V$  and  $k_{\text{cap}} \sim 2\pi\gamma$ , where  $V$  is the volume of gas in the capsule,  $P_0$  is the atmospheric pressure, and  $\kappa = 1.4$  is the adiabatic index. On the small scales relevant to our experiments, we obtain that the capillary stiffness is dominant and thus changing the AMB air volume  $V$  has less of an effect on the resonance frequency,  $\omega_0 = [(k_{\text{gas}} + k_{\text{cap}})/m]^{1/2}$ , than changing the aperture  $a$ . This is consistent with our findings in Fig. 2(b).

In order to quantitatively predict the resonance frequency, we develop a fluid mechanics model incorporating the dominant capillary effects but neglecting the physics of the gas inside the bubble. Similarly to classical work on free bubbles, we describe the flow as irrotational (potential), but here we have mixed boundary conditions on the AMB surface assumed to be perfectly spherical: no normal velocity on the solid portion of the capsule and normal stress balanced by capillary stresses on the aperture gap. The model reduces to a generalized eigenvalue problem which can be solved numerically. In that case, the model predicts a resonance frequency  $f_{\text{th}} = 333$  kHz for the first mode  $n = 1$  of our AMB (using the measured value  $\gamma = 69.7$  mN/m for a 25-wt % salted-water solution and particles). This is in excellent agreement with the experimental value of 320 kHz. The model can also be used to predict how the aperture size  $a$  affects the mode shapes in the limit  $a \ll r$  and their resonance frequency. Specifically, we predict the scaling

$$f_{\text{th}} \sim \left( \frac{n^3 \gamma}{\rho_l r^3 \theta_0^3} \right)^{1/2}, \quad (1)$$

where  $\theta_0 = \sin^{-1}(a/r)$  is the azimuthal angle where the aperture gap ends and  $n$  is the shape mode number.

The net flow created in the fluid as a result of the periodic oscillations of the free surface could be due to the combination of two phenomena, namely Eckart and steady streaming. Eckart streaming occurs when a sound wave dissipates part of its momentum in a viscous liquid, allowing that liquid to move [23]. Here, the attenuation of waves occurs at much longer distances than the AMB

size, so that we can discard the acoustic streaming that would follow the direction of propagation of the wave.

The second type of streaming, termed steady (or boundary acoustic) streaming, is then dominant [20]. It occurs in viscous boundary layers near walls (see Ref. [24]), for example, at an air-water interface such as the surface of a bubble [25,26]. There, momentum is imparted to the fluid, resulting in a steady rectified fluid motion out of an oscillating forcing.

## V. MODEL

In order to predict the streaming flow, we develop a theoretical model. We assume the body is spherical and undergoes small-amplitude, axisymmetric surface-shape oscillations [27]. Previous streaming work focused on simple-shape oscillations of single material boundaries [26,28–30] and our model is able to generalize these results to a rigid capsule with an oscillating cap. We assume oscillations of a small-amplitude  $\epsilon r$  and a small viscous boundary layer compared to the particle size (relative size  $\delta$ ), with  $\epsilon \ll \delta \ll 1$ . This is the relevant limit for micron-sized capsules in water being forced by ultrasound with kHz frequencies, and thus the relevant limit for our experiments.

In order to characterize the net streaming flow, we solve the problem as a power-series expansion in the relative oscillation amplitude  $\epsilon$ . At  $O(\epsilon)$ , the equations have an analytical solution (this is an unsteady Stokes problem). At  $O(\epsilon^2)$  we obtain the net streaming, which requires asymptotic matching to solve. The mixed boundary conditions of no motion on the rigid portion of the capsule and appropriate free-stress conditions on the aperture can then be applied to determine the surface-shape oscillation, and hence the streaming this oscillation generates. The Lagrangian stream function in spherical coordinates  $(\rho, \theta)$  around the AMB center is given by  $\bar{\Psi}_L(\rho, \theta) = \sum_{n,m} A_{n,m} \rho^{-n} \int_{\cos\theta}^1 P_m(x) dx$ , where  $P_m(x)$  is the  $m$ th Legendre polynomial [27]. The coefficients  $A_{n,m}$  are a sum of quadratic combinations of the amplitude of the vibration modes, described by a sum of spherical harmonics. As a result, we obtain a streaming field that behaves at long distances from the AMB as a point-force flow (Stokeslet). The intensity of this Stokeslet,  $F_{\text{thr}}$ , provides the thrusting force on the microswimmer of order

$$F_{\text{thr}} \sim \epsilon^2 \rho_l r^4 f^2, \quad (2)$$

with a prefactor which depends on the shape of the surface oscillation. For our AMB this prefactor is approximately 4, leading to the prediction  $F_{\text{thr}} = 0.3$  nN.

For an AMB away from the wall, the model accurately predicts the general shape of the streamlines using experimental parameters [see Fig. 3(a)]. Beyond the current application, our model will also be relevant to any spherical oscillating body in this regime (e.g.,

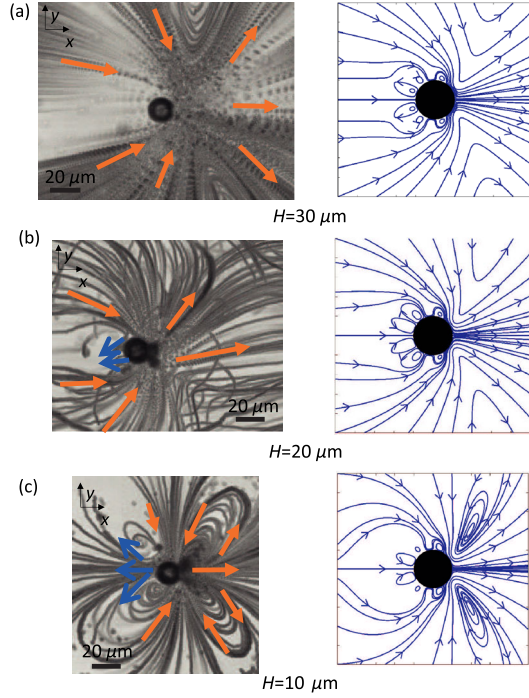


FIG. 3. Streaming flows induced by AMBs (left) and theoretical streamline predictions (right) with different poles lengths  $H$  [(a)  $H = 30 \mu\text{m}$ ; (b)  $H = 20 \mu\text{m}$ ; (c)  $H = 10 \mu\text{m}$ ] for  $f_{\text{transd}} = 320 \text{ kHz}$  and  $P_{\text{ac}} = 9.2 \text{ kPa}$ .

spherical microorganisms) or other more complex spherical mechanical swimmers [5,13,27].

The 3D streaming flow generated by the devices may be affected by the proximity of solid boundaries. A transport AMB would travel in tight channels (e.g., biological) and it is important to understand how the propulsion may be affected by the walls. To that purpose, we fabricate AMBs with different pole lengths  $H$  and compare the shapes of the streamlines. The effect of the value of  $H$  is noticeable [see Figs. 3(a)–3(c)]: at large distances from the wall the fluid is pushed forward (+ $x$  direction) everywhere, while close to the wall, a (small) backward flow appears behind the capsule (– $x$  direction). Closed circulation regions in the flow also appear near surfaces. We thus conjecture that propulsion may decrease in efficiency due to the backward flow occurring when AMBs approach boundaries.

The presence of a boundary is incorporated in the analytic streaming model by adding the image system for the leading-order Stokeslet flow decaying as  $1/r$  [31] and next-order Stresslet decaying as  $1/r^2$  [32]. Our model predicts the increasing efficiency of the propulsion away from the wall. Indeed, recirculation patterns in front of the AMB are closed for  $H = 10 \mu\text{m}$  and open up for  $H = 20$  and  $30 \mu\text{m}$ . The general flow pattern is also well predicted by the model. The model is not able to predict the small backward flow seen experimentally, and a full numerical approach would be required to capture all near-field features.

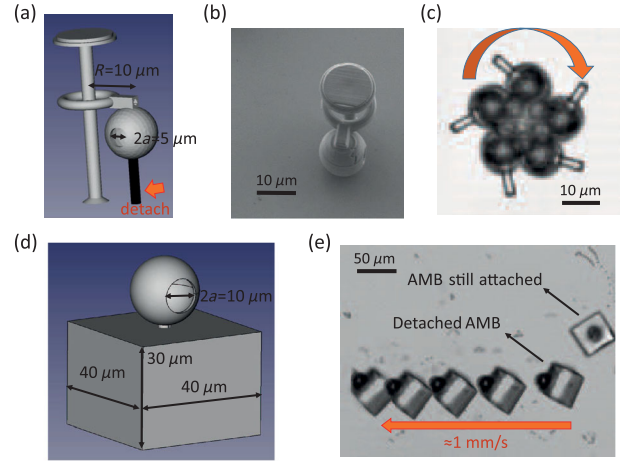


FIG. 4. (a) Spinner design; (b) fabricated spinner (SEM image); (c) AMB spinning at 175 Hz under actuation with  $P_{\text{ac}} = 96.2 \text{ kPa}$  and  $f_{\text{transd}} = 500 \text{ kHz}$ ; (d) AMB and pedestal; (e) moving AMB with  $P_{\text{ac}} = 5.6 \text{ kPa}$ .

## VI. SWIMMERS

With our understanding of the physics of force generation, we can now use these forces to generate propulsion. As AMBs tend to stick to the substrate after fabrication, we use a tungsten probe mounted on a microstage to detach them. To quantify the propulsion the AMBs generate, we design a two-part “spinner”: a pole and a legged AMB connected to a ring [see Fig. 4(a)]. The probe frees the latter part: it is buoyant but blocked by the top hat. The streaming propels the AMB which rotates around the central pole [Fig. 4(c)]. We achieve a rotation frequency of  $f_{\text{rot}} = 175 \text{ Hz}$ . If we assume the thrusting force  $F_{\text{thr}}$  is equal to the Stokes’ drag  $6\pi\eta Rv$ , with  $v = 2\pi Rf_{\text{rot}}$  and  $R$  gyration radius [see Fig. 4(a)], then we measure a force of  $F_{\text{thr}} = 0.75 \text{ nN}$ . This thrust is in good agreement with the analytic thrust prediction of Eq. (2). Note that this is a powerful thrust, 2 orders of magnitude larger than the buoyancy force acting on a 12- $\mu\text{m}$  bubble.

## VII. CONCLUSION

To conclude, in this paper we have fabricated and tested devices in the 10–20  $\mu\text{m}$  range able to generate contactless acoustic propulsion from stabilized microbubbles lasting hours. Our results are promising for future studies on other self-traveling microobjects, as basic elements for controlled active assemblies, and ultimately for applications in mixing and transport. For that purpose, we have also fabricated an AMB mounted on a solid pedestal countering the bubble buoyancy once freed. In Fig. 4(e) we show a free AMB next to one which is still attached. The free AMB is propelled at speeds of order 1 mm/s solely by the bubble-based steady streaming ( $F_{\text{thr}} \approx 0.6 \text{ nN}$ ). This velocity is on the order of magnitude required to travel in blood capillaries or to be efficiently used for mixing [33]. Future improvements will

include designing multiple bubble swimmers and fixed-legged AMBs acting as mixing agents in microchannels.

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